

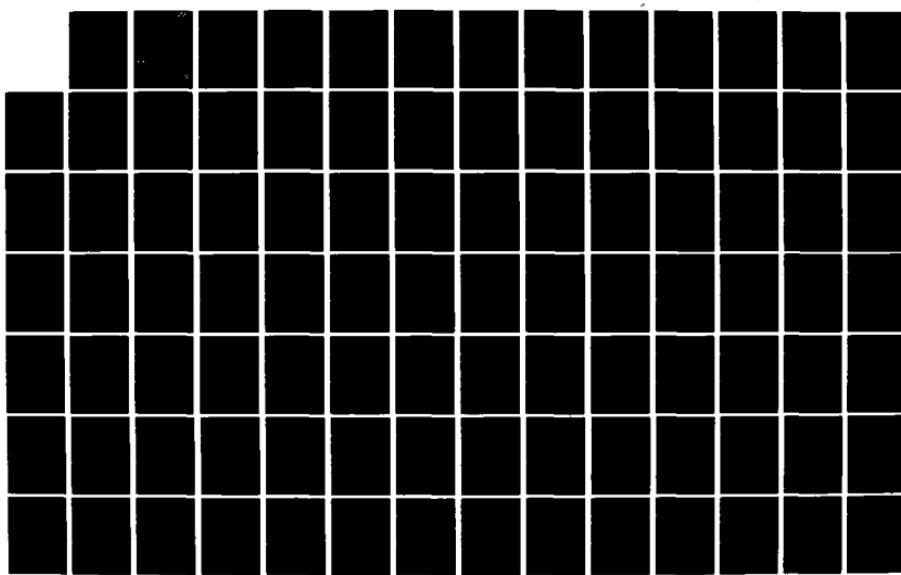
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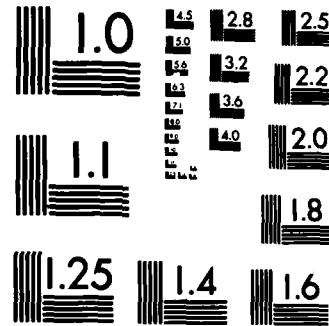
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RPSL1D

(A ONE-DIMENSIONAL VERSION OF REPSIL)

John D. Wortman

November 1982



US ARMY ARMAMENT RESEARCH AND DEVELOPMENT COMMAND  
BALLISTIC RESEARCH LABORATORY  
ABERDEEN PROVING GROUND, MARYLAND

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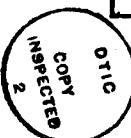
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20. ABSTRACT ( <i>Continue on reverse side if necessary and identify by block number</i> ) RPSL1D is a one-dimensional version of REPSIL, a computer program which calculates the large plastic deformation of thin Kirchhoff shells. The REPSL1D program is more efficient than REPSIL for axially symmetric shells (shells of revolution) and slab symmetric shells. It has been used most frequently for deformation calculations in laterally symmetric beams. The numerical algorithm differs from that of REPSIL as recorded in Reference 2 in that external, artificial, points are used to define boundary conditions,		

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and that forces are computed between mesh points as well as at mesh points. This gives a better simulation of clamped ends and a more compact, more stable system of difference equations. The report gives instructions for using RPSL1D. Listings of the program and the companion plotting program are given. Two example problems are discussed with tabulated input and output. A number of options, primarily for beams, are tabulated and discussed.

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## 1. INTRODUCTION

RPSL1D is a spin off of the FORTRAN IV computer program REPSIL<sup>1,2</sup> developed at the Ballistic Research Laboratory to treat large, transient, elastoplastic deformations in thin shells. Thin shell programs, such as REPSIL, replace the essentially three-dimensional geometry of the deforming shell with a two-dimensional reference surface and some assumptions about deformation through the thickness. The REPSIL family of programs is based on theory that restricts them to thin Kirchoff shells with negligible rotary inertia. The coding restricts the programs to shells of uniform thickness with clamped or hinged edges and a plane of symmetry. The RPSL1D version further restricts the program to geometries where the reference surface of the shell is a function of one variable and a symmetry assumption.

Three geometries have been incorporated: axially symmetric shells, slab symmetric shells, and laterally symmetric beams. The standard REPSIL (with some modifications) could be used for these problems, but RPSL1D is much faster and uses about one-fifth as much memory. Further, RPSL1D results should be more accurate; the numerical derivatives with respect to the symmetric variable are replaced by exact derivatives, and spacing for the other variable can be made smaller without unreasonable demands on machine storage or time.

The simpler one-dimensional model makes it easier to experiment with alternate methods of computing. Many changes and additions have been tried. Some of these have been incorporated into RPSL1D, as tabulated in Appendix C, and others discussed in Section 5 and Appendices E and F may be useful in the future.

The purpose of this report is to record RPSL1D, as cataloged in the CDC CYBER system at BRL, to record some user subroutines and useful options, to serve as a user's guide, and to give the basic information for further modifications. This is not a complete guide; the user should also obtain a copy of Reference 2, the user's manual for REPSIL. An understanding of the theory given in Reference 1 is also very helpful. The symbols in the present report, which are sometimes used without sufficient explanation, are those of Reference 1.

Sections 2 and 3 and the list of equations in Appendix A should bridge the gap between REPSIL and RPSL1D. In an effort to keep this report within reasonable bounds, we will not repeat theory given in Reference 1 nor will we repeat much of the information given in the REPSIL user's manual, Reference 2.

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<sup>1</sup>J. M. Santiago, "Formulation of the Large Deflection Shell Equations for Use in Finite Difference Structural Response Computer Codes," U.S. Army Ballistic Research Laboratories, Report No. 1571, Feb. 72.

<sup>2</sup>J. M. Santiago, H. L. Wisniewski, N. J. Huffington, Jr., "A User's Manual for the REPSIL Code," U.S. Army Ballistic Research Laboratories, Report No. 1744, Oct. 74.

The RPSL1D user must prepare input as described in Section 6 and must supply proper "user" subroutines as described in Section 4. He may retain the user subroutines with the cataloged program listed in Appendix C, replace them with copies of user subroutines listed in Appendices E and F, or replace them with his own subroutines.

For some particular applications it seems inevitable that some modifications are needed or desirable. A number of possibly useful options are described in Section 5 and listed in Appendices E and F.

The program, as stored in UPDATE form in the CDC 7600, is listed in Appendix C. A complete list of FORTRAN names used in RPSL1D is given in Appendix B along with their definition and some explanation. Two example problems are discussed in Appendix E.

The accessory program for plotting output from RPSL1D is discussed and listed in Appendix D.

The main purpose of this report is to serve as a guide to the use and possible modification of the RPSL1D program. The author's interest in this program is as a programmer. Hence, there is a bias in the presentation toward the frequent use of FORTRAN names and formulation, and the trivial details of programming, at the expense of physical meaning.

## 2. MODELS

The first one-dimensional version of REPSIL treated axially symmetric shells. This was modified to handle slab symmetric shells, and this in turn changed to treat symmetric beams.

The position,  $\tilde{x}^*$ , of any point in the shell may be expressed in terms of a point,  $\tilde{x}$ , on the reference surface, the normal,  $\tilde{n}$ , to the reference surface at  $\tilde{x}$ , and the distance from the reference surface,  $\xi$ :

$$\tilde{x} = \tilde{x} + \xi \tilde{n} . \quad (1)$$

Points on the reference surface are expressed in Cartesian coordinates as

$$\tilde{x} = Y^j \tilde{i}_j \equiv Y^1 \tilde{i}_1 + Y^2 \tilde{i}_2 + Y^3 \tilde{i}_3 , \quad (2)$$

where the  $\tilde{i}_j$  are the orthonormal basis for vectors in 3-space. The  $Y^j$ , and hence other dependent parameters, are functions of the material (Lagrangian) parameters,  $\xi^1$  and  $\xi^2$ , and of time,  $t$ .

$$\tilde{x} = \tilde{x}(\xi^1, \xi^2, t) = Y^j(\xi^1, \xi^2, t) \tilde{i}_j . \quad (3)$$

---

\*The symbols of Reference 1 will usually be followed in this report.  
The tilde below a letter denotes a vector (e.g.  $\tilde{x}$ ).

Our characterization of axial and slab symmetry is through specialization of the vector  $\underline{x}$  and the material coordinates  $\xi^1$  and  $\xi^2$ . The basic specializations are given in this section. A summary of equations is given in Appendix A.

The type of model is prescribed for RPSL1D by control parameters assigned in the user's subroutine INGEOM.

## 2.1 Axially Symmetric Shells

An axially symmetric shell is a shell of revolution (see Figure 2.1) such as a circular cylinder or the surface of a truncated right circular cone. A shell of revolution is the surface generated by revolving a plane curve (actually two parallel curves so the shell has thickness) about a line, called the axis of revolution, in its plane. Every point of the revolving curve describes a circle whose center is on the axis. For our shell, the curve must be smooth and must not intersect the axis in the region of interest.

We study the motion of the shell by following the motion of the curve formed by the intersection of the shell's reference surface and a plane through the axis. We do not allow twisting in our model, so the initial motion, and any loading, must be in this plane. Because of the axial symmetry, the internal forces are all in this plane.

Let  $\xi^1 = \theta$ ,  $\xi^2 = \xi$ , and introduce the orthonormal base vectors  $(\hat{\underline{x}}, \hat{\underline{y}}, \hat{\underline{k}})$  which vary with angle  $\theta$ . These are related to  $(\underline{i}_1, \underline{i}_2, \underline{i}_3)$  by the relations

$$\hat{\underline{x}} \equiv \hat{\underline{x}}(\theta) = \cos\theta \underline{i}_3 + \sin\theta \underline{i}_1 , \quad (4)$$

$$\hat{\underline{y}} \equiv \hat{\underline{y}}(\theta) = -\sin\theta \underline{i}_3 + \cos\theta \underline{i}_1 , \quad (5)$$

$$\hat{\underline{k}} \equiv \underline{i}_2 . \quad (6)$$

Then,

$$\underline{x} \equiv \underline{x}(\theta, \xi, t) = R \hat{\underline{x}} + Z \hat{\underline{k}} \quad (7)$$

where

$$R \equiv R(\xi, t) = \{(Y^3)^2 + (Y^1)^2\}^{1/2} , \quad (8)$$

$$Z \equiv Z(\xi, t) = Y^2 . \quad (9)$$

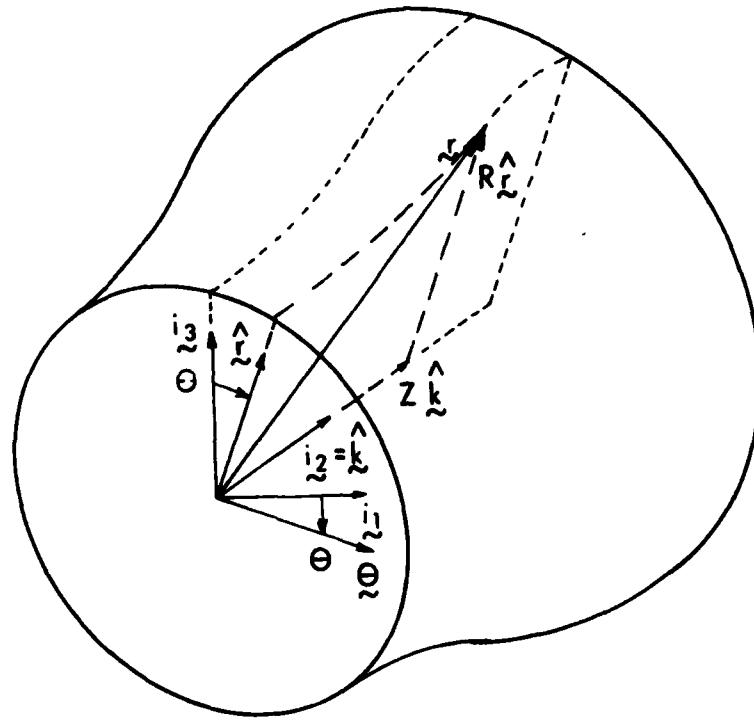


Figure 2.1 - Model for axial symmetry showing the two orthonormal bases  $(\hat{i}_1, \hat{i}_2, \hat{i}_3)$  and  $(\hat{\xi}, \hat{\theta}, \hat{k})$  with origin at the center of end 1 of the shell, and a vector  $\underline{x}$  to the surface of the axisymmetric shell.

The same formulation holds for every angle  $\theta$ . Since evaluation is needed at only one angle, we naturally choose  $\theta = 0$ . Then,

$$\begin{aligned} \underline{x}(0, \xi, t) &= R(\xi, t) \hat{\xi}(0) + Z(\xi, t) \hat{k} \\ &\equiv Y^3(0, \xi, t) \hat{i}_3 + Y^2(0, \xi, t) \hat{i}_2 . \end{aligned} \quad (10)$$

To test this new program for axial symmetry, the same problem (an initially cylindrical shell subjected to a uniform impulsive load) was simulated with both programs. Both programs were run undamped for 200 time cycles and then damped. Both self terminated on time cycle 262. The results were very close. The maximum difference noted in displacement was less than  $2 \times 10^{-8}$ . Since the displacement was about unity,  $2 \times 10^{-8}$  was also the relative difference in displacement. Strains and displacement increments are very sensitive to program differences. The largest relative difference noticed in either was  $10^{-5}$ . (There were relative differences of about 0.003 in all the printed energy values. This was because the circumference of the right circular cylinder was a circle in the axial symmetric program but was a 48-sided regular polygon in the standard REPSIL.)

## 2.2 Slab Symmetric Shells

Slab symmetric shells may be characterized by the displacement of any point being a function of its initial distance,  $\xi^2$ , from end 1 (see Figure 2.2).

Let  $\gamma^1 = \xi^1$ ,  $\gamma^2 = Y^2(\xi^2, t)$ , and  $\gamma^3 = Y^3(\xi^2, t)$ . Then,

$$r(\xi^1, \xi^2, t) = \xi^1 \hat{i}_1 + Y^2(\xi^2, t) \hat{i}_2 + Y^3(\xi^2, t) \hat{i}_3. \quad (11)$$

To utilize the formulation and programming for axial symmetry, we renamed variables

$$\xi^2 = \xi, \quad \gamma^3 = R, \quad \text{and } \gamma^2 = Z, \quad (12)$$

and confined evaluation to the plane  $\gamma^1 = 0$ . Then,

$$\begin{aligned} \mathbf{r} &\equiv \mathbf{r}(0, \xi, t) = R(\xi, t) \hat{i}_3 + Z(\xi, t) \hat{i}_2 \\ &= Y^3(\xi, t) \hat{i}_3 + Y^2(\xi, t) \hat{i}_2. \end{aligned} \quad (13)$$

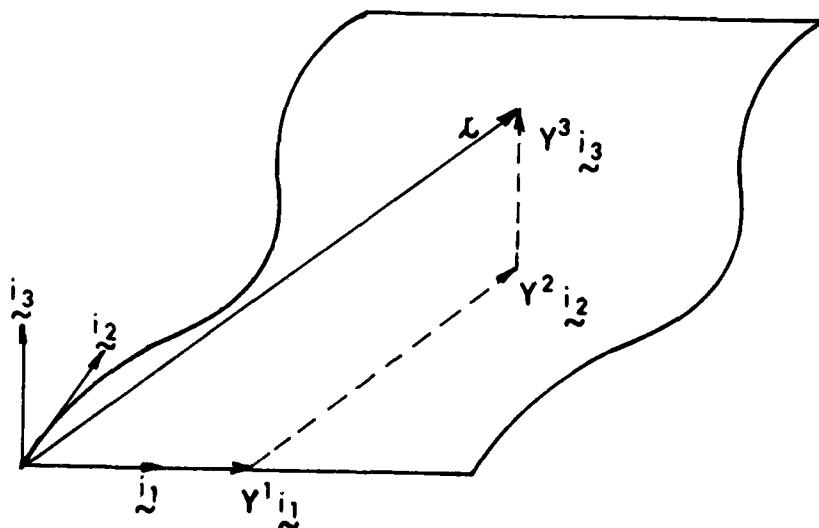


Figure 2.2 - Model for Slab Symmetry.

To maintain slab symmetry, the loading must not contain any component in the  $i_1$  direction.

For both axial and slab symmetry,  $Z$  is a distance along the  $i_2$  axis. For axial symmetry,  $R$  is the radial distance from the axis of rotation. For slab symmetry,  $R$  is the perpendicular distance from the  $Y^3 = 0$  plane.

The initial slab symmetric coding was also tested by comparing with results from a standard REPSIL run. (A minor coding change was needed to insert cyclic boundary conditions in REPSIL.) The results were nearly identical. The maximum relative errors in any variables were of the order of  $10^{-13}$ .

### 2.3 Laterally Symmetric Beams

One principal difference between slab symmetric shells and beams is in the change of concept from area to unit length (e.g. pressure, i.e. force/unit area, for shells and force/unit length for beams.) The original beam considered was a narrow rectangular beam. It was treated as a special case of slab symmetry with a uniaxial stress-strain relation replacing the biaxial one. The uniaxial stress-strain was imposed by using a new subroutine BMSTRS, instead of subroutine STRESS, for beams. By setting the width, DETA1 ( $\Delta\xi^1$ ), to unity, the concept of surface area became length without any further changes.

Treating beams that were not rectangular was more involved. It required changes in fact as well as in concept. The main change was for integrals through the thickness. The simplest form of numerical integration through the thickness is used in the standard REPSIL:

$$\int_{-h/2}^{+h/2} f(\xi) d\xi \sim \Delta\xi \sum_{k=1}^L f(\xi_k) . \quad (14)$$

The thickness,  $h$ , is divided into  $L$  layers of equal thickness,  $\Delta\xi = h/L$ , and the function to be integrated is evaluated at each  $\xi_k$ , where  $\xi_k$  is in the center of the  $k$ 'th layer. The parameter  $\xi$  is measured from the reference surface in the center of the shell.

For a beam, an integral through the thickness is replaced by an integral over the cross-sectional area. When multiplied by the proper metric, this becomes the quantity per unit surface area. For a beam, the desired result is the quantity per unit length. For a rectangular beam we could simply multiply the previous result by the width of the beam,  $\Delta\xi^1$ :

$$\int_A f(\xi) dA = \Delta\xi^1 \int_{-h/2}^{+h/2} f(\xi) d\xi \sim \Delta\xi^1 \Delta\xi \sum_{k=1}^L f(\xi_k) . \quad (15)$$

This has been replaced by a more general form for laterally symmetric but not necessarily rectangular beams:

$$\int_A f(\zeta) dA \sim \sum_{k=1}^L w_k f(\zeta_k) \quad (16)$$

where the  $\zeta_k$  are not necessarily evenly spaced and  $w_k$  is the area associated with  $\zeta_k$ . The  $\zeta_k$  and  $w_k$  should be assigned so that  $\sum w_k$  is the cross-sectional area of the beam,  $\sum \zeta_k w_k = 0$  (so the reference surface is the neutral surface for bending), and  $\sum (\zeta_k)^2 w_k$  is the area moment of inertia of the cross section. (This form of integral approximation is also suitable for Gaussian integration, which is preferred for shells; see Section 3.4.)

See the discussion of INGEOM for a beam (Section 4.1.1) for a particular example. The user subroutine INGEOM for beams must also supply the upper limit of the beam,  $ZU \equiv \zeta_u$ . Subroutine START assigns  $\zeta_l \equiv ZL = ZU - h$ . For beams,  $ZU$  and  $ZL$  merely locate positions, upper and lower respectively, at which surface strains are computed for output. The shell versions set  $ZU = h/2$ ,  $ZL = -h/2$ , and  $ZU$  is used in START to compute a number of program constants.

Results from RPSL1D for small elastic displacements for vibrations initiated in the fundamental transverse modes for clamped, hinged, and cantilever beams are in good agreement with simple beam theory. An example of buckling from lateral forces on the ends also agreed with theory.

### 3. CHANGES FROM REPSIL

The principal reason for programming a one-dimensional version of REPSIL was to have a simple vehicle for testing possible changes before implementing them in REPSIL. However, the program proved very useful for a variety of problems. Many changes have been coded. Some of these were purely experimental, but most of them were for particular problems. Most of the changes have been superseded or were not considered general enough to transfer to the CDC computer. Some of the changes have been incorporated as part of RPSL1D, as listed in Appendix C, and some are retained as options.

In this section we will outline the changes made to REPSIL to create the RPSL1D program stored in the CDC 7600 as of September 1978, and describe the new boundary condition implementation, the new differencing scheme, Gaussian integration, and the new stability criteria.

### 3.1 Outline of the Development of RPSL1D

Initially, the REPSIL program, listed in Reference 2, was changed to a one-dimensional code for axially symmetric shells (see Section 2.1). A change of notation was introduced to eliminate errors.

This program was modified to include slab symmetry (Section 2.2).

This was then modified to include beams with a vertical axis of symmetry (Section 2.3).

The method of imposing boundary conditions was changed by inserting artificial, external points as described in Section 3.3, and the divided difference approximations for partial derivatives were changed (see Section 3.2) to a form using only central differences. The system was made more compact by evaluating at mid-mesh points.

The code was changed to permit symmetry at either end and permit a free end at end 2. This change was intended for beams only. (An option called APLFRC, see Section 5.4, which permits forces to be applied at either or both ends of beams is available.)

Coding was added to monitor maximum deflection and extreme surface strains, and to print these extremes along with the surface strain prints.

The computation of surface strains was revised. The original surface strain computation was a copy of that described on pages 50 through 53 of Reference 2 except that  $a_{12}$  and  $b_{12}$  were zero and  $\zeta_u$  and  $\zeta_\ell$  replace  $h/2$  and  $-h/2$ . This was changed so that the interpolation in the non-zero covariant components of strain,  $\epsilon_{11}$  and  $\epsilon_{22}$  ( $\epsilon_{\alpha\beta} = \frac{1}{2}(\Sigma \Delta a_{\alpha\beta} \pm h \Sigma \Delta b_{\alpha\beta})$ ) was replaced by interpolation in the changes in the covariant components of the middle surface metric,  $\Sigma \Delta a_{11}$  and  $\Sigma \Delta a_{22}$ , and of the second fundamental tensor,  $\Sigma \Delta b_{11}$  and  $\Sigma \Delta b_{22}$ . Component  $\Sigma \Delta a_{22}$  is computed between mesh points.

Optional Gaussian integration through the thickness of shells was introduced.

Several minor, mostly cosmetic, changes were made.

The coding was transferred from the BRLESC computer to the CDC 7600 (with more minor changes) and stored in the UPDATE form.

### 3.2 New Difference Operators

The primary purpose of the REPSIL family of programs is to solve the equations of motion. This is done by a sequence of equations (see Reference 1, pp. 99-106) that carry the solution forward one time step.

Difference approximations of derivatives with respect to the material coordinates,  $\xi^1$  and  $\xi^2$ , are required at two points in this sequence of equations. First, we need approximations to the first and second derivatives of position,  $\underline{x}$ , and the incremental displacement,  $\Delta\underline{x}$ , to define the local geometry. Finally, we need approximations for first partial derivatives of the components of the stress resultant tensor,  $\hat{N}^{*\alpha}$ , and second partial derivatives of the normal vector components of the bending resultant tensor,  $\hat{M}^{*\alpha\beta}$ , for the equations of motion.

The REPSIL equations of motion for a mesh point, as given in Reference 1, are

$$\Delta\underline{x}^+ = \Delta\underline{x} + \frac{(\Delta t)^2}{\gamma*_0} \left[ \frac{\partial^2 \hat{M}^{*\alpha\beta}}{\partial \xi^\alpha \partial \xi^\beta} + \frac{\partial \hat{N}^{*\alpha}}{\partial \xi^\alpha} + \underline{R}^* \right]. \quad (17)$$

The change in the displacement of the point from time  $t$  to  $t + \Delta t$  is  $\Delta\underline{x}^+$ . The position at time  $t + \Delta t$  is  $\underline{x}^+ = \underline{x} + \Delta\underline{x}^+$ , where  $\underline{x}$  is the position at time  $t$ . The mass per unit initial middle surface area,  $\gamma*_0$ , is constant for a given point. (The term in brackets has units of force per initial unit area.)  $P^* = -P a^{\frac{1}{2}} n$  is the force per initial unit area along the normal,  $n$ , due to a pressure  $P$ . ( $a$  is the determinant of the covariant metric tensor.) The one-dimensional assumptions remove a number of the terms in the equations of motion. There are no terms in the  $i_1$  direction,  $\hat{M}^{*12} = \hat{M}^{*21} = 0$ , and the derivatives with respect to  $\xi^1$  are simple functions which are computed explicitly. The equations of motion become

$$\Delta\underline{x}^+ = \Delta\underline{x} + \frac{(\Delta t)^2}{\gamma*_0} \left[ \frac{\partial^2 \hat{M}^{*22}}{(\partial \xi^2)^2} + \frac{\partial \hat{N}^{*2}}{\partial \xi^2} + \underline{R}^* - (\hat{M}^{*11} \cdot \underline{i}_2 + \hat{N}^{*1} \cdot \underline{i}_1) \underline{i}_2 \right], \quad (18)$$

$$\underline{x}^+ = \underline{x} + \Delta\underline{x}^+. \quad (19)$$

(The transformation to this equation is outlined in Appendix A.) The vectors  $\hat{M}^{*22}$ ,  $\hat{N}^{*2}$ , and their derivatives and  $\underline{R}^*$  and  $\hat{M}^{*11}$  may all have components in both the  $i_3$  and  $i_2$  directions.  $\hat{N}^{*1}$  has a component in the  $i_1$  direction only.

For slab symmetry and beams, the term  $(\hat{M}^{*11} \cdot \underline{i}_2 + \hat{N}^{*1} \cdot \underline{i}_1)$  is deleted. For beams,  $\gamma*_0$  is mass/unit length and the term in brackets is force/unit length.

The difference scheme used in the standard REPSIL has two defects which have occasionally been troublesome: the use of forward, or backward, differences at fixed edges may introduce a slowly developing instability which cannot be eliminated by a reduction in the time step. (The presence of this instability could be detected by an eigenvalue analysis of the linearized equations of motion. We have carried out this analysis for a few cases with very small grids. Unfortunately, for most useful meshes, the eigenvalue analysis would take longer than the REPSIL program.) The other defect is that the differences for first derivatives are not compact enough. (As an experiment with the initial RPSL1D code, alternate points in a slab symmetric run were at rest or given an initial axial velocity. The points initially at rest remained at rest. The motion of the moving points produced a reasonable solution.) These two defects have been removed from RPSL1D (and from a modified version of REPSIL) by modeling fixed boundaries with artificial external points and by introducing new difference operators which compute and use forces between mesh points as well as at mesh points.

Three sets of central difference operators are used in RPSL1D to compute first and second differences with respect to the Lagrangian coordinate  $\xi = \xi^2$ . To simplify the notation, replace  $\xi^2$  by  $y$  and use the notation  $f_y(n)$  and  $f_{yy}(n)$  to represent the approximations to  $\partial f / \partial \xi^2$  and  $\partial^2 f / (\partial \xi^2)^2$ , respectively, evaluated at the  $n$ 'th mesh point.

The first set of difference operators is used in subroutine GRAD to approximate derivatives of position components, and derivatives of components of displacement increments, at mesh points:

$$f_y(n) = \{f(n+1) - f(n-1)\}/(2 \Delta y), \quad (20)$$

$$f_{yy}(n) = \{f(n+1) - 2 f(n) + f(n-1)\}/(\Delta y)^2. \quad (21)$$

The second set of difference operators is used in subroutine GRAD to approximate the same derivatives at mesh midpoints:

$$f_y(n+\frac{1}{2}) = \{f(n+1) - f(n)\}/\Delta y, \quad (22)$$

$$f_{yy}(n+\frac{1}{2}) = \frac{1}{2}\{f(n+2) - f(n+1) - f(n) + f(n-1)\}/(\Delta y)^2. \quad (23)$$

The third set of central difference operators is used in subroutine MOTION to compute terms for the equations of motion:

$$f_y(n) = \{f(n+\frac{1}{2}) - f(n-\frac{1}{2})\}/\Delta y, \quad (24)$$

$$f_{yy}(n) = \{f(n+1) - 2 f(n) + f(n-1)\}/(\Delta y)^2. \quad (25)$$

These difference operators, which are used at fixed edges as well as at internal points, are all central difference operators. The resulting system of difference equations is stable with the proper choice of time step,  $\Delta t$ . Unfortunately, when the program is run for a beam with a free end at end 2, say  $n = N$ , backward difference operators have to be used at the free end. In GRAD, for a free end, we use:

$$f_y(N) = \{3 f(N) - 4 f(N-1) + f(N-2)\}/(2 \Delta y), \quad (26)$$

$$f_{yy}(N) = \{2 f(N) - 5 f(N-1) + 4 f(N-2) - f(N-3)\}/(\Delta y)^2, \quad (27)$$

$$f_y(N-\frac{1}{2}) = \{f(N) - f(N-1)\}/\Delta y, \quad (\text{as before}) \quad (28)$$

$$f_{yy}(N-\frac{1}{2}) = \frac{1}{2}\{3 f(N) - 7 f(N-1) + 5 f(N-2) - f(N-3)\}/(\Delta y)^2. \quad (29)$$

In MOTION, for a free end, we have:

$$f_y(N) = \{f(N) - f(N-\frac{1}{2})\}/(\Delta y/2), \quad (30)$$

$$f_{yy}(N) = \{2 f(N) - 5 f(N-1) + 4 f(N-2) - f(N-3)\}/(\Delta y)^2. \quad (31)$$

It is also necessary to approximate  $\hat{N}^*{}^2 = -\partial \hat{M}^*{}^{22}/\partial \xi$  at the free end. This is done in subroutine RESULT with

$$\hat{N}^*{}^2(N) = \frac{1}{2}\{4 \hat{M}^*{}^{22}(N-1) - \hat{M}^*{}^{22}(N-2)\}/\Delta \xi, \quad (32)$$

because

$$\hat{M}^*{}^{22}(N) = 0.0. \quad (33)$$

### 3.3 External Points and End Conditions

The end conditions for a fixed or symmetric end is imposed in RFSL1D by the initial positioning and the incrementing of a fictional external point. The following notation is used to explain this:

$\xi$  A vector to a point on the reference surface at time t.  
 $(\xi = R \dot{\xi}_3 + Z \dot{\xi}_2)$

$\xi^+$  A vector to a point on the reference surface at time  $t + \Delta t$ .

$\Delta \xi$   $\xi^+ - \xi$

$\xi_B$  A vector to the end point.

$\xi_I$  A vector to the internal mesh point adjacent to  $\xi_B$ .

$\xi_E$  A vector to the external point adjacent to  $\xi_B$ .

$\eta_B$  The unit normal to the reference surface at  $\xi_B$ .  
 $(\eta_B = n_r \dot{\xi}_3 + n_k \dot{\xi}_2)$

$\xi$  The unit tangent to the reference surface at  $\xi_B$ .  
 $(\xi = -n_k \dot{\xi}_3 + n_r \dot{\xi}_2)$

The initial position of external points is established in subroutine BOUNDR and the corresponding increments are inserted in BOUNDU. The initial normal at clamped end points is supplied in a new user subroutine INNORM.

3.3.1 Hinged Ends. The initial position of an artificial external point at a hinged end is

$$\underline{x}_E = 2 \underline{x}_B - \underline{x}_I . \quad (34)$$

The increment is

$$\Delta \underline{x}_E = -\Delta \underline{x}_I . \quad (35)$$

We also set

$$\Delta \underline{x}_B = 0 . \quad (36)$$

These inserts impose  $\underline{x}_B$  is constant and  $\hat{M}^{22} = 0$  at the boundary. These are the conditions for a hinged end as given in Reference 1.

3.3.2 Clamped Ends. The initial position of an artificial external point adjacent to a clamped end is

$$\underline{x}_E = \underline{x}_I - 2\{(\underline{x}_I - \underline{x}_B) \cdot \underline{z}\} \underline{z} . \quad (37)$$

The corresponding increment is

$$\Delta \underline{x}_E = \Delta \underline{x}_I - 2\{\Delta \underline{x}_I \cdot \underline{z}\} \underline{z} . \quad (38)$$

We also impose

$$\Delta \underline{x}_B = 0 . \quad (39)$$

These inserts force  $\underline{x}_B$  and  $\underline{n}_B$  to remain constant. These are the conditions for a clamped end given in Reference 1. (The components of  $\underline{n}_B$  are set in a user subroutine INNORM and never changed for a clamped end.)

3.3.3 Symmetric Ends. The treatment of symmetric ends is simply a reduction of that for symmetric edges in REPSIL. A symmetry plane is assumed perpendicular to the basis vector  $\underline{i}_2$ . The initial position of an external point adjacent to a symmetry end is

$$\underline{x}_E = \underline{x}_I - 2\{(\underline{x}_I - \underline{x}_B) \cdot \underline{i}_2\} \underline{i}_2 . \quad (40)$$

The change in  $\underline{x}_E$  at each time step is

$$\Delta \underline{x}_E = \Delta \underline{x}_I - 2\{\Delta \underline{x}_I \cdot \underline{i}_2\} \underline{i}_2 . \quad (41)$$

These are the same equations used for clamped ends with  $\underline{i}_2 = \underline{z}$ .

The external points at symmetry ends represent real points. They are not artificial like those outside the fixed ends. Just as in the standard REPSIL, we need to know membrane and bending resultant components ( $\hat{N}^{*2}$  and  $\hat{M}^{*22}$ , respectively) outside the symmetry end. We have

$$(\hat{M}^{*22})_I \equiv \hat{M}^{*22} \underline{n}_I = (FM_r^{22})_I \underline{i}_3 + (FM_k^{22})_I \underline{i}_2 , \quad (42)$$

defined at  $\underline{x}_I$ , and

$$(\hat{N}^{*1})_I = (FN_\theta^1)_I \underline{i}_1 , \quad (43)$$

$$(\hat{N}^{*2})_I = (FN_r^2)_I \underline{i}_3 + (FN_k^2)_I \underline{i}_2 , \quad (44)$$

defined between  $\underline{x}_I$  and  $\underline{x}_B$ . The external values needed are

$$(FM_r^{22})_E = (FM_r^{22})_I , \quad (45)$$

$$(FM_k^{22})_E = -(FM_k^{22})_I , \quad (46)$$

$$(FN_\theta^1)_E = (FN_\theta^1)_I , \quad (47)$$

$$(FN_r^2)_E = -(FN_r^2)_I , \quad (48)$$

$$(FN_k^2)_E = (FN_k^2)_I . \quad (49)$$

**3.3.4 Free Ends.** One of the first problems posed for the RPSL1D code was for a cantilever I-beam. To model this beam a free end was needed. The conditions for a free end (see Ref. 2, Eq. 6.52) are imposed by

$$\hat{M}^{*22} = 0.0 , \quad \hat{N}^{*2} = -\partial \hat{M}^{*22} / \Delta \xi . \quad (50)$$

Regrettably, it is necessary to use non-central differences to approximate this derivative, and other derivatives, at free ends (see Section 3.2).

The cataloged program permits end 2 to be free but not end 1. (The optional coding called APLFRC, see Section 5.4, allows either end to be free with, or without, applied forces. The applied forces, or absence of forces, are supplied by a user's subroutine called ENDFRC.)

### 3.4 Optional Gaussian Integration

The original RPSL1D programming approximated integration through the thickness of shells by Riemann sums assuming equal layers about evenly spaced points. This is replaced with Gaussian integration only if IGAUSS is 1 and IB is not 1. (These control parameters are set in the user subroutine INGEOM, see Section 4.1.) IB = 1 specifies a beam. We do not use Gaussian integration with beams. We use Riemann sums with unevenly spaced points and "weights" equal to their associated cross-sectional area (as explained in Section 2.3). For the Riemann sums with L equal layers we choose

$$\zeta_k = \frac{1}{2} h (1 - (2k - 1) / L), \quad (k = 1, 2, \dots, L), \quad (51)$$

$$\Delta\zeta = h/L, \quad (52)$$

$$\zeta_u = h/2, \quad (53)$$

$$\zeta_l = -h/2, \quad (54)$$

and approximate integration through the thickness by

$$\int_{\zeta_l}^{\zeta_u} f(\zeta) d\zeta \sim \sum_{k=1}^L f(\zeta_k) \Delta\zeta, \quad (55)$$

where  $h$  is the shell thickness,  $\zeta_u$  and  $\zeta_l$  are the distances from the reference surface (the middle surface) to the upper and lower surfaces, respectively.  $L$  is the number of equally spaced layers, and  $\zeta_k$  is the center of the  $k$ 'th layer.

For Gaussian integration, we replace the  $\zeta_k$  by

$$\zeta_k = \frac{1}{2} h x_{kL}, \quad (56)$$

set

$$w_k = w_{kL}, \quad (57)$$

and use the approximation

$$\int_{\zeta_l}^{\zeta_u} f(\zeta) d\zeta \sim (h/2) \sum_{k=1}^K f(\zeta_k) w_k. \quad (58)$$

where the  $x_{kL}$  and  $w_{kL}$  are from tables for Gaussian integration (e.g. Table 25.4, Handbook of Mathematical Functions, National Bureau of Standards). The  $x_{kL}$  are zeros of the Legendre Polynomial

$$P_L(x) = \frac{1}{2^L L!} \frac{d^L (x^2 - 1)^L}{(dx)^L} \quad (59)$$

Two sets of integrals are affected: the integral for strain energy in subroutine STRESS, and the integrals for force and moment resultants in subroutine RESULT.

The Riemann sums for the force and moment resultants at a point  $N$  are of the form

$$I_N = TA \sum_k f_N(\zeta_k), \quad (60)$$

where

$$TA = \Delta\xi = h/L . \quad (61)$$

The  $f_N(\xi_k)$  signifies the value of the appropriate function at mesh point N and integration station  $\xi_k$ , where the  $\xi_k$  are in the center of evenly spaced layers.

All that is needed to change these sums to Gaussian integration is to select the proper  $\xi_k$  and  $w_k$ , replace  $f_N(\xi_k)$  by  $w_k f_N(\xi_k)$ , and set TA to  $h/2$ . For example,

$$I_N = (h/2) \sum_{k=1}^K w_k f_N(\xi_k) . \quad (62)$$

(The form is the same for approximations of integrals over cross-sectional area for beams. This is done with Riemann sums but with unevenly spaced  $\xi_k$ . The  $\xi_k$ , and  $w_k$  equal to the area associated with  $\xi_k$ , are selected in the user subroutine INGEOM, and TA is set to 1.0 in START.)

### 3.5 Convergence and Stability

Some examination of convergence and stability is essential for any numerical solution of differential equations. The usual method of examining convergence is to find solutions for a particular problem using an increasing number of meshes until these solutions converge. Such tests with the REPSIL family of programs have shown remarkable insensitivity to the number of meshes for problems with large plastic deformation. In those cases tested it seems that the maximum deflection, usually at the center, decreases slightly with additional meshes, and the strain and deflection increase slightly near the edges. For elastic vibration problems, more meshes may be needed to permit high vibration modes. The choice of mesh, and hence the accuracy, must ultimately be decided by the user.

Numerical solutions of differential equations by explicit methods such as that used with the REPSIL programs will be unstable, because any introduced error grows, unless the increment in the independent variable,  $\Delta t$  in our case, is small enough. Considerable time and effort have been expended in studying the stability for the various REPSIL codes. Even a cursory discussion of the stability analysis is beyond the scope of this report. The following bounds, based on a linear analysis of the non-linear difference equations of motion for RPSL1D assuming the central difference operators, are used in RPSL1D:

$$\Delta t_M = \Delta\xi/C , \quad (63)$$

$$\Delta t_B = \frac{1}{2}(\Delta\xi)^2/(CC_2) . \quad (64)$$

Here,  $\Delta t_M$  and  $\Delta t_B$  are the stable time step for the linearized membrane and bending equations of motion, respectively. C is the longitudinal wave speed.

$$C = (E/\rho)^{1/2}, \text{ for beams.} \quad (65)$$

$$C = \{E/\rho(1 - v^2)\}^{1/2}, \text{ for shells.} \quad (66)$$

Here, E is Young's modulus of Elasticity,  $\rho$  is mass density, and  $v$  is Poisson's ratio.  $C_2$  is the approximation of  $\{\int \zeta^2 dA / \int dA\}^{1/2}$ .

For beams,

$$C_2 = \{\sum w_k (\zeta_k)^2 / \sum w_k\}^{1/2}. \quad (67)$$

For shells  $h$  units thick, using Riemann sums with L equal layers,

$$C_2 = \{(1 - 1/L^2) h^2/12\}^{1/2}. \quad (68)$$

For shells with Gaussian integration the approximation is exact:

$$C_2 = (h^2/12)^{1/2}. \quad (69)$$

The program chooses a two digit truncated value 0.95 times the minimum of  $\Delta t_M$  and  $\Delta t_B$ . This is used instead of the input  $\Delta t$  on input card 3 if it is smaller than the input value given, or if the input value is negative or zero.

The term  $\Delta\xi$  used by RPSL1D in these equations is the DETA2 supplied by the user through subroutine INGEOM. The equations produce critical values for  $\Delta t_M$  and  $\Delta t_B$  if  $\Delta\xi$  is the minimum distance between mesh points. Both  $\Delta t_M$  and  $\Delta t_B$  have been shown to be close to the limits for stability for runs with fixed ends and a linearly elastic stress-strain relation. The user should examine all results for numerical instability. This shows up first as rapid vibrations in strain plots. The inclusion of plasticity tends to damp these vibrations, but they can still be detected. The user should be particularly critical if there is a free or forced end (non-central differences are used), or if the structure is compressed. The stability formulæ are based on the assumption that  $\Delta\xi$  is the minimum distance between mesh points. The factor 0.95 permits some compression, but only up to 5% compression if  $\Delta t_M$  is critical, or 2.5% if  $\Delta t_B$  is critical. We have not been able to prove the stability of the RPSL1D solutions when non-central difference equations are used, but we have not detected instability when a time step that satisfies the stability criteria is used.

#### 4. USER SUBROUTINES

There are four, or possibly five, subroutines in RPSL1D which the user must either write, or select from existing versions. These "user" subroutines describe the geometry and loading. Three of them, INGEOM, PRESS, and INVEL, are one-dimensional versions of subroutines described in Reference 2. Subroutine INNORM was added to supply the unit normal to the reference curve at clamped ends. Subroutine ENDFRC must be included, to supply the forces and moments on the ends of a beam, if the optional coding APLFRC, which assumes this type of loading (see Section 5.4), is added to the RPSL1D program. Some of these subroutines are always called and some are called under control of input parameters. Any of the user supplied subprograms may require input (this is under the control of the user). The input for them follows all the other input (input is discussed in Section 6). The user subroutines are listed below in the order they are called, with the conditions for calling.

INGEOM is called from START shortly after the start of each run.

INNORM is called immediately after INGEOM from START.

PRESS is called from the initiation part of the main program, RPSL1D, if LOAD  $\neq 0$ . It will be called each time cycle thereafter as long as LPRESS  $\geq$  NCYCLE.

INVEL is called in the initiation sequence in RPSL1D if LOAD  $\leq 0$ .

ENDFRC is called from RESULT each cycle, except cycle 0, if the APLFRC optional coding is included in the program.

Information is brought to each of the user subroutines, and carried back to the rest of the program, through COMMON. The RPSL1D program is intended to be stored in the UPDATE format. All the COMMON is contained in a COMDECK called MAIN which is listed at the beginning of Appendix C. This COMDECK is inserted into a subroutine with the directive \*CALL MAIN. Of course, information can also be passed from one user subroutine to another through labeled COMMON. The parameters in the following list are commonly needed in the user subroutines.

NMESH The number of mesh intervals in the length.

LAYER The number of layers in a cross section.

N1B = 2 The index of the mesh point at end 1.

N2B = N1B + NMESH The index of the mesh point at end 2.

N1V and N2V The range of indices of mesh points that move.

##### 4.1 INGEOM

Subroutine INGEOM is called from START. It supplies the initial geometry and some control parameters. The parameters supplied are listed below.

IB This is the dominant control parameter. It should be set to 1 or 0. If IB  $> 0$ , a beam is assumed. If IB  $< 0$ , it is not.

RADIUS This parameter separates radial symmetry from slab symmetry. If RADIUS > 0, radial symmetry is assumed and the parameter IS is set to 1 in START. Otherwise, IS is set to 2 in START. (RADIUS is superseded by IB. If IB > 0, RADIUS is set to 0 in START.)

DETA1 ( $\Delta\xi^1$ ) This parameter is a multiplier for energy computations. It should be the width of the reference surface. (If IB = 1, DETA1 is set to 1.0 in START. This makes the concept of surface area and beam length interchangeable.)

DETA2 ( $\Delta\xi$ ) This is the distance between mesh points in the material coordinate  $\xi$  ( $\xi^2$  of Ref. 1, and  $n^2$  of Ref. 2). (The stability criteria assume that DETA2 is the minimum distance between mesh points, and for some other coding it is tacitly assumed that  $\xi$  is arc length from end 1.)

R(N), Z(N) (N1B ≤ N ≤ N2B) The position vector of the N'th mesh point on the reference curve is  $\xi_N = R(N) \hat{i}_3 + Z(N) \hat{i}_2$ .

The point  $\xi_N$  corresponds to the material coordinate point  $\xi = (N - 1) DETA2$ .

ETAD2 and ETAG2(I), I=1,NSTRN The previously read parameters ETAD2 and ETAG2(I) may need to be converted to the same units as DETA2.

IGAUSS Set IGAUSS = 1 for Gaussian integration (see Section 3.4).

#### Extra Parameters for Beams

If IB = 1, the cross section of the beam must be described. It is defined as follows:

ZU (z<sub>u</sub>) The distance from the reference surface to the upper surface of the beam.

ZETA(K), K=1,LAYER (z<sub>k</sub>) These are the positions of integration stations through the thickness relative to the reference surface, the neutral axis for bending of the beam.  
(ZETA(1) < 0, ZETA(LAYER) > 0)

W(K), K=1,LAYER (w<sub>k</sub>) The cross-sectional area associated with ZETA(K).

It is assumed that

$$\sum_k w_k = \text{Area of the beam's cross section.}$$

$$\sum_k (z_k)^2 w_k = \text{Moment of inertia.}$$

$$\sum_k \zeta_k w_k = 0 \quad (\text{i.e. } \zeta = \text{zero on the neutral surface}).$$

4.1.1 INGEOM for a Flat Plate or Beam. This is the INGEOM stored with RPSL1D (see Appendix C).

If the run is for slab symmetric motion of an initially flat plate, there is only one input card. The input is simply length, SLABL, and one-half the width, SLABW, as it is for a similar standard REPSIL subroutine. First, the subroutine reads SLABL, SLABW, and IB with FORMAT (2E10.4,IS). The program sets

```
RADIUS = 0.0 ,
DETA1 = 2.0 * SLABW ,
DETA2 = SLABL/FLOAT(NMESH) ,
R(N) = 0.0
Z(N) = (N - N1B) DETA2 } N1B < N <= N2B ,
IGAUSS = 1.
```

If IB < 0, the responding surface is an initially flat plate. The subroutine is finished.

If IB > 0, data describing the cross section of the beam is needed. First, the subroutine reads ZU with FORMAT (3E10.2) and prints SLABL and ZU. It then reads WIDTHK, DZETAK, and ZETA(K) with FORMAT (3E10.2), sets W(K) = WIDTHK \* DZETAK, and writes K, WIDTHK, DZETAK, ZETA(K), and W(K) for K = 1, 2, ..., LAYER.

The following table is an example for a 6-layer subdivision of the cross section of a solid, right-circular, cylindrical rod of unit radius:

K	WIDTHK	DZETAK	ZETA(K)
1	1.032494026	1/3	-0.8253794582
2	1.717575183	1/3	-0.4952276749
3	1.962319769	1/3	-0.1650758916
4	1.962319769	1/3	0.1650758916
5	1.717575183	1/3	0.4952276749
6	1.032494026	1/3	0.8253794582

To produce this table we first arbitrarily chose layers of equal thickness, DZETAK = 1/3. Then, we found WIDTHK so that W(K) = DZETAK \* WIDTHK would equal the area of the circular cross section in that layer. Then we set  $Z_K = K/3 - 7/6$  as a tentative location in the center of each layer. Finally we found  $ZETA(K) = c Z_K$  where  $c^2 \sum (Z_K)^2 W(K) = \pi/4$ , the area moment of inertia for a circle of unit radius.

Similarly, for a rectangular beam with unit thickness and width w:

K	WIDTHK	DZETAK	ZETA(K)	Z <sub>K</sub>
1	w	1/6	-0.4225771	-5/12
2	w	1/6	-0.2535462	-3/12
3	w	1/6	-0.0845154	-1/12
4	w	1/6	0.0845154	1/12
5	w	1/6	0.2535462	3/12
6	w	1/6	0.4225771	5/12

(Choosing integration stations and weights for Gaussian integration, see Section 3.4, would probably be better for a rectangular beam.)

The choice of layers of equal thickness is convenient and reasonable for these two examples, but not for all cases. With an I-beam, for example, it would be more reasonable, and convenient, to approximate each flange with a layer and put four layers in the web. If the beam is not symmetric vertically it may be difficult to assign values that produce the correct area moment of inertia without altering the area, or the location of the centroid.

**4.1.2 INGEOM for a Cylindrical Shell.** The subroutine INGEOM for a cylinder is very simple (see the tabulation in Appendix F). The input is CYLL, RADIUS with FORMAT (2E12.6), where CYLL is the cylinder length and RADIUS is the radius to the midsurface. The subroutine sets

```

DETA1 = 2 π RADIUS
DETA2 = CYLL/FLOAT(NMESH)
IB = 0
R(N) = RADIUS
Z(N) = (N - 2) * DETA2

```

If Gaussian integration is desired, IGAUSS = 1.

#### 4.2 PRESS

Subroutine PRESS supplies the pressure, P(N), at each mesh point that moves ( $N1V \leq N \leq N2V$ ). (this is actually the pressure difference for shells and the normal force per unit length for beams, but we will continue to call it pressure.) The sign of P(N) was chosen so that a positive P(N) tends to crush a cylinder or push a flat plate down.

Most of the PRESS subroutines for RPSL1D have been in two sections. The first section reads input and makes other preliminary calculations the first time the subroutine is called. The other part supplies the P(N) each time the subroutine is entered.

PRESS is called in the initial portion of RPSL1D if LOAD = 0. It is called at the start of each succeeding time cycle in RPSL1D if LPRESS  $\geq$  NCYCLE. Also, as long as LPRESS  $\geq$  NCYCLE, the P(N) are multiplied by  $a^{\frac{1}{2}}$  in subroutine DGEOM. This multiplication changes the P(N) from pressure to force per unit initial area. If the program is continued beyond cycle NCYCLE = LPRESS, the values of P(N) do not change. This means the force per unit initial area is fixed, the pressure varies with the change in area. (At the initiation of

damping, when NCYCLE = MDAMP, subroutine DAMP sets LPRESS and all P(N) to zero. This must be changed if the program is to continue with both pressure and damping.) NCYCLE is the count of time cycles. LOAD, LPRESS, and MDAMP are input parameters. If LOAD = 0 and LPRESS = 0, PRESS will never be called and all P(N) will be zero.

One of the weak points of using deformation codes to simulate reality is in accurately describing dynamic pressure loading. It is difficult to obtain accurate, repeatable measurements for enough points even with well-controlled experiments. Such measurements tend to be very oscillatory due to both reality and instrumentation. Even assuming a completely accurate pressure record at some point, the effective pressure on a thin shell would be different because of its rapid reaction. The best we can hope for is an approximate simulation that will lead to useful results. Three examples are discussed.

**4.2.1 PRESS for Constant Pressure.** This is the PRESS subroutine with the program in Appendix C. In the initial entrance for each run, the constant pressure, PO, is read with FORMAT (E12.6). During each entrance it sets P(N) = PO for  $N_{1V} \leq N \leq N_{2V}$ .

**4.2.2 PRESS for Pressure as a Function of Time.** This subroutine, called LINPRS, is tabulated with Example 1 in Appendix E. Pressure is a tabular function of time. On the first entrance, this subroutine reads pairs of time and pressure data into TPR(I) and PPR(I) with FORMAT (2E10.3) until  $TPR(I) < 0$ . Then IPRESS, the number of data points in the table, is set to I-1. (It is assumed that IPRESS  $\leq 50$ ,  $TPR(1) = 0.0$ ,  $TPR(I+1) > TPR(I)$ , and  $TPR(IPRESS)$  is larger than any time to be reached.

Linear interpolation is used at every entry to compute the pressure, PO, corresponding to time, TIME. The program then sets P(N) = PO for  $N_{1V} \leq N \leq N_{2V}$ .

**4.2.3 PRESS for Two Phase Pressure Decay in a Cylinder.** This subroutine, labeled PRESS of 3/10/76, is listed in Appendix F. It simulates the pressure in a cylinder from an explosion at its center. This rather involved subroutine is included because it demonstrates a number of the common features of pressure subroutines that simulate the loading from explosions. The unbalanced pressure  $P_N$  at point N is zero until the shock front arrives at time  $TA_N$  with peak pressure  $PO_N$ . The pressure then decays rapidly. In this subroutine it decays rapidly, exponentially until it reaches a pressure  $\bar{P}$  which we call the quasi-static pressure. It then continues to decay exponentially at a much slower rate.

$$\begin{aligned} P_N &= 0.0 && \text{if } t < TA_N , \\ P_N &= PO_N e^{-\alpha(t - TA_N)} && \text{if } TA_N \leq t \leq TB_N , \\ P_N &= \bar{P} e^{-\beta(t - TB_N)} && \text{if } TB_N \leq t , \end{aligned} \quad (70)$$

where  $\alpha$  and  $\beta$  are the decay factors and  $\bar{P}$  is the quasi-static pressure which is reached at point N at time  $TB_N$  found from

$$\bar{P} = P_{0N} e^{-\alpha(TB_N - TA_N)} .$$

An outline of the subroutine is given below. First, the following input cards are read on the initial entry.

<u>Content</u>	<u>FORMAT</u>
$\bar{P}$ , $\beta$ , (PID(I), I=1,6)	(2E10.3,6A10)
CZ, CT, CP, TD	(4E10.3)
$\alpha$ , ---, ISM	(E10.3,10X,I10)
(ZSI(I), TSI(I), PSI(I), I=1,ISM)	(3E10.3)

Here,

$\bar{P}$  is the quasi-static pressure.

$\beta$  is the quasi-static decay factor.

PID(I) is an alphanumeric title.

CZ, CT, CP, and TD are conversion factors to transform data to the desired units (see below).

$\alpha$  is the shock pressure decay factor.

ISM is the number of entries in the following table (ISM < 50).

ZSI(I) is the distance from the middle, end 1, of the cylinder.

TSI(I) is the shock arrival time at ZSI(I).

PSI(I) is the peak shock pressure at ZSI(I).

These data cards are read, and their images printed, during the initial entrance. The tabular data is then converted to the desired units with the replacement formulas

$$ZSI(I) = CZ * ZSI(I) , \quad (71)$$

$$TSI(I) = CT * (TSI(I) - TD) , \quad (72)$$

$$PSI(I) = CP * PSI(I) . \quad (73)$$

This converted table is also printed. For  $N1V \leq N \leq N2V$ ,  $TS(N) \equiv TA_N$  and  $PS(N) \equiv P_{0N}$  are found by linear interpolation at distance  $Z(N) = (N - 2) * DETA2$ . (Here is an example where the material parameter  $\xi$  is assumed to be length from end 1. Symmetry is assumed at end 1 where  $N = N1B = N1V = 2$ .)  $TSB(N) = TB_N$  is found from

$$TSB(N) = TS(N) + \{\ln(PS(N)/\bar{P})\}/\alpha . \quad (74)$$

The program then forms PSB(N) and transforms PS(N) to a more convenient form, as follows:

$$PS(N) = PS(N) e^{\alpha TS(N)} \quad (75)$$

$$PSB(N) = \bar{P} e^{\beta TSB(N)} . \quad (76)$$

This completes the preparation. For every entry, the subroutine computes  $e^{-\alpha t}$  and  $e^{-\beta t}$  and then, for  $N1V < N < N2V$ ,

$$\begin{aligned} P(N) &= 0.0 && \text{if } t < TS(N) , \\ P(N) &= -e^{-\alpha t} PS(N) \equiv -P_0 N e^{-\alpha(t - TA_N)} && \text{if } TS(N) < t < TSB(N) , \\ P(N) &= -e^{-\beta t} PSB(N) \equiv -\bar{P} e^{-\beta(t - TB_N)} && \text{if } TSB(N) < t . \end{aligned} \quad (77)$$

#### 4.3 INVEL

Subroutine INVEL supplies the initial velocity of moving mesh points. The velocity of all mesh points is preset to zero before INVEL is called. This subroutine sets

$$DR(N) = \dot{R} , \quad (78)$$

$$DZ(N) = \dot{Z} , \quad (79)$$

where the velocity at point N is

$$\underline{v} = \dot{R} \underline{i}_3 + \dot{Z} \underline{i}_2 . \quad (80)$$

INVEL is called during the initiation sequence from the main program, RPSL1D, if parameter LOAD is less than or equal to zero. The components of initial velocity are changed to displacement increments by multiplication with  $\Delta t$  in RPSL1D.

**4.3.1 INVEL for Normal Velocity at Specified Points.** The INVEL subroutine catalogued with RPSL1D (see Appendix C) is a one-dimensional version of the INVEL tabulated in the User's Manual (Reference 2). This subroutine assumes all initial velocities are normal to the surface. The input cards are:

<u>Cards</u>	<u>FORMAT</u>
--,--,NI,NF,VR,NV	(10X,2I5,E12.6,I5)
--, N, V	(5X,I5,E12.6)

NI and NF are the minimum and maximum mesh numbers for an array of points to be given initial velocity VR. NV is the number of mesh points to receive initial velocity different from zero or VR. N is the mesh number for a point receiving velocity V.

Note! This subroutine was written to use input cards for the standard REPSIL. The mesh numbers NI, NF, and N are all relative to the mesh point at end 1 being number 1. In RPSL1D, the mesh point at

end 1 is numbered N1B. (Subroutine START sets N1B = 2.) Therefore, this INVEL subroutine adds N1B-1 (i.e. one) to all input mesh numbers.

The first input card is read and the program sets

$$DR(N) = -SNR(N) * VR , \quad (81)$$

$$DZ(N) = -SNK(N) * VR , \quad (82)$$

for  $N1 + N1B-1 \leq N \leq NF + N1B-1$ , where the unit normal at mesh point N is

$$\hat{n} = SNR(N) \hat{i}_3 + SNK(N) \hat{i}_2 .$$

(Notice the reversal in sign. VR is assumed to be an inwardly directed velocity, possibly from impulsive loading, on the outside surface.)

The program then reads NV cards of the second type (NV may be zero), replaces N by  $N + N1B-1$ , and sets

$$DR(N) = -SNR(N) * V , \quad (83)$$

$$DZ(N) = -SNK(N) * V . \quad (84)$$

The subroutine prints the initial velocities assigned. These initial velocities are later converted to displacement increments by multiplying them by  $\Delta t$ .

4.3.2 INVEL for Constant Lateral Velocity. This subroutine has been used to simulate a rod striking a wall. The changes in the catalogued INVEL to produce this are tabulated with Example 2 in Appendix E.

The constant velocity, VR, is read with effective FORMAT (20X,E12.6) and the subroutine sets

$$DZ(N) = -VR \quad (85)$$

for  $N1V \leq N \leq N2V$ .

4.3.3 INVEL for Beam Vibration in the Fundamental Transverse Mode. A number of INVEL subroutines have been coded for beams. They were either very simple or very specialized. As examples of simple codes for initially flat beams, consider the following for the fundamental mode of free transverse vibrations. The transverse velocity of any point x at time t may be written

$$\dot{R}(x,t) = A X(x) \cos(\omega t) \quad (86)$$

for these examples. The velocity at time  $t = 0$  is

$$\dot{R}(x,0) = A X(x) . \quad (87)$$

Here  $\omega$  is the fundamental frequency,  $X(x)$  is a function of the end conditions, and the parameter  $A$  is arbitrary. We assume evenly spaced points to find  $x$ . (These formulas are similar to those given by Enrico Volterra, E. C. Zachmanoglou, Dynamics of Vibration, Charles E. Merrill Books, Inc., Columbus, Ohio.)

Cantilever Beam:

$$x = 1.875104069 (N - N1B)/(N2B - N1B) , \quad (88)$$

$$DR(N) = A \{cosh(x) - cos(x) - 0.7340955137 [sinh(x) - sin(x)]\} . \quad (89)$$

Simply Supported Beam:

$$x = \pi(N - N1B)/(N2B - N1B) , \quad (90)$$

$$DR(N) = A \sin(x) . \quad (91)$$

Clamped Beam:

$$x = 4.730040744862 (N - N1B)/(N2B - N1B) , \quad (92)$$

$$DR(N) = A \{cosh(x) - cos(x) - 0.9825022145 [sinh(x) - sin(x)]\} . \quad (93)$$

#### 4.4 INNORM

Subroutine INNORM is a new user-supplied subroutine that assigns the unit normal at clamped ends. It is called in START immediately after the call of INGEOM. The version of INNORM catalogued with the program in Appendix C is the only one used up to now. It produces the correct normal if the shape of the reference curve at the end can be written as a quadratic equation in  $\xi$  through the last three end points.

The components of the unit normal,  $\underline{n}$ , are defined in terms of partial derivatives of the position vector  $\underline{x}$ .

$$\underline{x} = R \underline{i}_3 + Z \underline{i}_2 , \quad (94)$$

$$R2 = \partial R / \partial \xi , \quad (95)$$

$$Z2 = \partial Z / \partial \xi , \quad (96)$$

$$D = \{(R2)^2 + (Z2)^2\}^{1/2} , \quad (97)$$

$$SNR(N) = Z2/D , \quad (98)$$

$$SNK(N) = -R2/D , \quad (99)$$

$$\underline{n} = SNR(N) \underline{i}_3 + SNK(N) \underline{i}_2 . \quad (100)$$

The partial derivatives,  $R2$  and  $Z2$ , at the clamped ends are approximated by non-central differences through three end points.  $N$  is  $N1B$  or  $N2B$ , depending on which end is clamped.

$$(e.g. \left. \frac{\partial R}{\partial \xi} \right|_{N=N1B} = \{3R(N2B) - 4R(N2B - 1) + R(N2B - 2)\} / (2\Delta\xi) .) \quad (101)$$

#### 4.5 ENDFRC

Subroutine ENDFRC is a user subroutine that supplies the forces and moments imposed at the ends of a beam. It is really a part of the option called APLFRC (Section 5.4) which was programmed to utilize this type of loading. The subroutine is not needed unless the APLFRC option is included in the program and either IBCE1 = 4 or IBCE2 = 4. With the APLFRC option, IBCE1 = 4 signals an applied force and moment at end 1 and IBCE2 = 4 signals an applied force and moment at end 2. A free end is a special case for which the applied force and moment are zero.

Subroutine ENDFRC is called from subroutine RESULT when N = N1B if IBCE1 = 4, or when N = N2B if INCE2 = 4 and IBCE1 ≠ 4. Since RESULT is not called in the initial sequence, there are no initial forces on the ends.

When ENDFRC is called, the force components and moments are supplied. The force at end 1 is

$$EFR1 \hat{z}_3 + EFZ1 \hat{z}_2 . \quad (102)$$

The moment is EM1. Similarly at end 2, the force components and moment are EFR2, EFZ2, and EM2, respectively. Figure 4.5 and the following list both explain the sign conventions.

- |          |   |
|----------|---|
| EFR1 > 0 | decreases R(N1B) ,                          |
| EFZ1 > 0 | decreases Z(N1B) ,                          |
| EM1 > 0  | decreases R(N1B) and increases R(N1B + 1) , |
| EFR2 > 0 | increases R(N2B) ,                          |
| EFZ2 > 0 | increases Z(N2B) ,                          |
| EM2 > 0  | increases R(N2B - 1) and decreases R(N2B) . |

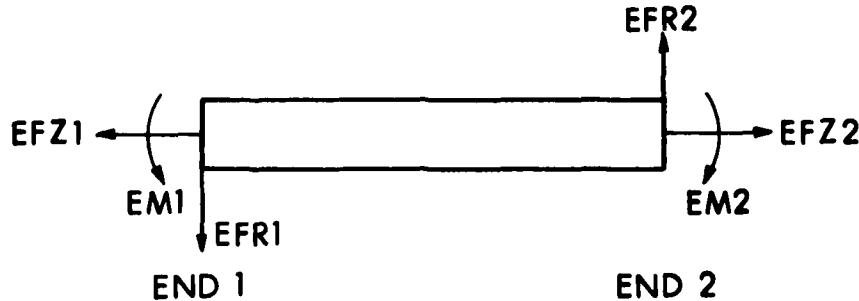


Figure 4.5 Sign Convention for Applied Forces on the End of a Beam.

The program outside ENDFRC does not change these terms. Constant values may be inserted in the initial entry and need not be reset. It does no harm to insert values at an end that is not forced ( $IBCE1 \neq 4$  or  $IBCE2 \neq 4$ ); the program ignores them.

**4.5.1 ENDFRC for One or Both Ends Free.** The initial entry into this ENDFRC sets  $EFR1$ ,  $EFZ1$ ,  $EM1$ ,  $EFR2$ ,  $EFZ2$ , and  $EM2$  to zero. These are never changed. The APLFRC option then imposes a free end at end 1 if  $IBCE1 = 4$ , and/or a free end at end 2 if  $IBCE = 4$ . This subroutine is tabulated in Appendix F.

**4.5.2 ENDFRC for Tabular Forces.** This ENDFRC subroutine assumes that end 2 is free and the forces on end 1 are supplied by tables. On the initial entry this subroutine reads three tables. Each has a heading giving the number of entries and an alphanumeric title.

<u>Cards</u>	<u>FORMAT</u>
NFZPTS, (LABELV(I), I=1,7)	(I10,7A10)
(EFZT(I), TMFZ(I), I=1,NFZPTS)	(2E15.7)
NFTHS, (LABELV(I), I=1,7)	(I10,7A10)
(EFTHT(I), TMTH(I), I=1,NFTHS)	(2E15.7)
NFMUS, (LABELV(I), I=1,7)	(I10,7A10)
(EFMUT(I), TMMU(I), I=1,NFMUS)	(2E15.7)

The first table is the horizontal force component,  $EFZ1$ , as a function of time. The other two tables are of parameters  $\theta$  and  $\mu$ , respectively, as functions of time.

The initial entrance into the subroutine reads and prints the input and sets  $EFZ2 = EFR2 = AM2 = 0$ . On each entry, the subroutine finds  $EFZ1$ ,  $\theta$ , and  $\mu$  as functions of time by linear interpolation. It then sets

$$EFR1 = \mu EFZ1 , \quad (103)$$

$$EM1 = -ZU \{ \sin(\theta) EFZ1 + [1 - \cos(\theta)] EFR1 \} . \quad (104)$$

**4.5.3 ENDFRC for Tabular Forces (Modified).** This subroutine is like the previous one except that the force on end 1 is assumed relative to the tangent,  $\xi$ , at end 1,

$$\xi = SNK(N1B) \dot{\xi}_3 - SNR(N1B) \dot{\xi}_2 , \quad (105)$$

rather than the horizontal direction,  $\dot{\xi}_2$ . The force components for that subroutine are replaced by

$$EFR1 = EFR1 * SNR(N1B) - EFZ1 * SNK(N1B) , \quad (106)$$

$$EFZ1 = EFR1 * SNK(N1B) + EFZ1 * SNR(N1B) . \quad (107)$$

A slightly modified version of this is tabulated with example 2 in Appendix E.

## 5. OPTIONAL CHANGES

As has been stated before, it seems that any particular project for RPSL1D requires some changes in the program. Ideally, these changes are restricted to the user subroutines, discussed in the previous section, which describe the geometry and loading. In this section we will briefly discuss the seven sets of optional coding that are used with the two examples in Appendix E. They are of varying utility. The plotting option PLOTP is usually wanted. It was not added to the catalogued program because it is very short and usually requires some additional coding to select the functions of time to be plotted. The APLFRC option which permits loading the ends of beams by force and moment has been used a number of times. If it is used, the option SHR3/1 must be included and the energy computation EAPFRC should be added.

It is hoped that these options will be useful and serve as a guide for introducing others in the future.

### 5.1 PLOTP (Plot Functions of Time)

This coding was originally inserted to plot the force/unit initial area (force/unit initial length, for beams) at prescribed mesh points as functions of time. However, any function of time may be plotted by inserting coding that stores that function's value into  $P(N)$  ( $N_{2B} < N < 104$ ) each cycle, and requesting a plot for that  $P(N)$  on the new input card, card 11a. If the PLOTP option is in the deck, this input card must be included (see Section 6). It selects  $NNPE$  ( $0 < NNPE < 9$ ) values of  $N$  for which plots of  $P(N)$  are to be made.

The changes to RPSL1D are rather trivial: some new terms are inserted into COMMON and MAIN, the new input card is read and its image is printed in START, and the requested  $P(N)$  are stored in the plot data file in PDATA. More extensive changes were required in the plotting program. These changes are included in the plotting program stored in file RPSL1DPLOT except for two orders (the optional coding) which read from the plot data file and activate  $P(N)$  plotting. This option is used with both examples in Appendix E. (In example 1 the total large core memory requested for plotting is greater than 131,071 words. Hence, the LCM = I parameter is specified on the FTN control card.)

### 5.2 SHR3/1 (Compute Moment, Shear and Axial Forces)

This option is for beams only. It computes the moment,  $M$ , at each mesh point and computes the shear force,  $V$ , and the axial force,  $Q$ , between mesh points. The following equations were extracted from Reference 1 by Dr. Santiago.

$$M = a \hat{M}^{22} \quad (108)$$

$$Q = a (\hat{Q}^{22} - b_2^2 \hat{M}^{22}) \quad (109)$$

$$V = (1/a^{1/2}) \partial(a \hat{M}^{22}/\partial\xi^2) \quad (110)$$

The RPSL1D analog of these equations is

$$AMS(N) = A22 * F22 \quad (111)$$

$$QS(N) = A22 * (Q22 - BM22 * F22) \quad (112)$$

$$VS(N) = \{AMS(N+1) - AMS(N)\}/(DETA2 * SRA) . \quad (113)$$

For convenience, the three new arrays are put in COMMON in COMDECK MAIN. The rest of this option is in subroutine RESULT. A table of N, AMS(N), VS(N), QS(N), A22, B22, Q22, and F22 may be printed at cycles prescribed by coding inserted with the option.

### 5.3 MSQSVS (Plot Moment, Shear and Axial Forces)

This option was inserted to plot the moment, shear force, and axial force computed with option SHR3/1 on the same cycles that cross-sectional plots are made (i.e. whenever NCYCLE = NC3DP(I)). The only change in RPSL1D is one statement in PDATA that stores the three arrays AMS(N), QS(N), and VS(N) on the plot data file. The bulk of this option is inserted into the plotting program catalogued in file RPSL1DPLOT.

### 5.4 APLFRC (Applied Forces at Beam Ends)

This option was inserted to permit an applied vector force and moment at either or both ends of a beam. We will refer to this as a forced end. A free end is a special case with zero applied force and moment. It is assumed that the option SHR3/1 is included in the program.

The equation of motion, given in Reference 1, reduced to that of a beam with no external forces is

$$\Delta \ddot{x}^+ = \Delta \ddot{x} + \left\{ \hat{M}^{*22},_{22} + \hat{N}^{*2},_2 \right\} (\Delta t)^2 / (A^{1/2} \Gamma_0) , \quad (114)$$

where

$$\hat{M}^{*22} = a^{1/2} \hat{M}^{22} \underline{n} \quad (115)$$

and

$$\hat{N}^{*2} = a^{1/2} (\hat{Q}^{22} \underline{z}_2 + \Gamma_{22}^2 \hat{M}^{22} \underline{n}) \quad (116)$$

are vectors defining the bending and stress resultant terms. The notation  $X,_{22}$  and  $X,_{22}$  denotes the partial derivatives  $\partial X/\partial\xi$  and  $\partial^2 X/(\partial\xi)^2$ . The mass per initial unit length for a beam is given by  $A^{1/2} \Gamma_0 = A^{1/2} \rho \sum w_k$ .

From the equations for shear force, axial force, and moment, respectively,

$$V = a^{\frac{1}{2}} (\hat{M}^{22})_{,2} + 2\Gamma_{22}^2 \hat{M}^{22} = (a \hat{M}^{22})_{,2}/a^{\frac{1}{2}}, \quad (117)$$

$$Q = a (\hat{Q}^{22} - b_2^2 \hat{M}^{22}), \quad (118)$$

$$M = a \hat{M}^{22}, \quad (119)$$

and the relations

$$a^{\frac{1}{2}}_{,2} = \Gamma_{22}^2 a^{\frac{1}{2}}, \quad (120)$$

$$\underline{n}_{,2} = -b_2^2 \underline{z}_2, \quad (121)$$

$$\underline{z}_2 = a^{\frac{1}{2}} \underline{\xi}, \quad (122)$$

which are also taken from Reference 1, we can transform the vector

$$(\hat{M}^{*22})_{,22} + (\hat{N}^{*2})_{,2} \quad (123)$$

into the form

$$(V \underline{n} + Q \underline{z})_{,2} \quad (124)$$

(The shear force and axial force, V and Q, are computed at all midmesh points, and the moment, M, is computed at all mesh points, except at forced ends, by option SHR3/1.) Vector  $\underline{n}$  is the unit normal to the reference surface and  $\underline{\xi}$  is the unit tangent.

$$\underline{n} = n_k \underline{i}_2 + n_r \underline{i}_3. \quad (125)$$

$$\underline{\xi} = n_r \underline{i}_2 - n_k \underline{i}_3. \quad (126)$$

The force vector

$$\underline{F} = (V \underline{n} + Q \underline{z}) \quad (127)$$

can be computed at all midmesh points if the moment  $M = a \hat{M}^{22}$  is known at the mesh points.

The APLFRC option assumes a new user subroutine, ENDFRC (see Section 4.5), which supplies M and  $\underline{F}$  at forced ends. In the following, we use the subscript N to denote quantities at the N'th mesh point, N = N1B at end 1 and N2B at end 2, and subscript  $N+\frac{1}{2}$  to denote midmesh quantities. If end 1 is forced,

$$\underline{F}_{N1B} = EFZ1 \underline{i}_2 + EFR1 \underline{i}_3, \quad M_{N1B} = EM1. \quad (128)$$

If end 2 is forced,

$$\underline{F}_{N2B} = EFZ2 \underline{i}_2 + EFR2 \underline{i}_3 , M_{N2B} = EM2 . \quad (129)$$

For  $N1B < N < N2B$ ,

$$\underline{F}_{N+\frac{1}{2}} = (V_{N+\frac{1}{2}} \underline{n}_{N+\frac{1}{2}} + Q_{N+\frac{1}{2}} \underline{\tau}_{N+\frac{1}{2}}) \quad (130)$$

Then, the force per unit length at  $N = N1B$  and  $N1B+1$  if end 1 is forced, and at  $N = N2B$  and  $N2B-1$  if end 2 is forced, is found by the difference formulas:

$$(\underline{F},_2)_{N1B} = (\underline{F}_{N1B+\frac{1}{2}} - \underline{F}_{N1B})/(\Delta\xi/2) , \quad (131)$$

$$(\underline{F},_2)_{N1B+1} = (\underline{F}_{N1B+3/2} - \underline{F}_{N1B+\frac{1}{2}})/(\Delta\xi) , \quad (132)$$

$$(\underline{F},_2)_{N2B-1} = (\underline{F}_{N2B-\frac{1}{2}} - \underline{F}_{N2B-3/2})/(\Delta\xi) , \quad (133)$$

$$(\underline{F},_2)_{N2B} = (\underline{F}_{N2B} - \underline{F}_{N2B-\frac{1}{2}})/(\Delta\xi/2) . \quad (134)$$

The first and last of these are not central differences. The form

$$(\underline{F},_2)_N = (\underline{M}^{*22},_{22} + \underline{N}^{*2},_2) , \quad (135)$$

where only central differences are employed, is used at all other moving mesh points (see Section 3.2).

This option includes minor changes in MAIN, START, BOUNDR, and BOUNDU, and more extensive changes in GRAD, RESULT, and MOTION, plus the user supplied subroutine ENDFRC. Option SHR3/1 must also be used, and option EAPFRC should be.

### 5.5 EAPFRC (Energy for APLFRC)

In the section for APLFRC we described the option which allows forces to be applied at either end of a beam. The optional coding labeled EAPFRC, which computes the work on the beam by the applied forces and moments, is properly a part of the APLFRC option.

The work by a directed force,  $\underline{F}$ , applied at a point that moves a vector distance  $\Delta\underline{r}$  is

$$\Delta W_F = \underline{F} \cdot \Delta\underline{r} . \quad (136)$$

The work by a moment of magnitude  $M$  applied while the normal vector changes by an angle  $\Delta\theta$  is

$$\Delta W_M = \pm M \Delta\theta . \quad (137)$$

(The sign in this equation depends on the sign convention used.)

This work must be compatible with work from pressure loading (see Reference 2, pages 45 and 46), so an analogous method is used in RPSL1D. That is, the external work from time  $t-\Delta t$  to time  $t$ ,  $\Delta W_F(t-\frac{1}{2}\Delta t)$  and  $\Delta W_M(t-\frac{1}{2}\Delta t)$ , will be the average of "work increments" computed at times  $t-\Delta t$  and  $t$ . We compute:

$$\Delta W_F(t) = \underline{F}(t) \cdot \{\underline{\Delta r}(t-\frac{1}{2}\Delta t) + \underline{\Delta r}(t+\frac{1}{2}\Delta t)\}/2 , \quad (138)$$

$$\Delta W_F(t-\frac{1}{2}\Delta t) = \{\Delta W_F(t-\Delta t) + \Delta W_F(t)\}/2 , \quad (139)$$

$$\Delta W_M(t) = - M(t) \cdot \{\Delta \theta(t-\frac{1}{2}\Delta t) + \Delta \theta(t+\frac{1}{2}\Delta t)\}/2 , \quad (140)$$

$$\Delta W_M(t-\frac{1}{2}\Delta t) = \{\Delta W_M(t-\Delta t) + \Delta W_M(t)\}/2 , \quad (141)$$

at the forced ends.

Except for some preliminary set up, all the computation is carried out in MOTION. The values of  $\Delta W_F(t-\Delta t)$ ,  $\Delta W_M(t-\Delta t)$ ,  $\Delta \theta(t-\frac{1}{2}\Delta t)$ , and the components of  $\underline{\Delta r}(t-\frac{1}{2}\Delta t)$  at any forced end, is saved from the previous cycle. The components of  $\underline{\Delta r}(t+\frac{1}{2}\Delta t)$  are already stored in the arrays DR(N) and DZ(N) by MOTION. We need to compute  $\Delta \theta(t+\frac{1}{2}\Delta t)$ . We define  $\Delta \theta(t+\frac{1}{2}\Delta t)$  by

$$\sin \{\Delta \theta(t+\frac{1}{2}\Delta t)\} \underline{i}_1 = \underline{n}(t) \times \underline{n}(t+\Delta t) . \quad (142)$$

The unit normal at time  $t$  is available,

$$\underline{n}(t) = SNK(N) \underline{i}_2 + SNR(N) \underline{i}_3 , \quad (143)$$

but  $\underline{n}(t+\Delta t)$  must be computed. This is computed from

$$\underline{a}_1 = \underline{i}_1 , \quad (144)$$

$$\underline{a}_2 = \partial \underline{r}^+ / \partial \xi , \quad (145)$$

$$\underline{n}(t+\Delta t) = (\underline{a}_1 \times \underline{a}_2) / |\underline{a}_1 \times \underline{a}_2| , \quad (146)$$

where

$$\underline{r}^+ = \underline{r}(t) + \Delta \underline{r}(t+\frac{1}{2}\Delta t) \quad (147)$$

and  $\underline{a}_2$  is approximated by a forward or backward difference.

## 5.6 EROD (Erosion at a Beam End)

This option depends on the applied force option, APLFRC, being in RPSL1D. The basis for our model is to assume an end mesh point, denoted by N = N1B, which moves according to forces on the true, nearby, eroding end. Whenever enough erosion has occurred, the end mesh point is shifted

to the next mesh point. (The eroding end cannot approach the midpoint of the first mesh too closely, or the difference equations at that end would be unreliable.)

We inserted a new subroutine, ERODE, which supplies the distance, DE, from the mesh point N1B to the eroding end and advances the endpoint indices when necessary. DE is zero initially. Whenever DE is greater than  $\Delta\xi/4$ , we subtract  $\Delta\xi$  from DE and add 1 to N1B, N1V, and N1A (these are all indices for the end point). The difference equations for the shear in RESULT and force/unit length on the end point in MOTION were changed to reflect the position of the eroding end. The ENDFRC subroutine was altered to use quadratic interpolation to find the normal at the eroding end. (This last change was not really tested since the rod remained straight in our application.) Several minor inserts were made to permit reasonable plots.

This option is not as well tested as other features of RPSL1D. This brief discussion is included because the option is used with Example 2 (Appendix E). The motivation for Example 2 was to demonstrate that the RPSL1D program could reproduce experimental strain records at two points along a long rod penetrator, assuming a stress-strain curve, and a table of force on the end and position of the end as functions of time, all derived from the Karmann-Taylor theory<sup>3</sup>. From the two strain records, the density, and the cross-sectional area of the rod, one can derive the stress and force for any strain, and the velocity at which the strain is propagated. From this velocity, one has a linear relation between initial time and position of the generating force for each strain. Two quite different tables of erosion and force as functions of time were generated, one for a slow erosion rate and the other for a rapid erosion rate. Both were used as input for RPSL1D, and both produced strain records in fair agreement with the test data.

### 5.7 BSTRS (Stress-Strain Option for Beams)

The optional coding labelled BSTRS is a stress-strain routine for beams, which models the uniaxial curve more smoothly than the mechanical sublayer model by using more points. The mechanical sublayer model in the catalogued routine is generally superior, but the number of segments in the polygonal stress-strain curve must be restricted.

In the erosion problem discussed in Section 5.6, we wanted to compare the experimental strain-time response in a bar with output from RPSL1D. The strain response from RPSL1D at points along a bar agree closely with those predicted by the Karmann-Taylor theory. That is, the velocity at which a particular strain level is propagated down the bar is proportional to the square root of the slope of the stress-strain curve at that strain. Therefore, the strain-time records from RPSL1D at points away from the end of the rod, in response to an increasing force at the end, have near discontinuities corresponding to the corners of the polygonal stress-strain curve where there are discontinuities in the slope. To remove these discontinuities, one needs

<sup>3</sup>A.D. Gupta and J.D. Wortman, "An Eroding Long Rod Penetrator Model for Hard Target Penetration," Proceedings of the Third ASCE/EMD Specialty Conference, September 17-19, 1979, University of Texas, Austin, Texas, pp 714-717.

a smoother stress-strain curve. This was impractical with the mechanical sublayer model, so the BSTRS option was coded. If enough points are used, the uniaxial loading curve will be smooth enough. This option is not programmed for a strain rate effect nor for biaxial loading and there is no obvious simple way to include either of them.

The positive portion of the uniaxial stress-strain curve is entered as a table of points,  $(\varepsilon_i, \sigma_i)$ , with FORMAT (2E15.7), which terminates with  $\varepsilon_{NEST} < 0.0$  ( $NEST \leq 100$ ). Then  $\varepsilon_{NEST}$  is replaced by 1000.0 (infinity) and  $\sigma_{NEST}$  is replaced by  $\sigma_{NEST-1}$ . As before, for the mechanical sublayer model, the point  $(\varepsilon_1, \sigma_1)$  is replaced by (SIGZ/E, SIGZ), the elastic limit. Also, as before, these are not engineering stress nor strain. If  $(\varepsilon_{ei}, \sigma_{ei})$  are engineering values,

$$\varepsilon_i = \varepsilon_{ei}(1 + \frac{1}{2}\varepsilon_{ei}), \quad (148)$$

$$\sigma_i = \sigma_{ei}(1 + \sigma_{ei}). \quad (149)$$

These correspond to the mixed components  $\varepsilon_2^2, \sigma_2^2$  of the strain and stress tensors used in RPSL1D. The negative portion of the curve is defined by  $\sigma(-\varepsilon) = -\sigma(\varepsilon)$ .

For each integration station at each mesh point and at each midmesh, we keep track of the strain,  $\varepsilon = \varepsilon_2^2$ , and the mean value of possible elastic strain variation,  $\varepsilon_m$ , in addition to  $\sigma = \sigma_2^2$ .

Initially we set  $\varepsilon = \varepsilon_m = \sigma = 0.0$ .

Assume we have  $\varepsilon$ ,  $\sigma$ , and  $\varepsilon_m$  from time  $t-\Delta t$  (call these  $\bar{\varepsilon}$ ,  $\bar{\sigma}$ , and  $\bar{\varepsilon}_m$ ), and  $\Delta\varepsilon$ , the change in  $\varepsilon$  from time  $t-\Delta t$  to time  $t$ . We recognize five cases for  $\bar{\varepsilon}_m \geq 0$ . (For  $\bar{\varepsilon}_m < 0$ , we change the sign on  $\bar{\varepsilon}$ ,  $\bar{\sigma}$ ,  $\bar{\varepsilon}_m$ , and  $\Delta\varepsilon$ , and proceed as for  $\bar{\varepsilon}_m \geq 0$ . At the end, we change the signs on  $\varepsilon$ ,  $\sigma$ , and  $\varepsilon_m$ .)

In all five cases we set

$$\varepsilon = \bar{\varepsilon} + \Delta\varepsilon. \quad (150)$$

<u>Case</u>	<u>Conditions</u>	<u>Results</u>
(1)	$\Delta\varepsilon \geq 0$	$\sigma = \bar{\sigma} + E \Delta\varepsilon$
	$\varepsilon \leq \bar{\varepsilon}_m + \varepsilon_1$	$\varepsilon_m = \bar{\varepsilon}_m$
(2)	$\Delta\varepsilon < 0$	$\sigma = \bar{\sigma} + E \Delta\varepsilon$
	$\varepsilon > \bar{\varepsilon}_m - \varepsilon_1$	$\varepsilon_m = \bar{\varepsilon}_m$

<u>Case</u>	<u>Conditions</u>	<u>Results</u>
(3)	$\Delta\epsilon > 0$ $\epsilon > \epsilon_m + \epsilon_1$	$\sigma = \sigma(\epsilon)$ $\epsilon_m = \epsilon - \epsilon_1$
(4)	$\Delta\epsilon < 0$ $\epsilon < \epsilon_m - \epsilon_1$ $\epsilon \geq -\epsilon_1$	$\sigma = \sigma(\epsilon + 2\epsilon_1) - 2\sigma_1$ $\epsilon_m = \epsilon + \epsilon_1$
(5)	$\Delta\epsilon < 0$ $\epsilon < \epsilon_m - \epsilon_1$ $\epsilon < -\epsilon_1$	$\sigma = -\sigma(-\epsilon)$ $\epsilon_m = \epsilon + \epsilon_1$

The functional value  $\sigma(\epsilon)$  is found from the table  $(\epsilon_i, \sigma_i)$  by linear interpolation.

(We should mention that the computation of strain energy is as if unloading to  $\sigma = 0$  can take place entirely elastically. That is, the strain energy is assumed proportional to  $(\sigma)^2$  where  $\sigma = E \epsilon$ . This assumption, also used in REPSIL and the catalogued RPSL1D, is only true if  $|\sigma| < 2\sigma_1$ .)

The UPDATE cards for this option include some changes and additions to COMMON in COMDECK MAIN, a few orders to initiate some new arrays to zero in the main program, RPSL1D, a change to read and store the new form for the stress-strain table (input cards 7) in subroutine START, and a complete replacement of BMSTRS, the stress subroutine for beams.

## 6. INPUT

RPSL1D originally used the same input as the standard REPSIL (see Reference 2, pp 54-71). This is still true if the user-coded subroutines are compatible. Table 6.1 lists the input for RPSL1D. The first part of this table is like table 3.1 of Reference 2. Variables that are no longer used have a line drawn through them. A brief description for each variable is given after the table. More complete descriptions for some variables may be found in Reference 2. These descriptions are still valid and will not be repeated here. Units are indicated in brackets: T for time, L for length, and F for force. Limits are those for the catalogued RPSL1D (Appendix C); most of these can be easily increased.

The input for user subroutines is listed in the order the subroutines will be called. Examples of input are given, but the descriptions of the input variables for the user subroutines are purposely sketchy. The user must check these subroutines and their input. INGEOM is always called first and usually requires input from cards. INNORM is called next; no input has been needed so far. PRESS is called next if LOAD  $\neq 0$ . INVEL is called

next if LOAD  $\leq 0$ . (PRESS will be called next if LPRESS  $> 0$ ; so far no version of PRESS has been coded for RPSL1D that required input after the initial entrance. The initial entrance should occur with LOAD  $\neq 0$ , but it could occur here.) ENDFRC will be called next if option APLFRC is used. Subroutine ERODE, part of option EROD, would be called last.

CARD	VARIABLES	FORMAT
1	TITLE	8A10
2	MESH, NMESH, LAYER, YLDFAC	3I5,E12.6
3	MAXC, NCONT, WRITE, DELTAT	3I5,E12.6
4	IBCE1, IBCE2, IBCE3, IBCE4	16I5
5	LOAD, LPRESS, MDAMP, DAMPF, DFACT	3I5,2E12.6
6	E, FNU, SIGZ, RHO, THICKN, NSFL, ISR	5E12.6,2I5
7	(SSIG(J), SEPS(J), DSR(J), PSR(J), J=1,NSFL)	4E15.7
8	NPRINT, (JCHK(I), I=1,3)	16I5
9	NUMCY, (NCYCH(I), I=1,NUMCY)	16I5
10	NLPRIN, (JCYNLP(I), I=1,NLPRIN)	16I5
11	N3D, (NC3DP(I), I=1,N3D)	16I5
11a	NNPE, (NPE(I), I=1,NNPE)	16I5
12	ETAD1, ETAD2, NSTRN	2E10.4,15
13	(ETAG1(I), ETAG2(I), ANGLE(I), ANDLB(I), NETAG(I), I=1,NSTRN)	4E10.4,15
14	Input for subroutine INGEOM	
15	Input for subroutine INNORM	
16	Input for subroutine PRESS	
17	Input for subroutine INVEL	
18	Input for subroutine ENDFRC	
19	Input for subroutine ERODE	

Table 6.1 List of Input Cards

- o IBCE4 from card 4 is stored in position IBCE1 and used as end 1 condition control.
- o Omit card 7 if NSFL = 0 (or if NSFL = 1 and ISR = 0), unless the BSTRS option is used. Card 7 is read with a different form and FORMAT with option BSTRS.
- o Card 11a is included if, and only if, the PLOTP option is used.

The input variables in table 6.1 are described below. Additional descriptions are given in Chapter 3 of Reference 2.

- Card 1 TITLE Alphanumeric statement for output identification.
- Card 2 MESH Not used.
- NMESH Number of meshes. (NMESH < 102)
- LAYER Number of integration stations through the thickness. This may be the number of layers as in REPSIL.

	<b>YLDFACT</b>	Parameter controlling substep size for plastic yielding in shells. (2.0 is suggested)
Card 3	<b>MAXC</b>	Final time step. (Total number of time steps to final time.)
	<b>NCONT</b>	Initial time step for run. This must be zero. Restart capability has been deleted from the CDC version.
	<b>WRITE</b>	Not used.
	<b>DELTAT</b>	Suggested time step. This will be replaced if it is larger than the computed stable $\Delta t$ or if entered as 0.0. [T]
Card 4	<b>IBCE1</b>	Numbers for end conditions. IBCE1 and IBCE3 are unused.
	<b>IBCE2</b>	IBCE2 is for end 2 ( $\xi$ a maximum). IBCE4 is for end 1 ( $\xi$ a minimum).
	<b>IBCE3</b>	IBCE3 is stored in FORTRAN position IBCE1).
	<b>IBCE4</b>	<ul style="list-style-type: none"> <li>1 - clamped end,</li> <li>2 - symmetry end,</li> <li>3 - hinged end,</li> <li>4 - free end, end 2 only with catalogued RPSL1D, forced end, either end with APLFRC option.</li> </ul>
	<b>LOAD</b>	Parameter which controls calls of INVEL and PRESS in the initiation sequence. (If LPRESS > 0, LOAD ≠ 0) <ul style="list-style-type: none"> <li>1 - call PRESS (supplies initial pressure)</li> <li>0 - call INVEL (supplies initial velocity)</li> <li>-1 - call PRESS and then call INVEL</li> </ul>
Card 5	<b>LPRESS</b>	Subroutine PRESS is called each time cycle to supply pressures in P(N), and then P(N) is converted to force/initial unit surface area, until NCYCLE > LPRESS. The force/initial unit area is constant after NCYCLE > LPRESS. (For beams, unit area is replaced by unit length.)
	<b>MDAMP</b>	If NCYCLE ≥ MDAMP, the damping procedures are carried out. If NCYCLE = MDAMP, the program sets LOAD = 0, LPRESS = 0, and all P(N) = 0.
	<b>DAMPF</b>	Viscous damping coefficient. [FT/L <sup>3</sup> ]
	<b>DFACT</b>	Parameter controlling termination of program during damping. (Suggest .001)
	<b>E</b>	Young's modulus. [F/L <sup>2</sup> ]
Card 6	<b>FNU</b>	Poisson's ratio, ν.
	<b>SIGZ</b>	Yield stress, $\sigma_0$ . [F/L <sup>2</sup> ] (See card 7.)
	<b>RHO</b>	Initial mass density per unit volume, ρ. [FT <sup>2</sup> /L <sup>4</sup> , (i.e. mass/L <sup>3</sup> )]
	<b>THICKN</b>	Thickness of shell. [L]
	<b>NSFL</b>	Number of changes of slope in the polygonal approximation of the uniaxial loading curve. (NSFL < 5) <ul style="list-style-type: none"> <li>0 - elastic behavior, no plasticity.</li> <li>1 - elastic perfectly plastic response,</li> <li>&gt;1 - elastoplastic, strain hardening response.</li> </ul>

	<b>ISR</b>	Strain rate sensitivity control. 0 - no strain rate effect, 1 - plasticity is strain rate dependent.
<b>Card 7</b>	<b>SSIG(J), SEPS(J)</b>	Stress, $\sigma$ [ $F/L^2$ ], and strain, $\epsilon$ [ $L/L$ ], at the corners of the polynomial, uniaxial loading curve. (SSIG(1) is replaced by SIGZ and SEPS(1) by SIGZ/E.) (If $(e_s, \sigma_s)$ are 'engineering' strain and stress, $\sigma = \sigma_s(1 + e_s)$ , $\epsilon = e_s(1 + e_s/2)$ . )
	<b>DSR(J), PSR(J)</b>	Constants for strain rate behavior. These must be included if ISR = 1.

With the BSTRS option, cards 7 become:

(EEB(I), SSB(I), I = 1,2,...)      FORMAT(2E15.7)

	<b>EEB(I), SSB(I)</b>	Table of strain and stress (reverse order from above) for up to 100 points. Table is terminated by EEB(NEST) < 0. SSB(1) is replaced by SIGZ, and EEB(1) by SIGZ/E, EEB(NEST) by 1000, and SSB(NEST) by SSB(NEST-1).
<b>Card 8</b>	<b>NPRINT</b>	Number of cycles between surface strain prints at points defined on cards 13. Maximum deflection and extreme strains are also printed.
	<b>JCHK(I)</b>	Numbers controlling printing at cycles on cards 9. JCHK(1), or JCHK(2), > 0 requests prints of coordinates, displacement increments, and pressure at mesh points. JCHK(3) > 0 requests prints of surface normals, and a print of surface strains on both surfaces, at all mesh points.
<b>Card 9</b>	<b>NUMCY</b>	Number of cycles for JCHK(I) controlled prints (card 8) and energy balance summary. (<51)
	<b>NCYCH(I)</b>	Cycles for prints.
<b>Card 10</b>	<b>NLPRIN</b>	Number of JCNLP(I) to follow. (<51)
	<b>JCNLP(I)</b>	Cycles to print IMAT(N,K) and IMATM(N,K) arrays.
<b>Card 11</b>	<b>N3D</b>	Number of cycles at which displacement plots are to be made. (<51)
	<b>NC3DP(I)</b>	Cycles for plots.
<b>Card 11a</b>	(Include if PLOTP option is used.)	
	<b>NNPE</b>	Number of plots to be made. ( $0 \leq NNPE \leq 9$ )
	<b>NPE(I)</b>	Index, N, of P(N) to plot as a function of time. P(N) ( $N1V \leq N \leq N2V$ ) is force per initial unit area (initial length for beams). Other functions of time may be plotted by storing them in P(N) ( $N2B < N < 104$ ) and listing N on this card.

Card 12	<b>ETAD1</b>	Not used.
	<b>ETAD2</b>	Material coordinate of location at which displacement as a function of time is to be plotted.
	<b>NSTRN</b>	Number of locations at which surface strains listed on cards 13 are to be plotted and printed (see card 8). (< 6)
Card 13	<b>ETAG1(I)</b>	Unused.
	<b>ETAG2(I)</b>	Material coordinate for surface strains.
	<b>ANGLE(I), ANGLB(I)</b>	Angles for additional strain calculations. (See Figure 3.5, Reference 2)
	<b>NETAG(I)</b>	Selects surface for strain. 0 - Surface toward positive normal, 1 - Surface away from positive normal.

[Sample Input for User's Subroutines. See USER SUBROUTINE section.]

Card 14 [Subroutine INGEOM]  
 [INGEOM for semi-infinite flat plate or beam. Catalogued INGEOM]  
 (SLABL, SLABW, IB (2E10.5,I5))

SLABL Length.

SLABW Multiplier for energy computation,  $\Delta\xi^1 = 2.0 * SLABW$ .  
 Not used for beams.

IB IB > 0 denotes a beam.  
 If IB > 0, also read  
 (ZU (E10.2))

((WIDTH(K), DZETA(K), ZETA(K), K = 1,...,LAYER) (3E10.2))  
 ZU Location of upper surface.  
 WIDTH(K) Width of K'th layer.  
 DZETA(K) Thickness of K'th layer.  
 ZETA(K) Integration station for K'th layer.

[INGEOM for a right circular cylinder]

**Card 15** [Subroutine INNORM]  
(No input for present INNORM)

**Card 16** [Subroutine PRESS]  
[Constant pressure. Catalogued PRESS]  
(PO (E12.6))  
**PO** Pressure. Force per unit length for beams.

[Time dependent pressure, LINPRS]

((TPR(I), PPR(I), I=1,2,...)	(2E10.3))
((-1.0)	(2E10.3))

[PRESS with tabular data, exponential decay. 3/10/76]

(PBAR, BETA, (PID(I), I=1,6)	(2E10.3,6A10))
(CONCZ, CONCT, CONCP, TDIF	(4E10.3))
(ALPHAS, NOUSE, ISM	(2E10.3,I10))
((ZSI(I), TSI(I), PSI(I), I=1,ISM)	(3E10.3))

Card 17

[Subroutine INVEL]

[Catalogued INVEL. Like INVEL in Reference 2.]

(--, NI, NF, VR, NV	(10X,2I5,E12.6,I5))
(--, N, V	(5X,I5,E12.6))

NI, NF      Mesh points NI+1 through NF+1 are given initial normal velocity -VR. Mesh point at end 1 is numbered N1B = 2.

VR      Velocity. [L/T]

NV      Number of cards to follow.

N      Mesh point N+1 given initial normal velocity -V.

V      Velocity. [L/T]

Card 18

[Subroutine ENDFRC. Needed with APLFRC option.]

[End 2 free. Tabular forces on end 1.]

(NFZPTS, (LABELV(I), I=1,7)	(I10,7A10))
((EFZT(I), TMFZ(I), I=1,NFZPTS)	(2E15.7))
(NFTHS, (LABELV(I), I=1,7)	(I10,7A10))
((EFTHT(I), TMTH(I), I=1,NFTHS)	(2E15.7))
(NFMUS, (LABELV(I), I=1,7)	(I10,7A10))
((EFMUT(I), TMMU(I), I=1,NFMUS)	(2E15.7))

Card 19

[Subroutine ERODE. Part of EROD option.]

(NTEPTS, (LABEL(I), I=1,7)	(I10,7A10))
((TEROD(I), XEND(I), I=1,NTEPTS)	(2E15.7))

## APPENDIX A

### LIST OF EQUATIONS

In this section, we list the principal equations used in the standard REPSIL and the corresponding equations used for axial symmetry, slab symmetry, and beams, respectively. The notation in column 1 is that of Reference 1, except we have assigned an orthonormal cartesian vector basis ( $\hat{i}_1$ ,  $\hat{i}_2$ ,  $\hat{i}_3$ ) and defined corresponding Cartesian components of position,  $Y^1$ ,  $Y^2$ , and  $Y^3$ .

The notation 'same' in any column means that the entry is the same as that in the column to the left.

Standard RFSIL

$$x = \underline{x}(r^1, r^2, r, t)$$

$$= \underline{x}(r^1, r^2, t) + u_0(r^1, r^2, t)$$

Axial Symmetry

$$\underline{x} = \underline{x}(r^1, r^2, r, t)$$

$$= \underline{x}(r^1, r^2, t) + u_0(r^1, r^2, t)$$

Evaluate at  $\theta = 0$ .

$$\underline{x} = \underline{v}^1 \hat{j}_k = \underline{v}^1 \hat{j}_1 + \underline{v}^2 \hat{j}_2 + \underline{v}^3 \hat{j}_3$$

$$\hat{j}_k = \underline{v}^k(r^1, r^2, t)$$

$$\underline{x} = \underline{R}(r^1, r^2, t) + Z(r^1, r^2, t) \hat{k}$$

$$\hat{k} = \cos \theta \hat{j}_3 + \sin \theta \hat{j}_2 (= -d\theta/d0)$$

$$\underline{Z} = -\sin \theta \hat{j}_3 + \cos \theta \hat{j}_2 (= d\theta/d0)$$

$$R = ((\underline{v}^3(r^1, r^2, t))^2 + (\underline{v}^1(r^1, r^2, t))^2)^{1/2}$$

$$= \underline{v}^3(r^1, r^2, t)$$

$$Z = \underline{v}^2(r^1, r^2, t) = \underline{v}^2(0, r, t)$$

$$a_{11} = R_{11} = \underline{v}^2/r^1$$

$$a_{21} = R_{21} = R_{12} \hat{x} + Z_{12} \hat{k}$$

$$\underline{x} = R(r, t) \hat{j}_3 + Z(r, t) \hat{k}$$

$$Z = R(r, t) \hat{j}_3 + Z(r, t) \hat{k}$$

$$\underline{x} = \underline{x}(Y^1, r, t)$$

$$Z = Y^2(0, r, t)$$

$$R = Y^3(0, r, t)$$

$$R = R(r, t) \hat{j}_3 + Z(r, t) \hat{k}$$

$$x = x(r, t)$$

$$z = z(r, t)$$

$$y = y(r, t)$$

$$x = x(Y^1, r, t)$$

$$z = z(Y^1, r, t)$$

$$y = y^2(Y^1, r, t)$$

$$R = R(Y^1, r, t)$$

$$R = R(Y^1, r, t)$$

$$x = x(Y^1, r, t)$$

$$z = z(Y^1, r, t)$$

$$y = y^2(Y^1, r, t)$$

$$R = R(Y^1, r, t)$$

$$x = x(Y^1, r, t)$$

$$z = z(Y^1, r, t)$$

$$y = y^2(Y^1, r, t)$$

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$$R = R(Y^1, r, t)$$

$$x = x(Y^1, r, t)$$

$$z = z(Y^1, r, t)$$

$$y = y^2(Y^1, r, t)$$

$$R = R(Y^1, r, t)$$

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<u>Standard REPSII</u>	<u>Axial Symmetry</u>	<u>Slab Symmetry</u>	<u>Laterally Symmetric Beam</u>
$\underline{u} = (\underline{a}_1 \times \underline{a}_2)/a^2$ ( $a^2 =  \underline{a}_1 \times \underline{a}_2 $ )	$\underline{n} = n_r \hat{x} + n_k \hat{z}$ $n_r = z_{,2}/D$ , $n_k = -R_{,2}/D$	$\underline{n} = n_r \hat{x}_3 + n_k \hat{z}_2$ same	same
$b_{\alpha R} = \underline{n} \cdot \underline{I}_{,\alpha R}$	$b_{11} = -n_r^R$ $b_{22} = n_r^R z_{,22} + n_k z_{,22}$	$b_{11} = 0$ same	same
$b_\alpha^\beta = a^{\beta\gamma} b_{\gamma\alpha}$	$b_1^1 = a^{11} b_{11}$ , $b_2^2 = a^{22} b_{22}$	same $b_1^1 = 0$	same
$\underline{z}^\alpha = a^{\alpha R} \underline{a}_R$	$\underline{z}^1 = a^{11} \underline{R}_0^1$ $\underline{z}^2 = a^{22} R_{,2} \hat{z} + a^{22} z_{,2} \hat{z}$	$\underline{z}^1 = a^{11} \hat{z}_1$ $\underline{z}^2 = a^{22} R_{,2} \hat{z}_3 + a^{22} z_{,2} \hat{z}_2$	same
$\underline{r}_{\alpha R}^Y = \underline{a}^Y \cdot \underline{I}_{,\alpha R}$	$r_{11}^2 = -a^{22} R_{,2}^R$ $r_{22}^2 = a^{22} (R_{,2}^R z_{,22} + z_{,2} z_{,22})$ ( $r_{12}^1 = r_{21}^1 = a^{11} R_{,2}^R$ unused)	$r_{11}^2 = 0$ same ( $r_{12}^1 = r_{21}^1 = 0$ )	same
$\Delta \underline{u} = (\underline{x} - \underline{x}')$ ( $= \Delta u^k \hat{x}_k + \Delta \underline{x}'$ )	$\Delta \underline{x}' = \Delta R \hat{x} + \Delta Z \hat{z}$	$\Delta \underline{x}' = \Delta R \hat{x}_3 + \Delta Z \hat{z}_2$	same
$\Delta n_\alpha = \underline{n}^- \cdot \underline{N}_\alpha$	$\Delta n_2 = -(n_r^- \Delta R_{,2} + n_k^- \Delta Z_{,2})$	same	same
$\Delta \underline{u} = (u_\alpha + \frac{\Delta n_\alpha}{1 + \underline{g}^- \cdot \underline{g}}) \underline{v} + \underline{a}^\alpha \Delta n_\beta$	$\Delta \underline{u} = \Delta n_r \hat{x} + \Delta n_k \hat{z}$ $\Delta n_r = (R_{,2} + n_r^- d n) a^{22} \Delta n_2$ $\Delta n_k = (Z_{,2} + n_k^- d n) a^{22} \Delta n_2$ $d n = \Delta n_Z / (1 + n_r^- n_r + n_k^- n_k)$	$\Delta \underline{u} = \Delta n_r \hat{x} + \Delta n_k \hat{z}$ same same same same	same

Standard REPSIL	Axial Symmetry	Slab Symmetry	Laterally Symmetric Beam
$\delta_{\alpha\beta} = \delta_{\alpha\beta} - 2\epsilon b_{\alpha\beta}$	same $\{\delta_{12} = \delta_{21} = 0\}$	same $\{\delta_{11} = 1\}$	same $\{\delta_{11} \text{ not used}\}$
$R = \delta_{11}\delta_{22} - (\delta_{12})^2$	$R = \delta_{11}\delta_{22}$	same $\{g = \delta_{22}\}$	same
$\delta^{11} = \delta_{22}/\delta + \delta^{22} = R_{11}/\delta$	same $\{\delta^{11} = 1/\delta_{11} + \delta^{22} = 1/\delta_{22}\}$	same $\{\delta^{11} = 1\}$	same $\{g^{11} \text{ not used}\}$
$\delta^{12} = \delta^{21} = -\delta_{12}/\delta$	$\delta^{12} = \delta^{21} = 0$	same	same
$\Delta c_{\alpha\beta} = \frac{1}{2}(a_{\alpha} \cdot \Delta u_{\beta} + a_{\beta} \cdot \Delta u_{\alpha} - \Delta u_{\alpha} \cdot \Delta u_{\beta}) - \epsilon(\Delta u_{\alpha} \cdot \Delta u_{\beta} + \Delta u \cdot \Delta u_{\alpha\beta})$	$\Delta c_{\alpha\beta} = \frac{1}{2}\Delta a_{\alpha\beta} - \epsilon\Delta b_{\alpha\beta} - \Delta c_{12} = \Delta c_{21} = 0$	same $\{\Delta c_{11} = 0\}$	same
	$\frac{1}{2}\Delta a_{11} = \Delta R(R_{,2} - \frac{1}{2}\Delta R_{,2})$	$\Delta a_{11} = 0$	same
	$\frac{1}{2}\Delta a_{22} = \Delta R_{,2}(R_{,2} - \frac{1}{2}\Delta R_{,2}) + \Delta Z_{,2}(Z_{,2} - \frac{1}{2}\Delta R_{,2})$	same	same
	$\Delta b_{11} = -(\Delta R n_{\Gamma}^{-} + R \Delta n_{\Gamma})$	$\Delta b_{11} = 0$	same
	$\Delta b_{22} = (n_{\Gamma}\Delta R_{,22} + n_{\Gamma}^{-}\Delta Z_{,22}) + (\Delta n_{\Gamma} R_{,22} + \Delta n_{\Gamma} Z_{,22})$	same	same
$\Delta r_{\beta}^{\alpha} = g^{\alpha\gamma}\Delta c_{\gamma\beta}$	$\Delta r_{\beta}^1 = g^{11}\Delta c_{11}, \Delta r_{\beta}^2 = g^{22}\Delta c_{22}$	same $\{\Delta r_{\beta}^1 = 0\}$	same
$\sigma_{\beta}^{-\alpha} = g_{\beta\gamma}\sigma^{-\gamma\alpha}$	$\sigma_1^{-1} = g_{11}\sigma^{-11}, \sigma_2^{-2} = g_{22}\sigma^{-22}$	same	same $\{\sigma_1^{-1} \text{ not used}\}$
$\Delta o_{\beta}^{\alpha} = \frac{E}{1-\nu}(\Delta c_{\beta}^{\alpha} + \frac{\nu}{1-\nu}\Delta c_{\gamma}^{\alpha}o_{\beta}^{\alpha})$	$\Delta o_1^1 = (\Delta e_1^1 + \nu\Delta e_2^2)E/(1-\nu)$ $\Delta o_2^2 = (\nu\Delta e_1^1 + \Delta e_2^2)E/(1-\nu)$	same	same $\{\Delta o_1^1 \text{ not used}\}$
$T_{\beta}^{\alpha} = o_{\beta}^{-\alpha} + \Delta o_{\beta}^{\alpha}$	$T_1 = \sigma_1^{-1} + \frac{E_1}{\Delta o_1^1}, T_2 = \sigma_2^{-2} = \frac{E_2}{\Delta o_2^2}$	same	same $\{T_1 \text{ not used}\}$

<u>Standard REPSIL</u>	<u>Axial Symmetry</u>	<u>Slab Symmetry</u>	<u>Laterally Symmetric Beam</u>
$\psi_T = \frac{1}{4} \left( \frac{T_1 T_2}{\rho_a^2} - \left( \frac{\sigma_1}{\rho_a} \right)^2 - \left( \sigma_0 \right)^2 \right)$	$\psi_T = \frac{T_1}{\sigma_1} \frac{T_2}{\sigma_1 - \sigma_2} + \frac{T_2}{\sigma_2}^2 - \left( \sigma_0 \right)^2$	same	$\psi_T = \left( \frac{T_2}{\sigma_2} \right)^2 - \left( \sigma_0 \right)^2$
If $\psi_T < 0$ , $\sigma_1^n = \sigma_2^n$	Same for $\sigma_1^1$ and $\sigma_2^2$	same	If $ \sigma_2  < \sigma_0$ , $\sigma_2^2 = \sigma_2^1$
If $\psi_T > 0$ , the stress increment is partly plastic. Correction is	same	same	If $ \sigma_2  > \sigma_0$ , $\sigma_2^2 = \sigma_0$
$\sigma_1^{-n} = \sigma_B^n - \frac{1-2\nu}{3(1-\nu)} \sigma_1 \delta_\rho^n$	$C_1^{-1} = (2-\nu) \sigma_1^{-1} - (1-2\nu) \sigma_2^{-2}$	same	$T_2 > \sigma_0$ , $\sigma_2^2 = \sigma_0$
where $\lambda$ is the smaller root of	$C_2^{-2} = (2-\nu) \sigma_2^{-2} - \left( \frac{1-2\nu}{1-\nu} \right) \sigma_1^{-1}$	same	$T_2 < -\sigma_0$ , $\sigma_2^2 = -\sigma_0$
$\sigma_B^n = \sigma_\beta^n - \lambda \sigma_\beta^n$	same for $\sigma_1^1$ and $\sigma_2^2$ , where $A\lambda^2 - BA + C = 0$	same	$\sigma_2^2 = g \sigma_2^2$
$3\rho_a^2 \sigma_B^n - \left( \frac{\sigma_1}{\rho_a} \right)^2 = 2(\sigma_0)^2$	$A = \sigma_1^1 (\sigma_1^1 - \sigma_2^2) + (\sigma_2^2)^2$ , $C = \psi_T$	same	
	$B = \sigma_1^1 (\sigma_1^1 - \sigma_2^2) + \sigma_2^2 (2\sigma_2^2 - \sigma_1^1)$	same	
$\sigma^{eff} = g^{inv} \sigma_\gamma^n$	$\sigma^{11} = g^{11} \sigma_1^1$ , $\sigma^{22} = g^{22} \sigma_2^2$	same	
		same	
This stress calculation is for an elastic perfectly plastic material with no complications. The mechanical sublayer model is used for strain hardening. Strain rate dependency may be used with the equations:		same	
$\sigma_0 = \sigma_0(\text{static}) [1 + (\dot{\epsilon}/d)(1/p)]$		same	
$\dot{\epsilon} = \frac{1}{\Delta t} \left[ \frac{3}{2} \Delta \epsilon_B^{\alpha} \Delta \epsilon_B^{\beta} - \frac{1}{2} (\Delta \epsilon_\gamma^1)^2 \right]^{\frac{1}{2}}$	$\dot{\epsilon} = [\Delta \epsilon_1^1 (\Delta \epsilon_1^1 - \Delta \epsilon_2^2) + (\Delta \epsilon_2^2)^2]^{\frac{1}{2}} / \Delta t$	same	
If $\psi_T$ is large, the above procedure is repeated L times with $\Delta \epsilon_B^{\alpha}/L$ in place of $\Delta \epsilon_B^{\alpha}$ . If $\lambda$ is negative or complex, L is increased. If L is too large, or if $\dot{\epsilon} < 0$ , the run is aborted.		same	No iteration.

<u>Standard REPSIL</u>	<u>Axial Symmetry</u>	<u>Slab Symmetry</u>	<u>Laterally Symmetric Beam</u>
" For each pseudo-sublayer is stored for future use. A weighted stress, $\sigma_k^{RF}$ is found for layer k.	same for $\sigma^{11}$ and $\sigma^{22}$	same	same for $\sigma^{22}$
$\Sigma_a \sigma_k^{RF} = \Sigma_k \sigma_k^{RF} (\tau_k)^a$ $\forall \gamma$ , (a=0,1,2)	same for $\tau_a^{11}$ and $\tau_a^{22}$	same	$\Sigma_a W_k \sigma_k^{22} (\tau_k)^a$ ( $W_k$ = Area_k)
$\hat{Q}^* \sigma_k^{RF} = a \left[ \Sigma_0 \sigma_k^{RF} - b_\gamma^\gamma \Sigma_1 \sigma_k^{RF} \right]$	$\hat{Q}^*^{11} = a^{\frac{1}{2}} [\Sigma_0^{11} - (b_1^1 + b_2^2) \Sigma_1^{11}]$	same $\{b_1^1 = 0\}$	$Q^*^{11} = 0$ not used
	$\hat{Q}^*^{22} = a^{\frac{1}{2}} [\Sigma_0^{22} - (b_1^1 + b_2^2) \Sigma_1^{22}]$	same	same
$\hat{M}^* \sigma_k^{RF} = a \left[ \Sigma_1 \sigma_k^{RF} - b_\gamma^\gamma \Sigma_2 \sigma_k^{RF} - b_\gamma^{(a} \Sigma_2^{\gamma b)} \right]$	$\hat{M}^*^{11} = a^{\frac{1}{2}} [\Sigma_1^{11} - (2b_1^1 + b_2^2) \Sigma_2^{11}]$	same	$M^*^{11} = 0$ not used
	$\hat{M}^*^{22} = a^{\frac{1}{2}} [\Sigma_1^{22} - (b_1^1 + 2b_2^2) \Sigma_2^{22}]$	same	same
$\hat{N}^* \sigma_k^{RF} = \hat{Q}^* \sigma_k^{RF} \hat{\alpha}_\beta + \Gamma_{\beta\gamma}^\alpha M^* \hat{\gamma}_\gamma$	$\hat{N}^* 1 = F N_0^1 \hat{\alpha}_2 = Q^* 11 \hat{\alpha}_2$	$\hat{N}^* 1 = \hat{Q}^* 11 \hat{\alpha}_1$ not used	same $\{\hat{N}^* 1 = 0\}$
	$\hat{N}^* 2 = F N_T^2 \hat{\alpha}_2 + F N_K^2 \hat{\alpha}_2$	$\hat{N}^* 2 = F N_T^2 \hat{\alpha}_3 + F N_K^2 \hat{\alpha}_2$	same
	$F N_T^2 = \hat{Q}^* 22 \hat{\alpha}_{R,2} + n_{R,CSM}^2$	same	same
	$F N_K^2 = \hat{Q}^* 22 \hat{\alpha}_{Z,2} + n_{Z,CSM}^2$	same	same
	$CSM^2 = \Gamma_{11}^2 \hat{M}^* 11 + \Gamma_{22}^2 \hat{M}^* 22$	same $\{\Gamma_{11}^2 = 0\}$	same
$\hat{M}^* \sigma_k^{RF} = \hat{M}^* \sigma_k^{RF} \hat{\eta}$	$\hat{M}^* \alpha\alpha = F M_T^{\alpha\alpha} \hat{\alpha}_3 + F M_K^{\alpha\alpha} \hat{\alpha}_2$ (no sum)	$\hat{M}^* 11 = F M_T^{11} \hat{\alpha}_3 + F M_K^{11} \hat{\alpha}_2$ (no sum)	same $\{\hat{M}^* 11 = 0\}$
	$F M_T^{11} = \hat{M}^* 11 n_T$	same	same
	$F M_K^{22} = \hat{M}^* 22 n_K$ ( $F M_K^{11}$ not used)	same	same

Standard REPSIL	Axial Symmetry	Slab Symmetry	Laterally Symmetric Beam
$\dot{F}_T^* = -a \dot{\gamma}_k p \cdot \dot{n}$	$\dot{F}_T^* = F_T^* \dot{\gamma}_T + F_k^* \dot{\gamma}_k$ $F_T^* = -a \dot{\gamma}_T \Delta p n_T, \quad F_k^* = -a \dot{\gamma}_k \Delta p n_k$	$\dot{F}_T^* = F_T^* \dot{\gamma}_3 = F_k^* \dot{\gamma}_2$ same	same
$\dot{M}_T^+ = M_T + \lambda t^2 (\dot{y}^{*\alpha} \beta_{\alpha\beta} + \dot{y}^{*\beta} \alpha_{\alpha\beta})$ $= \lambda y^k \dot{\gamma}_k$	$\dot{M}_T^+ = \Delta R + \lambda t^2 (F_M_T^{22} - F_M_T^{11} + F_N_T^{22} - F_N_T^{11})$ $+ F_T^*)/r_0^*$	$\Delta R^+ = \Delta R + \lambda t^2 (F_M_T^{22} - F_N_T^{22} + F_N_T^{22} - F_M_T^{22})$ $+ F_T^*)/r_0^*$	same
$\dot{M}_k^+ = M_k + \lambda y^k \dot{\gamma}_k$	$\Delta Z^+ = \Delta Z + \lambda t^2 (F_M_k^{22} - F_N_k^{22} + F_N_k^{22} - F_M_k^{22})$ $+ F_k^*)/r_0^*$		same
$y_0^* = \rho h A \dot{x} \{A = a \text{ at time } t=0\}$	$y_0^* = \rho A \frac{\dot{z}}{k} \Delta N_k$		
$\dot{x}^* = \dot{x} + \lambda \dot{y}_T^+$			same
$y^k+ = y^k + \lambda y^k$	$R^+ = R + \Delta R^+, \quad Z^+ = Z + \Delta Z^+$		same

## APPENDIX B

### DEFINITION OF FORTRAN VARIABLES

In this appendix we list the FORTRAN variables used in RPSL1D as listed in Appendix C, plus additional variables used in the options PLOTP, SHR3/1, MSQSVS, APLFRC, EAPFRC, EROD, or BSTRS. Extra columns and a lot of symbology were used in an attempt to shorten the appendix but still give enough information.

Column 1 is the FORTRAN name. Closely related names may be grouped. Arrays are indicated by enclosing a symbolic integer in parentheses (e.g. AMS(N)).

Column 2 lists the symbol used in this text or one of the references. If the symbol is in braces, { }, it was first used in this report; if in brackets, [ ], it was first used in Reference 2; if in neither braces or brackets, the symbol originated in Reference 1. Examples:

{M} Symbol for moment in this report.

[A] Coefficient used in Reference 2.

a<sub>11</sub> Symbol of a covariant metric component first used in Reference 1.

Column 3 partially describes the status of the variable:

C denotes a name in COMMON.

I denotes input.

S<sub>0</sub> denotes a variable that is rarely changed throughout a computer run, as opposed to

S denoting a term that is fixed for one time step, or

T denoting a transient value.

D denotes a dummy argument for a subroutine.

Column 4 names the subroutine which stores the variable. (If other subroutines change or store these variables, they will usually be listed in Column 6.)

Column 5 gives the stage of development when the variable originated:

U denotes a variable described in Appendix C of the User's Manual (Reference 2),

R denotes a newer name included in the RPSL1D file,

any name is the option that first used the variable.

Column 6 defines the variable and may give additional information such as a reference, a limiting value, a formula, the input card that it is entered on, or any other item of interest.

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
<b>A</b>	[A]	T	STRESS	U	Coefficient for computing TAMBDA.
<b>ANS(:,:)</b>	(N)	C,S	RESULT	SIR3/1	Moment at mesh point N. ANS = A22*F22.
<b>ALIN</b>		T	STRAIN	U	Alphanumeric for output: "INNER" or "OUTER".
<b>ANGIL</b>	[θ]	T	STRAIN	U	ANGLE(I) or ANGLB(I) in radians.
<b>ANGLR(I), ANGLF(I)</b>	[θ]	I,C,S <sub>0</sub>	START	U	Angles to locate surface strains. (Input 13 in degrees) (Ref. 2, p. 52)
<b>AR11, AR22</b>	a <sup>11</sup> ,a <sup>22</sup>	T	DGEOM	U	Contravariant components, reference surface metric.
<b>ASA(I), ASB(I)</b>	[2σ <sup>2</sup> ]	C,S	STRAIN	U	(σ = sinθ. Ref. 2, p. 52)
<b>AV3IN</b>	u · u	T	STRAIN	R	Square of displacement.
<b>A11,A22</b>	a <sub>11</sub> ,a <sub>22</sub>	C,T	DGEOM	U	Covariant components, reference surface metric.
<b>A11f(I),A22f(I)</b>	A <sub>11</sub> ,A <sub>22</sub>	C,S <sub>0</sub>	ABINIT	U	Initial a <sub>11</sub> ,a <sub>22</sub> at NI1(I).
<b>A112(I),A222(I)</b>	A <sub>11</sub> ,A <sub>22</sub>	C,S <sub>0</sub>	ABINIT	U	Initial a <sub>11</sub> ,a <sub>22</sub> at NI2(I).
<b>B</b>	[-2B]	T	STRESS	U	Coefficient for computing TAMBDA.
<b>BEPI,BEP2,BEPS</b>		T	BSTRS		Temporary.
<b>B411,B422</b>	b <sub>1</sub> <sup>1</sup> ,b <sub>2</sub> <sup>2</sup>	C,T	DGEOM	U	Mixed components of 2nd fundamental tensor.
<b>BSA(I),BSB(I)</b>	[2B <sup>2</sup> ]	C,S <sub>0</sub>	STRAIN	U	(β = cosθ. Ref. 2, p. 52)
<b>BT</b>	b <sub>1</sub> <sup>1</sup> *b <sub>2</sub> <sup>2</sup>	C,T	DGEOM	U	Trace of 2nd fundamental tensor.
<b>B11,B22</b>	b <sub>11</sub> ,b <sub>22</sub>	C,T	DGEOM	U	Covariant components, 2nd fundamental tensor.
<b>B111(I),B221(I)</b>	B <sub>11</sub> ,B <sub>22</sub>	C,S <sub>0</sub>	ABINIT	U	Initial b <sub>11</sub> ,b <sub>22</sub> at NI1(I).
<b>B112(I),B222(I)</b>	B <sub>11</sub> ,B <sub>22</sub>	C,S <sub>0</sub>	ABINIT	U	Initial b <sub>11</sub> ,b <sub>22</sub> at NI2(I).
<b>C</b>	[C] {C}	T	START	U	Sound speed. C=√(ρ(1-v <sup>2</sup> )). If IB>0, C=√E/ρ.
<b>CA</b>	λε <sup>1</sup> * λε <sup>2</sup>	C,S <sub>0</sub>	START	U	Surface area. Used for kinetic energy and work.
<b>CB</b>	C,S <sub>0</sub>	START	R	CB = $\frac{1}{4} \lambda \epsilon \Delta \epsilon / \Delta \epsilon^2 / E$ . If IB>0, CB = $\frac{1}{4} \lambda \epsilon / \Delta \epsilon^2 / E$ . If IGauss = 1, CB = $\frac{1}{4} \lambda \epsilon \Delta \epsilon / \Delta \epsilon^2 / E$ .	

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
CINEP	[T*]	C,S	DAMP	U	Kinetic energy, KE, removed by KIA. (Ref. 2, p.46)
CINER	[T]	C,S	KINET	U	KE at time t-Δt/2.
CV,N	[T,T,1]	C,S	KINET	U	KE at time t+Δt/2 or t+2Δt/2. (REFSLD,DAMP,DESTP)
CJ,M...	[1,1]	C,T	DAMP	U	Previous CINER in DAMP and DESTP.
CINES2	[T]	C,S	DAMP	R	Previous CINES in DAMP. (Unused)
CINET		C,S	KINET	U	Kinetic energy at current time, t. (DESTP)
Ch	T	KINET	U		Portion of CA uscd with mesh point. (TWORK)
CS1?		T	RESULT	U	Normal component of N* <sup>2</sup> /TB.
CS21!,CS22?	r <sub>11</sub> <sup>2</sup> ,r <sub>22</sub> <sup>2</sup>	C,T	DGEOM	U	Christoffel symbols. Used in RESULT.
CTWO	{C <sub>2</sub> }	T	START	R	Constant for computing DFLM. (See 3.5)
C1	[C2/(C2+4)]	C,S <sub>0</sub>	START	U	Constant for equation of motion. (DESTP)
C10ff	T	DESTP	U		Old value of C1 in DESTP.
C2	(a <sub>22</sub> ) <sup>-1</sup>	C,S <sub>0</sub>	START	U	(C2 = 2 * DELTAT * DAMPF / GAMZ.) (DESTP)
D	[B <sup>2</sup> -AC]	T	DGEOM	R	Temporary storage in DGEOM, MOTION, and INNORM.
D	a	T	STRESS	U	Discriminant of quadratic equation for TAMBDA.
DA		T	DGEOM	U	Determinant of surface metric. DA = A11*A22.
DAMPF	[W]	C,I,S <sub>0</sub>	START	U	Viscous damping coefficient. (Input 5)
DAT(I)	T	PDATA	U		Array for storage on plotting file.
DA1(N)	b <sub>1</sub> (a <sub>11</sub> -A <sub>11</sub> )	C,S	DGEOM	R	Σ DA11 at mesh point N. (All at time 0.)
DA2(N)	b <sub>2</sub> (a <sub>22</sub> -A <sub>22</sub> )	C,S	DGEOM	R	Σ DA22 at mesh point N. (A22 at time 0.)
DA11,DA22	b <sub>1</sub> a <sub>11</sub> ,b <sub>2</sub> a <sub>22</sub>	T	DGEOM	U	Half the increment in a <sub>11</sub> ,a <sub>22</sub> from t-Δt to t.
DA22P	b <sub>2</sub> a <sub>22</sub>	T	DGEOM	R	b <sub>2</sub> a <sub>22</sub> at previous midmesh.
DB1(N)	b <sub>11</sub> -b <sub>11</sub>	C,S	DGEOM	R	Σ DB11 at mesh point N.
DB2(N)	b <sub>22</sub> -b <sub>22</sub>	C,S	DGEOM	R	Σ DB22 at mesh point N.
DB11,DB22	Δb <sub>11</sub> ,Δb <sub>22</sub>	T	DGEOM	U	Increment in b <sub>11</sub> ,b <sub>22</sub> from t-Δt to t.
DE		C,S	ERODF	EROD	Erosion from end mesh point. May be negative.

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
DECOM	{Δε/4}	S <sub>0</sub>	ERODE	EROD	Maximum permitted DE before changing end point.
DEFX		C,S	STRAIN	R	Maximum displacement.
DEFXT		T	STRAIN	R	Maximum displacement in current cycle.
DELR	[Δt <sub>B</sub> ]	T	START	U	Critical bending time increment. $\Delta t_B = \frac{1}{2}(\Delta tA)^2 / (C/CTWO)$ $\Delta t_B = \sqrt{3}(\Delta tA)^2 / (Ch)$ , if IGAUSS = 1.
DEFME		T	ERODE	R	Erosion increment from t-Δt to t.
DEFGM	(Δt) <sup>2</sup> /γ <sub>0</sub> ) (Δt) <sup>2</sup> /GMZ	C,S <sub>0</sub>	RPSLID	U	Constant used in DGFM for TEMP(N).
DELIN	Δt	T,I	START	U	Temporary storage for input Δt.
DEFIN	[Δt <sub>M</sub> ]	T	START	U	Critical membrane time increment. $\Delta t_M = \Delta t/C$ .
DEFMIN		T	START	U	0.95*MIN[DELB,DEIM] rounded.
DELR,DELS		T	DESTEP	U	Factors used in DESTEP to change DELTAT.
DELSQ	(Δt) <sup>2</sup>	C,S <sub>0</sub>	START	U	Square of time increment. (DESTEP)
DELTAT	Δt	C,I,S <sub>0</sub>	START	U	Time increment. (Input 13) May change to minimum of DELIN and DEIMIN. (DESTEP)
DELTTR,DELTZ		T	BOUNDR	R	Components used to locate external point.
DEPS1(K), DEPS2(K)	Δε <sub>11</sub> ,Δε <sub>22</sub>	C	DGEOM	U	Covariant components of strain increment at ξ <sub>k</sub> . $\Delta\epsilon_{ab} = \epsilon_{ka}\epsilon_{ab} - \epsilon_{kb}\epsilon_{ab}$ .
DEPS11,DEPS22	Δε <sub>11</sub> <sup>1</sup> ,Δε <sub>22</sub> <sup>2</sup>	T	STRESS	U	Mixed components of strain increment. (BMSTRS)
DETA1	Δε <sup>1</sup> [Δn <sup>1</sup> ] Δε <sup>2</sup> [Δn <sup>2</sup> ] {Δξ}	C,S <sub>0</sub>	INGEOM	U	Increment in the ξ <sup>1</sup> ,ξ <sup>2</sup> Lagrange coordinates. (If IB=0, Δε <sup>1</sup> =1.0 in START.)
DFACT		C,I,S <sub>0</sub>	START	U	Factor for terminating damped run. (Input 5)
DFIG	S <sub>0</sub>		PRESS		In PRESS of 10/12/76, DFPC = -DAMPF/GMZ.
DG	g	T	STRESS	U	Determinant of metric. DG = G11*G22.
DN	Δn	T	DGEOM	U	Normal component of Δn. $\Delta n = \Delta n_1 \xi^1 + \Delta n_2 \xi^2 + \Delta n$ .

NAME	SYMBOL	STATUS	STORED	ORIGIN	COMMENTS
DN1(1), DN2(1), DN3(1)	C,S <sub>0</sub> C,S <sub>0</sub>	START START	U R	U R	Interpolation coefficients for surface strains. DN3(1) added with revision.
DNK, DNR	{Δn <sub>k</sub> , Δn <sub>r</sub> }	T	DGEOM	R	Cartesian components of Δn. Δn = DNK i <sub>2</sub> + DNR i <sub>3</sub> .
DNL2, DNR2	Δn <sub>2</sub> , Δn <sub>2</sub> <sup>2</sup>	T	DGEOM	U	Covariant, contravariant tangential component of Δn.
DR(N), DZ(N)	{R, Z} {ΔR, ΔZ}	C,S	INVEL	R	[Replaces Uj(M,N)] Components of velocity (INVEL). Displacement increments RPSLD, MOTION, and other.
DRI, DZI	{ΔR <sup>-</sup> , ΔZ <sup>-</sup> }	T	MOTION	R	Components of displacement increment, t-Δt to t.
DRI1, DZI1	{ΔR <sup>-</sup> , ΔZ <sup>-</sup> }	T	MOTION	EAPFRC	DRI and DZI at end 1.
DRMSN	{ΔR}	T	DGEOM	R	Radial displacement increment.
DRS, DZS	{ΔR, ΔZ}	T	MOTION	R	Components of displacement increment, t to t+Δt.
DR2, DZ2	{ΔR, <sub>2</sub> } {ΔZ, <sub>2</sub> }	C,T C,T	DGEOM	R	[Replace U2i.] First finite differences of displacement increment components.
DR22, DZ22	{ΔR, <sub>22</sub> } {ΔZ, <sub>22</sub> }	C,T C,T	DGEOM	R	[Replace U22i.] Second finite differences of displacement increment components.
DSG11, DSG22L	E <sup>a</sup> <sub>β</sub> /L	T	STRESS	U	Subincrement division of E <sup>a</sup> <sub>β</sub> .
DSG11, DSG22	ΔE <sub>β</sub> <sub>1</sub> , ΔE <sub>β</sub> <sub>1</sub>	T	STRESS	U	Mixed components of elastic stress increment.
DSQOLD		T	DESTEP	U	Old (Δt) <sup>2</sup> .
DSR(j)	{d <sub>j</sub> }	C,I,S <sub>0</sub>	START	U	Constant for strain rate (Input 7).
DTHT	{Δθ}	T	MOTION	EAPFRC	Angle increment [t, t+Δt] current end.
DM1P, DM2P	{Δθ <sub>j</sub> <sup>-</sup> }	C,S <sub>0</sub>	MOTION	EAPFRC	Angle increment [t-Δt, t] (end 1, end 2).
DM1P, DM2P	{ΔW <sub>j</sub> <sup>-</sup> }	C,S <sub>0</sub>	MOTION	EAPFRC	Work increment [t-Δt, t] (end 1, end 2).
DMF	{ΔM <sub>F</sub> }	T	MOTION	EAPFRC	Work increment [t, t+Δt], from force on the end.
DMF1	{ΔM <sub>M</sub> }	T	MOTION	EAPFRC	Work increment [t, t+Δt], from moment on the end.

<u>VAR.</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
DW	$\eta \cdot \Delta x$	T	PWORK	U	Normal component of $\Delta x$ .
D <sup>-</sup> (N)	$f(\Delta z)$	C,S	MOTION	R	See DR(N).
DZETAK	$(\Delta x_k)$	I,T	INGEON	R	Thickness of k'th layer of a beam. (Input 14)
BZI, BZII, BZS		T	MOTION	R	See DRI, DRII, DRSS.
DZ2,DZ22		C,T	DGEOM	R	See DR2 and DR22.
D1,D2	$ u^3, u^2 $	C,S	STRAIN	U	Components of displacement at ETAN2. { $\Delta R, \Delta Z$ }
E	E	C,I, $S_0$	START	U	Young's modulus. (Input 6)
HER(1),SSB(1)	$(\epsilon_i, \sigma_i)$	C,I, $S_0$	STA <i>n</i> ,	BSTRS	Uniaxial stress-strain curve. 1<1<100. (Input 7) (See comments at SEPS(J).)
EPM1	{u(t)}	T	ENDFRC	APLFRC	$\nu$ at time t. EPM1 = EPMU * EFZ1.
EPMUT(1),TMMU(1)	{v <sub>i</sub> ,t <sub>i</sub> }	I, $S_0$	ENDFRC	APLFRC	Table of $\mu$ vs. time. (Input 18)
EFR1		C,S	ENDFRC	APLFRC	$\lambda_3$ force, end 1. EFR1>0 decreases $\Delta R(N1B)$ .
EFR2		C,S	ENDFRC	APLFRC	$\lambda_3$ force, end 2. EFR2>0 increases $\Delta R(N2B)$ .
EFTM1	{θ(t)}	T	ENDFRC	APLFRC	θ at time t.
EFTMUT(1),TMU(1)	{θ <sub>i</sub> ,t <sub>i</sub> }	I, $S_0$	ENDFRC	APLFRC	Table of θ vs. time. (Input 18)
EFZT(1),TMFZ(1)		I, $S_0$	ENDFRC	APLFRC	Table of force, EFZ1, vs. time, t. (Input 18)
EFZ1		C,S	ENDFRC	APLFRC	$\lambda_2$ force, end 1. EFZ1>0 decreases $\Delta Z(N1B)$ .
EFZ2		C,S	ENDFRC	APLFRC	$\lambda_2$ force, end 1. EFZ2>0 increases $\Delta Z(N2B)$ .
FMI		C,S	ENDFRC	APLFRC	Applied moment at end 1. EM1>0 decreases $\Delta R(N1B)$ , increases $\Delta R(N1B+1)$ .
EM2		C,S	ENDFRC	APLFRC	Applied moment at end 2. EM2>0 increases $\Delta R(N2B)$ , decreases $\Delta R(N2B-1)$ .
EN	[ΔM(t-Δt)]	T	MOTION	U	Work increment, [t-Δt,t]. (EN = ½CA*(EMR+ENS))
ENR	[ΔW(t-Δt)/CA]	C,S	MOTION	U	Work increment/CA, [t-½Δt,t-½Δt]. (RPSL10)

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
ENS	[ $\Delta N(t)/CA$ ]	C,S	PWORK	U	Work increment/CA, [ $t - \frac{1}{2}\Delta t, t + \frac{1}{2}\Delta t$ ]. (RPSLID)
FPC(1)		T	STRAIN	R	Temporary for computing extreme surface strains.
EPCL1, EPCL2	[ $E_1, E_2$ ]	T	DGEOM	R	Elongational components of strain on the lower surface in $\xi_1$ and $\xi_2$ directions. (Output)
EPCL11, EPCL12	[ $E_1, E_2$ ]	T	DGEOM	R	Elongational components of strain on the upper surface in $\xi_1$ and $\xi_2$ directions. (Output)
EPLB(N,K)	{ $\epsilon_m$ }	C,S	BMSTRS	BSTRS	Mean of possible elastic variation of EPSB(N,K).
EPLBm(N,K)	{ $\epsilon_m$ }	C,S	BMSTRS	BSTRS	Mean of possible elastic variation of EPSBM(N,K).
EPSANR(I)	[ $E_0$ ]	T	STRAIN	U	Elongational surface strains in $\theta$ directions at location I. (Output)
EPSANG(I)	[ $E_0$ ]	T	STRAIN	U	Elongational surface strains in $\theta$ directions at location I. (Output)
EPSB(N,K)	$\epsilon_2^2$	C,S	BMSTRS	BSTRS	Uniaxial strain at mesh point N, layer K.
EPSRN(N,K)	$\epsilon_2^2$	C,S	BMSTRS	BSTRS	Uniaxial strain at midmesh N, layer K.
EPSROT	$\dot{\epsilon}$	T	STRESS	U	Deviator strain rate in strain rate equation.
EPSSL1(N), EPSSL2(N)	$\epsilon_{11}, \epsilon_{22}$	C	DGEOM	U	Covariant strain components on the lower surface at mesh point N.
EPSR1, EPSR2	[ $\epsilon_1, \epsilon_2$ ]	T	STRAIN	U	Intermediate values in surface strain calculations.
EPSI1(I), EPSI2(I)	[ $E_1, E_2$ ]	C	STRAIN	U	Elongational components of strain in the $\xi^1$ and $\xi^2$ directions at position I.
EPSU1(N), EPSU2(N)	$\epsilon_{11}, \epsilon_{22}$	C	DGEOM	U	Covariant strain components on the upper surface at mesh point N.
EPSX(I)	C,S	STRAIN	R	Extreme strains. (I = 1,8) (Output)	
EPSSXT(I)		T	STRAIN	R	Extreme strains at current time. (Output)
EPSZ	$\sigma_0/E$	C, $S_0$	START	BSTRS	Strain at yield stress.
EROD		T	ERODE	EROD	Erosion from initial end 1 up to time t.
ERODP	S	ERODE	EROD	EROD	Erosion from initial end 1 up to time $t - \Delta t$ .

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
ETADZ	$\xi^2 \{ \xi \} [n^2]$	C,I,S <sub>0</sub>	START	U	Material coordinate of the point at which displacement is plotted. (Input 12) (INGEOM?)
ETA <sub>12</sub> (I)	$\xi^2 \{ \xi \} [n^2]$	C,I,S <sub>0</sub>	START	U	Material coordinate for the I'th surface strain. (Input 13) (DGFOM?)
FLOATL	[1/L]	T	STRESS	U	Proportion of stress increment per step.
FN1IR(N)	{M*}J1 {n <sub>r</sub> }	C,T	RESULT	R	Components of bending resultant tensor times the normal at mesh point N. (These replace FN <sub>uB</sub> (M,N) of Reference 2.)
FN2RK(N)	{N*}J2 {n <sub>k</sub> }				
FN2R(N)	{N*}J2 {n <sub>r</sub> }				
TNU	C,I,S <sub>0</sub>	START	U		Poisson's ratio. (Input 6)
FN1IR(N),FN2K(N)	v	C,T	RESULT	R	Components of stress resultant tensor at midmesh N. [These replace FN <sub>aj</sub> of Reference 2.]
FN2R(N)					
F11,r <sup>2</sup> ,	$\hat{M}^{11}_{IR}, \frac{\hat{N}^{22}}{T_B}$	T	RESULT	U	Proportional to bending resultant components.
G	$\frac{M}{v}/(1+v)$	C,S <sub>0</sub>	START	U	Shear modulus. Used in STRESS.
GAE	$\gamma_0^* [r_0]$	C,S <sub>0</sub>	START	U	If IB=0, $r_0 = ph$ = mass/unit surface area.
					If IB=1, $r_0 = \rho * SUMAR = mass/unit length.$
G111(I),G122(I)	$\hat{G}^{11}, \hat{G}^{22}$	C,S <sub>0</sub>	STRAIN	U	Contravariant components of initial metric at I.
G111(N),G122(N)	$\hat{G}_{11}, \hat{G}_{22}$	C,S <sub>0</sub>	RPSLID	R	Initial metric, lower surface, mesh point N.
GR11,GR22	$\hat{g}^{11}, \hat{g}^{22}$	T	STRESS	U	Contravariant metric components. (BMSTRS)
G111(N),G122(N)	$\hat{G}_{11}, \hat{G}_{22}$	C,S <sub>0</sub>	RPSLID	R	Initial metric, upper surface, mesh point N.
G11,G12	$\hat{g}_{11}, \hat{g}_{22}$	T	STRESS	U	Metric at $\zeta_k$ in STRESS and BMSTRS.
	$\hat{G}_{11}, \hat{G}_{22}$	T	STRAIN	U	Initial metric on surfaces in STRAIN.
I	[i,j]	T		U	General index. Multiple use.
IB	C,S <sub>0</sub>	INGEOM	R		Signal from INGEOM: 0 or 1. IB=1 denotes a beam.

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
ICCE1	C,I,S <sub>0</sub>	START	R		Boundary condition end 1. Edge 4 of REPSIL.
ICCE2	C,I,S <sub>0</sub>	START	U		Boundary condition end 2. Edge 2 of REPSIL.
INPA4	(T)	(NONE)	U		REPSIL name. Read into ICCE1. (See input 4)
IENDFR	S <sub>0</sub>	ENDFRC	APLFRC		Signal for initial entry into ENDFRC.
IFRODE	S <sub>0</sub>	ERODE	EROD		Signal for initial entry into ERODE.
IFLAG	T	FDATA	U		Control number in subroutine FDATA.
IGAUSS	C,S <sub>0</sub>	INGEOM	R		Signal from INGEOM. IGAUSS=1 denotes Gaussian integration through thickness. (If 1B=1, IGAUSS=0 in START.)
II	S <sub>0</sub>	PDATA	U		Dimension of DAR(1) array. II = 2*NSTRN+8.
IP	C	DGFOM	R		IP=1 denotes mesh point, IP=2 denotes midmesh.
IPRESS	S <sub>0</sub>	PRESS	R		Signal for initial entry into PRESS.
IS	C,S <sub>0</sub>	START	R		(IS=1 or 2) 1-radial symmetry, 2-slab symmetry.
ISR	C,I,S <sub>0</sub>	START	U		Strain rate sensitivity control. (Input 6)
J	[J]	T	U		General index. Occasionally mechanical sublayer.
JCHK(J)	C,I,S <sub>0</sub>	START	U		Print controls. (Input 8)
JCNLP(I)	C,I,S <sub>0</sub>	START	U		Time cycles to print LMAT and LMATM. (Input 10)
J1,J2	T	STRAIN	U		Temporary indices equal to NI1(1) and NI2(1).
J3	T	STRAIN	R		Temporary index equal to NI3(1).
K	[k]	T	U		General index. Frequently integration stations.
MP	[k]	D	DGEOM	U	Dummy argument for station K in STRESS and BMTRS.
KEY	D	(none)	U		Dummy argument in MRTAPE. MRTAPE is not called.
KJ	T	STRESS	U		Index. J'th mechanical sublayer at K'th station.
KJMAX	C,S <sub>0</sub>	START	U		Maximum KJ. KJMAX = NSFL*LAYER.
KN	T	STRESS	U		KN = NSFL*(K-1). So, KJ = KN + J. (BMTRS)

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
L	[l.]	T	STRESS BMSTRS	U R	Number of subincrements for stress increment. L=0, or L=I + INT(4.0*( SIGGZ /SIGGZ - 1.)) General index.
L		T		R	
LAYER(I)	I,S <sub>0</sub>		ENDFRC	APLFRC	Label for force table. (Input 18)
LABEL(I)	I,S <sub>0</sub>	ERODE	EROD		Label for XEND vs. TEROD table. (Input 19)
LAYER	(l.) [K]	C,1,S <sub>0</sub>	START	U	Number of integration stations through thickness. (Input 2)
LC		T	STRESS	U	Counter in STRESS. 1 < LC < L+1.
LEPSX(I)	S <sub>0</sub>		STRAIN	R	Label for EPSX(I) in output. (Output)
LINK	T,D		STRAIN	U	Control switch in STRAIN and PDATA.
LMAT(N,K)	C		STRESS	U	Matrix of LMAT at mesh points. (BMSTRS) (Output)
LMATH(N,K)	C		STRESS	R	Matrix of LMAT at midmeshes. (BMSTRS) (Output)
LINK	T		STRESS	U	Maximum l., station K, mesh or midmesh N. (BMSTRS)
LOAD	C,I,S <sub>0</sub>	START	U		Mode of loading. (Input 5) (DAMP)
LPRESS	C,I,S <sub>0</sub>	START	U		Last time step to call PRESS in RPSLID and modify P(N) in DGEOM. (Input 5) (DAMP)
M	T	MOTION	APLFRC		Temporary index. [Not M of REPSIL in Ref. 2.]
MAXC	C,I,S <sub>0</sub>	START	U		Final cycle. (Input 3) (DAMP)
MDAMP	C,I,S <sub>0</sub>	START	U		Time cycle at which damping begins. (Input 5)
MRITE	C,S <sub>0</sub>	START	U		Time cycle to store restart data. (unused)
N	[n]	T		U	Mesh number in £ <sup>2</sup> direction. 1 < NN < 105.
NCNT	C,I,S <sub>0</sub>	START	U		Initial time step. (Input 3) (Always 0)
NCYC	T	POSITION	U		NCYC = NCYCLE-1.
NCYCH(I)	C,I,S <sub>0</sub>	START	U		Cycles for JCNIK and energy summary print. (Input 9)
NCYCLF	C,S	RPSLID	U		Time cycle. (DAMP)

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
NC3DP(1)	C,I,S <sub>0</sub>	START	U	U	Time steps for "3D" plots. (Input 11) (DAMP)
ND	D	DGEOM	U	U	Dummy for N in STRESS, BMSTRS, GRAD, and RESULT.
NDEFIX	C,S	STT,IN	R		N corresponding to DEFIX.
NDEFXT	T	STRAIN	R		N corresponding to DEFXT.
NDEL,P	C,S <sub>0</sub>	START	U		Time steps between surface strain prints.
NFPSX(1)	C,S	STRAIN	R		Mesh number of EPSX(1).
NFPSXT(1)	T	STRAIN	R		Mesh number of FPSXT(1).
NFST	C,S <sub>0</sub>	START	ESTRS		Number of EEB,SSB pairs in table. (NEST<100)
NETAG(I)	C,I,S <sub>0</sub>	START	U		Surface for I'th surface strain. (Input 13)
NF	T,I	INVEL	U		Maximum mesh point for uniform velocity. (Input 15)
NFMUS	I,S <sub>0</sub>	ENDFRC	APLFR		Number of EFMMT entries in table. (Input 18)
NFTIS	I,S <sub>0</sub>	ENDFRC	APLFR		Number of EFTHT entries in table. (Input 18)
NTZPTS	I,S <sub>0</sub>	ENDFRC	APLFR		Number of EFZT entries in table. (Input 18)
NI	T,I	INVEL	U		Minimum mesh point for uniform velocity. (Input 17)
NI1(1),NI2(1)	C,S <sub>0</sub>	START	U		Mesh points that bracket surface strain points.
NI3(1)	C,S <sub>0</sub>	START	R		Midmesh just less than surface strain points.
NLP	C,S	RPSLID	U		Counter for array JCYNLP(1). (DGEOM)
NLPRIN	T,I	START	U		Number of JCINP(1) entries. (Input 10)
NMESH1	C,I,S <sub>0</sub>	START	U		Number of meshes. (Input 2)
NN	C,S <sub>0</sub>	START	U		NN = N2B+1. (NN=N2B if IRCE2=4) (NN<105)
NNN	C,S	RPSLID	U		Counter indexing NCYCH(1). (MOTION)
NNPF	C,I,S <sub>0</sub>	START	PLOTP		Number of "PLOTP" plots to make. (Input 11a)
NN3D	C,S	PDATA	U		Counter indexing array NC3DP(1).
NOISE	T	START	R		Position for unwanted input.

<u>N#</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
NPE(1)	C,I,S <sub>0</sub>	START	PLOTP	N of P(N) at which "PLOT" plots are desired. If N>NN, P(N) may be any function of time inserted by programming. (Input 11a)	
NPLOT	C,S <sub>0</sub>	RPSL1D	U	File number for plotting data. (NPLOT = 3)	
NPRINT	C,S <sub>I</sub>	STRAIN	U	Next time step for surface strain print.	
NPPTS	T	PDATA	R	Number of points in "3D" plots. NPPTS = N2B-N1B+1.	
NQ1,NQ2	C,S <sub>0</sub>	START	U	Mesh points bracketing ETAD2 location.	
NRITE	C,I,S <sub>0</sub>	START	U	Restart print interval. (Input 3) (unused)	
NSFL	[J]	C,I,S <sub>0</sub>	START	Number of mechanical stress sublayers. (Input 6)	
NSTRN	C,I,S <sub>0</sub>	START	U	Number of surface strain locations. (Input 12)	
NTPTS	I,S <sub>0</sub>	ERODE	EROD	Number of TEROD,XEND pairs in table. (Input 19)	
NCYCY	T,I	START	U	Number of NCYCH(1) entries. (Input 9)	
NV	T,I	INVEL	U	Number of points with individual velocities. (Input 15)	
NIA,N2A	C,S <sub>0</sub>	START (ERODE)	R EROD	Limits on N for 1 loop in DGECOM. (Same as N1B,N2B) NIA increases if end point erodes (DE>DECOM).)	
NIB,N2B	C,S <sub>0</sub>	START (ERODE)	R EROD	N at end 1, end 2. N1B = NMESH+2. NIB increases if end point erodes (DE>DECOM).)	
NIV,N2V	C,S <sub>0</sub>	START (ERODF)	R EROD	Limits on N for points that move. NIV increases if end point erodes (DE>DECOM).)	
N3D	T,I	START	U	Number of time steps for "3D" plots. (Input 11)	
P(N)	$\Delta P, [P^*, P^**]$	C,S	PRESS	Pressure or force/unit initial area at mesh point N. (DGECOM) (If IB=1, P(N) is force/length.)	
PINT(1)	C,S	PDATA	PLOTP	Array for P(N), N=NPE(1), for plots.	
PHIT	T	STRESS	U	Yield function.	
PI	$\pi$	S <sub>0</sub>	U	3.141592653589793. (Too long for CDC.)	

<u>NAME:</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
PLAST	[N_p]	C,S	MOTION	U	Plastic work. PLAST = TNRG-CINET-STREN-TDAMP.
PN(I)	[n]	C,S <sub>0</sub>	START	U	PN(I) = 2.0 + ETAG2(I)/DETA2.
PN1,PN2		T	START	U	PN1 + NI1(I), PN2 = NI2(I).
PPR(I)	1	PRESS	LINRS		Pressure at time TPR(I). (Input 16)
PRAT	E/(1- $\nu^2$ )	C,S <sub>0</sub>	START	U	Material constant used in STRESS.
PSR(J)		C,I,S <sub>0</sub>	START	U	Constant for strain rate. (Input 7)
P0		I,S <sub>0</sub>	PRESS	R	Pressure in some PRESS subroutines. (Input 16)
QN	[n]	C,S <sub>0</sub>	START	U	QN = 2 + ETAD2/DETA2. NQ1 < QN < NQ2 = NQ1+1.
QN1,QN2		C,S <sub>0</sub>	START	U	Interpolants for displacement at ETAD2.
QS(N)	(Q)	C,S	RESULT	SIR3/1	Axial force at midmesh N.
Q11,Q22		T	RESULT	U	Proportional to membrane components of stress resultant. (Q11 = SUMA11 - BT*SUMB11)
R(N),Z(N)	[Y <sup>3</sup> , Y <sup>2</sup> ]	C,S	INGEOM	R	Coordinate of mesh point N. { $X = R \hat{x}_3 + Z \hat{x}_2$ }
Q111,Q222		C,S <sub>0</sub>	INGEOM	R	[Replaces $\xi = Y^j \hat{x}_j$ of REPSIL] (POSITION)
RADIUS					Signals Type of symmetry. If RADIUS>0, radial symmetry. (If IB>0, RADIUS = 0.0 in START.)
RD22	{1/(\Delta\xi) <sub>2</sub> <sup>2</sup>	C,S <sub>0</sub>	START	U	Program constants for differences.
RD22M	{1/(\Delta\xi) <sub>2</sub> <sup>2</sup>	C,S <sub>0</sub>	START	R	
RG	1/g	T	STRESS	U	RG = 1.0/DG;
RHO	$\rho$	T,I	START	U	Mass density. (Input 6)
RNSN	{R}	T	DGEOM	R	R at current mesh point or midmesh.
RNK,RNR	{n <sub>k</sub> , n <sub>r</sub> }	T	DGEOM	R	Components of normal at time t-At. [Replace RN <sub>j</sub> ]
RSUM		T	START	U	Temporary for computing At.
RTD2	{1/\Delta\xi}	C,S <sub>0</sub>	START	U	Constant for differences.
RTD2M	{1/\Delta\xi}	C,S <sub>0</sub>	START	R	Constant for differences.

NAME	SYMBOL	STATUS	STORED	ORIGIN	COMMENTS
RZ(N),ZZ(N)	C,S <sub>0</sub>	DGEOM	R		Initial position of mesh point N.
R2,R22	[Y <sub>2</sub> <sup>3</sup> ,Y <sub>22</sub> <sup>3</sup> ] [a, $\beta$ ]	C,T	GRAD	R	Difference approximations of $\partial R / \partial \xi$ , $\partial^2 R / (\partial \xi)^2$ .
SA,SB	T	STRAIN	U		Temporary constants for surface strains.
SE(I)	C,S <sub>0</sub>	START	U		Slope of J'th segment of stress-strain curve.
SERS(J),SSIG(I)	C,I,S <sub>0</sub>	START	U		Stress, strain at J'th corner of uniaxial stress-strain curve. (If $(e_s, \sigma_s)$ is a corresponding engineering stress-strain point, $e_j = e_s(1+e_s/2)$ , $\sigma_j = \sigma_s(1+e_s)$ .) (Input 7) $((\epsilon_1, \sigma_1) = (\text{SIG}/E, \text{SIGZ}))$
SIGYSQ	$(\sigma_0)^2$	T	STRESS	U	Square of dynamic yield stress.
SIGYZ	$\sigma_0$	T	BMSTRS	R	Dynamic yield stress.
SIGZ	$\sigma_0$	C,I,S <sub>0</sub>	START	U	Static uniaxial yield stress. (Input 6)
SIGZSQ(J)	$I(\sigma_{0j})^2$	C,S <sub>0</sub>	START	U	Square of static yield stress, sublayer j.
SIGZZ(J)	$[\sigma_{0j}]$	C,S <sub>0</sub>	START	R	Static yield stress, sublayer j. ( $\sigma_{0j} = E*\text{SEPS}(J)$ )
SIG1(N,KJ), SIG2(N,LJ)	$\sigma^{11}, \sigma^{22}$	C,S	STRESS	U	Contravariant components of stress, mesh point N, station K, sublayer J. (BMSTRS, RPSL1D)
SIG1M(N,KJ), SIG2M(N,KJ)	$\sigma^{11}, \sigma^{22}$	C,S	STRESS	R	Contravariant components of stress, midmesh N, station K, sublayer J. (BMSTRS, RPSL1D)
SIG1,I,SIG2,I	$\sigma_1, \sigma_2$	T	STRESS	U	Mixed components of stress. (BMSTRS)
SIG1ID,SIG22D	$C_1, C_2$ $c_1, c_2$	T	STRESS	U	Mixed components of plastic flow corrector stress.
SIG1II1,SIG22I	$\sigma_1^{-1}, \sigma_2^{-2}$	T	STRESS	U	Mixed components of stress at time t-At. (BMSTRS)
SIG1II1,SIG22I	$T_1, T_2$ $c_1, c_2$	T	STRESS	U	Mixed components of trial stress.
SLABL, SLARM	T,I	INGEOM	R		Length, width of slab symmetric plate. (Input 14)
SNK(N),SNR(N)	{n <sub>k</sub> ,n <sub>r</sub> }	C,S	DGEOM	R	Components of unit normal, mesh point N. (INNORM) [Replace SNI(M,N)]
SNRM(N),SNRM(N)	{n <sub>k</sub> ,n <sub>r</sub> }	C,S	DGEOM	R	Components of unit normal, midmesh N.

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
SNKN, SNRN	$(n_x, n_y)$	T	DGEOM MOTION	R EAPFRC	Components of unit normal at time t. Components of unit normal at time t+Δt.
SPA	$a_i^j$	C,T	DGEOM	U	SRA = SQRT(DA).
SPG	$g_i^j$	T	STRESS	U	SRG = SQRT(DG).
SSB(1)	$[\sigma_i]$	C,I,S <sub>0</sub>	START	BSTRS	See EEB(1).
SSIG(J)	$[\sigma_j]$	C,I,S <sub>0</sub>	START	U	See SEPS(J).
SSIM(K), SS2M(K)	$\sigma^{11}, \sigma^{22}$	C	STRESS	U	Contravariant components of stress, station K. (BMSTRS)
SS11, SS22	$\sigma_1^1, \sigma_2^2$	T	STRESS	U	Mixed components of stress for layer.
STREN	[V]	C,S	STRESS	U	Elastic strain energy. (BMSTRS, DGEOM)
SIM		T	START	U	Temporary for computing Δt.
SUMAR		T	START	R	EM(X), cross-sectional area for beans.
SUMA11, SUMA22	$r_0^{11}, r_0^{22}$	T	RESULT	U	Approximation of $\int_{ZL}^U \sigma^{\alpha\beta} d\zeta$ .
SUMR11, SUMB22	$r_1^{11}, r_1^{22}$	T	RESULT	U	Approximation of $\int_{ZL}^U \zeta^{\alpha\beta} d\zeta$ .
SUMC11, SUMC22	$r_2^{11}, r_2^{22}$	T	RESULT	U	Approximation of $\int_{ZL}^U (\zeta)^2 \sigma^{\alpha\beta} d\zeta$ .
SUM2AR		T	START	R	$\sum_k (\text{ZETA}(k))^2 \text{#W}(k)$ . (Moment of inertia for a beam.)
TA		C,S <sub>0</sub>	START	U,R	If IB=1, TA = 1. If IGAUSS = 1, TA = THICKN/2. Otherwise, TA = Δζ = THICKN/ZAYER.
TAMBDA	Δλ	T	STRESS	U	Factor for amount of plastic flow. (Ref. 1, App. C)
TB		T	RESULT	U	Program constant. TB = SRA*TA.
TDAMP	[W <sub>D</sub> ]	C,S	DAMP	U	Damping work. (RPSLD)
TDEFX		C,S	STRAIN	R	Time of DEFX, maximum displacement.
TEMP(N)		C,S <sub>0</sub>	DGEOM	U	Program constant for mesh point N. Used in MOTION and KINET. TEMP(N) = DELGAM/A <sub>3</sub> .

<u>NAME</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
TEMPU		T	ROUNDR	R	Temporary. (ROUNDU)
TEPSX(I)		C,S	STRAIN	R	Time for EPSX(I), the I'th extreme strain.
TEROD(I),XEEND(I)		I,S <sub>0</sub>	ERODE	EROD	Table of erosion vs. time at end 1. (Input 19)
THICKN	h	T,I	START	U	Thickness of shell. (Input 6)
TIME:	t	C,S	RPSLID	U	Time. (START-temporary for output) (DAMP)
TITLE(I)		I	START	U	Array for image of identification. (Input 1)
TMFZ(I)		I,S <sub>0</sub>	ENDFRC	APLFRC	Time with FMZT(I). (Input 18)
TMU(I)		I,S <sub>0</sub>	ENDFRC	APLFRC	Time with FFMU(I). (Input 18)
TMUH(I)		I,S <sub>0</sub>	ENDFRC	APLFRC	Time with FFMUH(I). (Input 18)
TNGC	[w]	C,S	MOTION	U	Total energy. Initial kinetic energy plus work by external forces.
TPR(I),PPR(J)		I,S <sub>0</sub>	PRESS	LINPRS	Time, pressure table for PRESS. (Input 16)
v	[v]	T,I	INVEL	U	Velocity. (Input 17)
VR	[v]	T,I	INVEL	U	Velocity. (Input 17)
VR,VZ		T	MOTION	R	Forces, equations of motion. [Replace VM <sub>i</sub> +VN <sub>i</sub> +VF <sub>i</sub> ]
VS(N)	(V)	C,S	RESULT	SIR3/1	Shearing force at midmesh N.
W(K)	{W <sub>k</sub> }	C,S <sub>0</sub>	INGROM	R	If IB = 1, W(K) is weight with K'th layer in beam. If 1GAUSS=1, W(K) is weight with K'th Gauss point.
WIDTH(K)		T,I	INGROM	R	Width of K'th layer of beam. (Input 14)
WT(J)		C	START	U	Weight associated with J'th mechanical sublayer.
WEFTA(K)		C,S <sub>0</sub>	START	R	Constants for Gaussian integration.
WEFTSQ(K)	{(W <sub>kL</sub> ) <sup>2</sup> }	C,S <sub>0</sub>	ERODE	EROD	See TEROD(I).
XEEND(I)		I,S <sub>0</sub>	START	U	Parameter for plastic flow computation. (Input 2)
YLDFAC		C,I,S <sub>0</sub>	INGROM	R	See R(N). (POSITM)
Z(N)		C,S			

<u>NAME:</u>	<u>SYMBOL</u>	<u>STATUS</u>	<u>STORED</u>	<u>ORIGIN</u>	<u>COMMENTS</u>
ZAYER	T	START	U		Floating point representation of LAYER.
ZB	T	STRAIN	R		$\zeta$ on upper, or lower, surface, (ZU or ZL)
ZETA(K)	C,S <sub>0</sub> I	START	R		IF IGAUSS=1, Gaussian points through thickness. If IB=1, integration points in beams. (Input 14) Otherwise, midpoints of equal layers.
ZETAK	T	STRESS	U		2.0*ZETA(K).
ZETASQ(K)	( $\zeta$ ) <sup>2</sup>	C,S <sub>0</sub>	START	U	Square of ZETA(K).
ZL	{ $\zeta_L$ }	C,S <sub>0</sub>	START	R	$\zeta$ on lower surface. [Replaces -H.]
ZH	{ $\zeta_H$ }	C,S <sub>0</sub>	START	R	$\zeta$ on upper surface. [Replaces H.] (Input 14)
ZZ(N)	C,S <sub>0</sub>	DGEOM	R		See RZ(N).
Z2,Z22	[ $y_2^2, y_{22}^2$ ]	T	GRAD	R	Approximations for $\partial z / \partial \xi$ , $\partial^2 z / (\partial \xi)^2$ .

## APPENDIX C

### LISTING OF REPSL1D

The version of RPSL1D discussed in this report is catalogued in file RPSL1D, cycle 2, in UPDATE form at the time of this writing. There is one COMDECK called MAIN. This is listed first. The remainder of the listing is the COMPILE file image of RPSL1D formed through UPDATE with the COMDECK images replaced with the COMMENT statement \*CALL MAIN HERE.

This listing gives the correct UPDATE card identifiers. It could be used for simple changes through UPDATE. For involved changes, the user would be wise to use the FTN compiler to create symbolic reference maps.

```

*COMMDCK MATH
COMMON L(103),Z(103),PL(103),DZ(103),FL11P(103),FL22W(103),MAIN 2
1 FM22K(103),FN1TH(103),FN2R(103), FN2K(103),SNK(103),SK(103),MAIN 3
2 TEMP(103),P(103),FPSL1(103),EPSI2(103),FPSU1(103),FPSU2(103),MAIN 4
3 SIG1(103,36),SIG2(103,36),LMAT(103,6),MAIN 5
4 DEPS1(6),DEPS2(6),ZETA(6),ZFTASC(6),SS1MM(6),SS2MM(6),W(6),MAIN 6
5 NCYCH(50),AC3DH(50),JCYNLP(50),MAIN 7
COMMON NN, RD22,HTD2,DETA1,DETA2,RADIUS,NN3D,DAMMF,MAIN 8
1 DFACT,NDAMP,TDAFP,LOAD,DELGAM,MAIN 9
COMMON F,FNU,G,PHAT,SIGZ,GANZ,ZU,ZI,AYER,DELTAT,TIME,LPHES4,MAIN 10
1 NNN,NCYCLF,NWTF,ACONT,ASTHN,CINER,CINFS,CINFP,C1,C2,NPLOT,MAIN 11
2 DELSG,TA,MAXC,MWHITE,CF,CH,CINET,STREN,PLAST,TANG, EPS,FMH,MAIN 12
3 NPRINT,MDLP,NMESH,INCE1,INCE2,ISh,NSFL,KUMAX,YLDIFAC,ALH,MAIN 13
COMMON N1(6),N12(6),DN1(6),DN2(6),PN(6),FTAG2(6),ANGLE(6),MAIN 14
1 ANGLP(6),NETAG(6),FPSS1(6),FPSS2(6),JCHK(3),MAIN 15
COMMON BN,NQ1,NQ2,GN1,GN2,D1,D2,FTAD2,MAIN 16
COMMON ASA(6),HSA(6),ASB(6),HSR(6),GI11(6),GI22(6),MAIN 17
1 A111(6),A221(6),H111(6),H221(6),A112(6),A222(6),H112(6),H222(6),MAIN 18
COMMON DSH(6),PSH(6),SSIG(6),SEPS(6),SF(6),SIGZSG(6),WT(6),MAIN 19
COMMON SIGZZ(6),JH,MAIN 20
COMMON F2,Z2,DR2,D22,F22,Z22,DR22,DZ22,MAIN 21
COMMON A11,A22,SHA,CS211,CS222,H11,H22,BT,HH11,HH22,CINES1,CINES2,MAIN 22
COMMON IS,MAIN 23
COMMON N1R,N2R,N1V,N2V,MAIN 24
COMMON N1A,N2A,IP,HTD2M,RDPPM,MAIN 25
COMMON SNHM(103),SNKM(103),SIG1M(103,36),SIG2M(103,36),LMATM(103,6),MAIN 26
COMMON DFX,NDFX,TDFX,EPSX(R),NFPDX(R),TFPSX(H),MAIN 27
1 RZ(103),ZZ(103),GU11(103),GU22(103),GL11(103),GL22(103),MAIN 28
COMMON NM3(6),N13(6),PA1(103),DP1(103),PA2(103),DP2(103),MAIN 29
COMMON IGAUSS,WZFTA(6),WZFTSU(6),MAIN 30

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DELGAM=DELSQ/RFMZ
CALL DGFUN
CALL STRAIN
DO 33 N=M1H,N2F
  A22 = 0.5*(DA2(N)+PA2(N-1))
  IF(N .EQ. N1H)A22=PA2(N)
  IF(N .EQ. N2F)A22=DA2(N-1)
  GU11(N) = 1.0/(DA1(N)-2.0*ZU*DPL1(N))
  GU22(N) = 1.0/(-422-2.0*ZU*DPL2(N))
  GL11(N) = 1.0/(DA1(N)-2.0*ZL*PA1(N))
  GL22(N) = 1.0/(-422-2.0*ZL*DPL2(N))
  DA1(N) = 0.0
  DA2(N-1) = 0.0
  DPL1(N) = 0.0
  DPL2(N) = 0.0
33 CONTINUE
  DA2(N2F) = 0.0
  EPSU2(N2F) = 0.0
  EFSL2(N2F) = 0.0
  IF(LOAD .GT. 0)GOTO 43
C
C     CALL INVFL
C
C     DO 35 N=M1V,N2V
C       DR(N)=DELTAT*UR(N)
C       DZ(N)=DELTAT*UZ(N)
35 CONTINUE
C     CALL ROUNDII
C     CALL KINFT
C     CINES=2.0*CINFT
C     TMRG=CINES
C
C     IF(LOAD) 42,45,43
C
C     42 CALL PWOK
C
C     43 DO 44 N=M1V,N2V
C       DR(N)=DR(N)-P(N)*SNR(N)*TEMP(N)
C       DZ(N)=DZ(N)-P(N)*SNK(N)*TEMP(N)
44 CONTINUE
C     CALL ROUNDII
C     CALL PWOK
C     FNC=FNC
C     CALL KINET
C     TMRG=CINFT
C     45 IF(NCYCL(1) .EQ. 0)RNN=2
C        WRITE INITIAL CARTESIAN COORDINATES, PRESSURE
C        WRITE(6,300)
C        WRITE(6,410)(X,P(N)+Z(N),P(N),N=1,N)
C
C     46 CALL PWATA (1)
C
C     END INITIALIZATION
C
C     50 NCYCLE=NCYCLE+1
C     TIME=TIME+DELTAT
C     CHECK FOR FINAL STEP
C     IF(NCYCL .GT. MAXC)GOTO 70
C     CHECK IF CALL PPFSS IS NEEDED
C     IF(LPPFSS .GE. NCYCLE)CALL PPFSS
C     CALL POSITN
C     CALL STRAIN
C
C     70 CONTINUE
C
C     71 CALL PWOK
C
C     72 CALL PWOK
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C     73 CALL PWOK
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C     74 CALL PWOK
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C     75 CALL PWOK
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C     76 CALL PWOK
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C     77 CALL PWOK
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C     78 CALL PWOK
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C     79 CALL PWOK
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C     80 CALL PWOK
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C     81 CALL PWOK
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C     105 CALL PWOK
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C     106 CALL PWOK
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C     107 CALL PWOK
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C     108 CALL PWOK
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C     109 CALL PWOK
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C     110 CALL PWOK
C
C     111 CALL PWOK
C
C     112 CALL PWOK
C
C     113 CALL PWOK
C
C     114 CALL PWOK
C
C     115 CALL PWOK
C
C     116 CALL PWOK
C
C     117 CALL PWOK
C
C     118 CALL PWOK
C
C     119 CALL PWOK
C
C     120 CALL PWOK
C
C     121 CALL PWOK
C
C     122 CALL PWOK
C
C     123 CALL PWOK
C
C     124 CALL PWOK
C
C     125 CALL PWOK
C
C     126 CALL PWOK

```

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CALL MOTION          RFSI,10  127
CALL PDATA (2)      RFSI,10  128
CALL DAMP           RFSI,10  129
C     CHECK FOR RESTART LUMP
C     IF (NCYCLF .NE. NRITE) GOTO 50
CALL WRITAPE (1)    RFSI,10  130
C     NRITE=NRITE+NWRITE
C     CALL PDATA (3)   RFSI,10  131
C     GOTO 50          RFSI,10  132
    70 IF (NCYCLE .LT. 2) STOP! MAIN. NCYCLE .LT.
    CALL PDATA (3)   RFSI,10  133
    CALL PDATA (4)   RFSI,10  134
    STOP! PROGRAM COMPLETE!
300 FORMAT(1H1,20X,10HINITIAL COORDINATES,23X,1HPRESSURE//  

1 3X,1H,11X,4H(N),21X,4H(Z),21X,4H(R))
400 FORMAT((I4,3(2X,E23.16)))
    FND
    SUBROUTINE START
C     *CALL MAIN HERE
    DIMENSION TITLE(8)
    READ(F,100) TITLE
    WRITE(6,140) TITLE
    FEAD(5,105) NOUSE,NMESH,LAYER,YLDFAC
    FEAD(5,105) MAXC,NCONT,NRITE,DELTAT
    FEAD(5,110) NOUSE,IPCE2,NOUSE,IPCE1
    FEAD(5,115) LOAD,LPRESS,MCANF,DANPF,DFACT
    FEAD(5,120) E,FNU,SIGZ,FHO,THICKN,NSFL,ISH
    WRITE(6,160) E,FNU,SIGZ,FHO,THICKN
    WRITE(6,170) NCONT,MAXC,NPINT,NRITE
    WRITE(6,175) LAYER,ASTRN,LOAD,LPRESS
    WRITE(6,180) IPCE1,IPCE2
    IF (NSFL .EQ. 1 .AND. ISF .EQ. 0) GOTO 700
    IF (NSFL .EQ. 0) GOTO 700
    READ (5,125) (SSIG(J),SFPS(J),OSR(J),PSR(J),J=1,NSFL)
    700 IF (NSFL .LT. 1) ISP=-1
    IF (NSFL .LT. 1) NSFL=1
    READ(5,110) NPRINT,(JCRR(J),J=1,3)
    READ(5,110) ALMCY,(NCYCH(J),J=1,NUMCY)
    READ(5,110) ALPHIN,(JCYNLP(J),J=1,NLPRIN)
    READ(5,110) N3D,(NC3DH(J),J=1,N3D)
    SSIG(1)=SIGZ
    SFPS(1)=SIGZF
    KJMAX=LAYER*NSFL
    SF(1)=E
    SIGZSQ(1)=SSTG(1)**2
    SIGZZ(1)=SSIG(1)
    PH 700 J=1,NSFL
    IF (ISP .LT. 1) GOTO 704
    IF (OSR(J) .GT. 0.0 .ANI. PSR(J) .GT. 0.0) GOTO 743
    741 WRITE(6,702)
    742 FORMAT(1/4AH ERROR IN STRAIN MAPPING OR STRAIN RATE DATA )
    STOP! START. ERROR IN STRAIN DATA!
    743 PSR(J)=1.0/PSR(J)
    704 IF (J .EQ. 1) GOTO 795
    IF (SFPS(J).LE.SFPS(J-1)) GOTO 791
    SF(J)=(SSTG(J)-SSIG(J-1))/(SFPS(J)-SFPS(J-1))
    WT(J-1)=(SF(J-1)-SF(J))/E
    SIGZSL(J)=(E*SFPS(J))**2
    SIGZZ(J)=(E*SFPS(J))
    705 CONTINUE
    WT(NSFL)=SF(NSFL)/F
    CFT UP BOUNDARY CONDITIONS
    IFC(I=1,2,4,6,8,10)=1,FC(I=1,2,4,6,8,10)=0
    75

```

```

N1H=2
NPP=NM+1
NV=N1H+1
N2V=N2H+1
N1A=N1V+1
N2A=N2V+1
IF (IH>F1 .LE. 2) N1V=V1H
IF (IH>F2 .LE. F2+0.04 .LT. F2+0.4) N2V=N2H
NM=N2A+1
IF (IH>F2 .LE. 4) NA=N2H
READ(5,130) NCOUSE,ETA02,NSTRN
NCAP(5,15) = (NCOUSE+ETAG2(I)+ANGLE(I)+NETAG(I)+I=1+NSTRN)
CALL INGFM
CALL INMOPW
C SIGNALS TO PROGRAM FROM INGEOM.
C      AXIAL SYMMETRY (RADIUS > 0.0,  I=0).
C      SLAB SYMMETRY (RADIUS = 0.0  I=0).  REAM (IH>0).
C IS = 1
IF (IH .GT. 0) RADIUS = 0.0
IF (RADIUS .LE. 0.0) IS=2
IF (IR .GT. 0) DETA1=1.0
RN=FLOAT(N1H) +ETA02/DETA2
RN1=0.0
RN2=RN1+1
RN1=RN-FLOAT(RN1)
RN2=FN(FLOAT(RN2)-RN)
IF 20  I=1,NSTRN
RN(I)=FLOAT(N1H) +ETAG2(I)/DETA2
NI1(I)=RN(I)
NI2(I)=NI1(I)+1
RN1=NI1(I)
RN2=NI2(I)
RN1(I)=RN(I)-RN1
RN2(I)=RN2-RN1(I)
IF (DN1(I) .GT. 0.5) GOTO 16
C      DN1(I) LE 1/2.
NI3(I) = NI1(I)-1
DN3(I) = DN1(I) + 0.5
GOTO 17
C      DN1(I) GT 1/2
16 NI3(I) = NI1(I)
RN3(I) = RN1(I) - 0.5
17 IF (NI3(I) .GT. 1) GOTO 18
C      NI3(I) LT 2.  SHIFT TO 2.
NI3(I) = 2
RN3(I) = 0.0
GOTO 20
C      IF (NI3(I) .LT. N2H-1) GOTO 20
NI3(I) GT N2H-2.  SHIFT TO N2H-2.
NI3(I) = N2H-2
RN3(I) = 1.0
20 CONTINUE
ZAYER=1AYER
C      PROGRAM CONSTANTS
IF (IH .LE. 0) ZU=0.5*THICKN
ZL=ZU-THICKN
GAM=THICKN*PI
NDFLP=NPFINT
NWHITE=NCOUNT*NWHITE
SUMAP=0.0
SUMAR=0.0
      READ(5,15) IS  SIGNAL FOR GAUSSIAN INTEGRATION THRU THICKNESS.
IF (IH .LE. 0) IS=0.0

```

	START	44
	START	49
	START	50
	START	51
	START	52
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	START	86
	START	87
	START	88
	START	89
	START	90
	SIGN	-1
	START	92
	START	93
	START	94
	START	95
	START	96
	START	97
	START	98
	START	99
	START	100
	START	101
	START	102
	START	103
	START	104
	START	105
	START	106
	START	107
	START	108
	ST/LT	109
	SIGN	110

```

      IF (IGAUSS .NE. 1) GOTO 50
C      SET UP FOR GAUSSIAN INTEGRATION WITH LAYER (I,LE,F) POINTS
      GOTO(61+62+63+44+45+46)*LAYFF
  41 ZETA(1) = 0.0
      V(1) = 2.0
      GOTO 50
  42 ZFTA(2) = 0.57775026515626471
      ZFTA(1) = -ZFTA(2)
      W(1) = 1.0
      V(2) = 1.0
      GOTO 50
  43 ZFTA(3) = 0.774506640241483*7D
      ZETA(2) = 0.0
      ZFTA(1) = -ZETA(3)
      V(1) = 0.5555555555555555
      W(2) = 0.6444444444444444
      V(3) = W(1)
      GOTO 50
  44 ZETA(4) = 0.461136311594053*7D
      ZFTA(3) = 0.334441043584856*7D
      ZFTA(2) = -ZFTA(3)
      ZFTA(1) = -ZETA(4)
      W(1) = 0.347854845137454
      W(2) = 0.652145154862546
      W(3) = W(2)
      V(4) = W(1)
      GOTO 50
  45 ZFTA(5) = 0.406179845938664*7D
      ZFTA(4) = 0.534649310105643*7D
      ZFTA(3) = 0.0
      ZFTA(2) = -ZETA(5)
      ZFTA(1) = -ZETA(5)
      W(1) = 0.236926885056149
      W(2) = 0.478624670499366
      W(3) = 0.5644444444444444
      W(4) = W(2)
      W(5) = W(1)
      GOTO 50
  46 ZETA(6) = 0.932469514203152*7D
      ZFTA(5) = 0.661209346464256*7D
      ZFTA(4) = 0.234614146463197*7D
      ZFTA(3) = -ZFTA(4)
      ZFTA(2) = -ZETA(5)
      ZFTA(1) = -ZFTA(5)
      V(1) = 0.1713264442374170
      V(2) = 0.360761573048139
      V(3) = 0.467913934572691
      V(4) = W(3)
      V(5) = V(2)
      V(6) = V(1)
  50 C(ONTINUE
      F0_3 K=1*LAYFF
      IF (IGAUSS .NE. 1) GOTO 51
      IF (JF .LT. 0) ZFTA(K)=2.0-(2.0*FLOAT(K)-1.0)/ZAYFF
  51 C(ONTINUE
      SUMP=SUMPAR+V(K)
      ZFTASU(K) = ZFTA(K)**2
      SUMPAR=SUMPAR+(K)*ZFTASG(K)
      WZFTA(K) = V(K)*ZFTA(K)
      WZFTSG(K) = ZFTA(K)*WZFTA(K)
  3 CONTINUE
      TA=THICKN/ZAYFF
      V(I)=V(I)+TA*I=1.0
      START 111
      START 112
      START 113
      START 114
      START 115
      START 116
      START 117
      START 118
      START 119
      START 120
      START 121
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      START 167
      START 168
      START 169
      START 170
      START 171
      START 172
      START 173

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IF (IGAUSS .EQ. 1) TA=2D
IF (IH .GT. 0) GAMZ=RHO*SLMAH
      TIME INCREMENT BY VOF ALUMAN
C=SMTH/E/RHO
IF (IR .LE. 0) ( =C/SQRT(1.0-F(1.0+C))
DELM=DETA2/C
CTK=(1.0-1.0/2AYFH**2)*TH(CAN**2/12.0
IF (IH.GT.0) CTK=SUMPAH/SUMAH
DELR= 0.5*DETA2**2/(C*SMTH(CTK))
IF (IGAUSS .EQ. 1) DFLH = SQRT(3.)*DETA2**2/(C*THICKN)
DELMIN=A MIN(DFLH,DELM)
DELMIN= 0.95*AMIN(DFLH,DELM)
SIM=1.0
25 FSUM=1.0/SUM
SIM=SUM*10.0
IF (RSUM .LT. DELMIN) GOTO 30
GOTO 25
30 DFLMIN=A INT(DELMIN*SUM)/SUM
DELIN=DELTAT
IF (DELIN .GT. 0.0) DFLMIN=AMIN1(DELIN,DELMIN)
DELTAT=DELIN
DELSH=DELTAT**2
C=.5*F/(1.0-FNU**2)
PFAT=F/(1.0-FNU**2)
HT02=1.0/(2.0*DETA2)
HL22=1.0/DETA2**2
HT02M=1.0/DETA2
HL22M=0.5/DETA2**2
CA=DETA1*DETA2
CF = C*.5*TA*DETA1*DETA2/5
      DAMPING CONSTANTS
C2=2.0*DELTAT*DAMPF/GAMZ
C1=C2/(4.0+C2)
      WRITE(6,130)
      WRITE(6,140) TITLE
      WRITE(6,150) UETA1, NMESH,DETA2
      WRITE(6,160) DFLH,DELM,DELIN,DELTAT
      WRITE(6,160) F,FNU,SIGZ,RHO,THICKN
      WRITE(6,170) NCNT,NMAX,NPRINT,NRITE
      WRITE(6,175) LAYER,ASTRA,LOAD,LPHESS
      WRITE(6,180) THCE1,THCE2
      WRITE(6,185) (JCHK(I),I=1,3)
      WRITE(6,190) (J,ICN(I),I=1,NLCYC)
      WRITE(6,190) (JCYNLH(J),J=1,NLPRINT)
      WRITE(6,195) (NCBEP(I),I=1,NBL)
      IF (ISF .EQ. -1) WRITE(6,400)
      IF (NSFL .NE. 1 .AND. ISH .EQ. 1) WRITE(6,405)
      IF (NSFL .LT. 1 .AND. ISH .EQ. 0) WRITE(6,410)
      IF (NSFL .EQ. 1 .AND. ISP .EQ. 0) WRITE(6,415)
      IF (NSFL .LT. 1 .AND. ISP .EQ. 1) WRITE(6,420)
      IF (NSFL .LT. 1) WRITE(6,420) NSFL
      WRITE(6,421) (L,SSIG(J),SEPS(J),DSP(J),PSR(J),J=1,NSFL)
      WRITE(6,421)
      TIME=DELTAT*FLDAT(NPANE)
      WRITE(6,200) NUMP,TIME,DAMPF,DEFACT
      IF (JHCH1 .LT. 4 .AND. THCE2 .LT. 5) RETURN
      WRITE(6,46474)
      STOP! START. END FILE DEFINITION
06474 FORMAT(20x,'FILE CONDITIONS NOT ALLOWED')
15 FORMAT(1H10.4,1H)
16 FORMAT('0217')
107 FORMAT(315.2,17.0)

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110 FORMAT(1E15)
115 FORMAT(3I5,2F12.6)
120 FORMAT(5E12.6,1E14)
125 FORMAT(4E15.7)
130 FORMAT(2E10.6,1E5)
140 FORMAT(1H1,5Z*1SHPRAL HEPSSIL COEFF//24X,1H10/)
150 FORMAT(43X,74FLUWHI:TATIONS FOR ENERGY USE DETA1=E12.6/
1      3HX,14.25H MESHES IN ETAB DIRECTION.3A,7H(DETA2=E12.6+1H)/) START 237
160 FORMAT(32X,17HYUNG'S MODULUS =E12.6
1      /32X,17HYUSSON'S RATIO =E12.6,10X,17HYIELD STRESS =E12.6 START 238
2      /32X,17HMASS DENSITY =E12.6+10X,17HTHICKNESS =E12.6 START 239
3/1) START 240
170 FORMAT(55X,1HMSTART AT TIME STEPIS/55X,1HMFINAL TIME STEP +15/ START 241
1      55X,21HSURFACE STRAINS EVERYIS,10M TIME STEP/ START 242
2      55X,21HRESTART WHILE EVERYIS,10M TIME STEP/) START 243
175 FORMAT(43X,7HAYER =I5,18X,8HMSTAN =I5/ START 244
1      43X,7HLOAD =I5,1HX,8HLPRESS =I5/) START 245
180 FORMAT(54X,1HINITIAL CONDITIONS/24X,*1/2/3/4/ = CLAMPED// START 246
1      *SYMMETRY/HINGED/HFFE/
2      50X,*FEND (IHCE4) =*,14/50X,*ENDP (IHCE2) =*,14/) START 247
185 FORMAT(50X,25HPRINT OPTION CONTROL CARD/52X,20H0/1 = NO PRINT/PIN) START 248
1T/50X,14.24H DISPLACEMENT INCREMENTS/ START 249
2 50X,14.32H CARTESIAN COORDINATES, PRESSURE/ START 250
3 50X,14.33M SURFACE NORMAL VECTOR COMPONENTS/) START 251
190 FORMAT(24X,45HPRINT INFORMATION AT THE FOLLOWING TIME STEPS/(24X*
1(1E15))) START 252
195 FORMAT(24X,3H#3=0) PLOTS FOR THE FOLLOWING TIME STEPS/(24X*(1E15))) START 253
200 FORMAT(/64X,29HSTART DAMPING AFTER TIME STEPIS,5X,6HTIME =E10.4/
140X,7HDPNPF =F10.4+16X,7HDFACT =F10.4) START 254
205 FORMAT(47X,25HLEAVING TIME INCREMENT= E12.6/67X,25HMEMPHANE TIME START 255
1INCREMENT= E12.6/67X,25HINPUT TIME INCREMENT= E12.6//44X,31HTIM START 256
2F INCREMENT USED BY REFSIL= F12.6/) START 257
400 FORMAT(/37X,32HCONSTITUTIVE RELATION ELASTIC) START 258
405 FORMAT(/37X,7HCONSTITUTIVE RELATION ELASTOPLASTIC-NO WORK HARD START 259
1ING-STRAIN RATE INDEPENDENT) START 260
410 FORMAT(/37X,77HCONSTITUTIVE RELATION ELASTOPLASTIC-WORK HARDENI START 261
1NG-STRAIN RATE INDEPENDENT) START 262
415 FORMAT(37X,80HCONSTITUTIVE RELATION ELASTOPLASTIC-NO WORK HARD START 263
1ING-STRAIN RATE INDEPENDENT) START 264
420 FORMAT(37X,75HCONSTITUTIVE RELATION ELASTOPLASTIC-WORK HARDENIN START 265
1G-STRAIN RATE INDEPENDENT) START 266
430 FORMAT(24X,49HPRINT L MATRIX (LMAT) AT THE FOLLOWING TIME STEPS/ START 267
1(24X*1E15)) START 268
#20 FORMAT(35X,31HSTRESS=STRAIN APPROXIMATION HAS,13+1D SUPERLAYER) START 269
#21 FORMAT(/46X,40HSTRESS=STRAIN AND STRAIN RATE PARAMETERNS/
130X,1HJ,4X,7HSEPS(J),4X,7HSEPS(J),14X,6HDSH(J),4X,8H1/PSH(J)/) START 270
2(26X*15.6)+1E215.7,5X,2E15.7)) START 271
END START 272
SUBROUTINE TVVFL
      *CALL MAIN HEIF
      EVALUATE THE INITIAL VELOCITY AT TIME=0 FOR ALL MESH POINTS
      INPUT INDICES RELATIVE TO FEND POINT = 1
      #FEND(5,1D0)NI,NF,VH,NV
      NI=NI+N1H-1
      NF=NF+N1H-1
      WRITE(6,200) NI,NF,VH
      .
      DO 30 N=NJ,NF
      RG(N)=VH*SNR(N)
      GZ(N)=VH*SNK(N)
      30 CONTINUE
      IF (AV>1.F. .OR. AV<0) GO
      WRITE(6,300)
      .
      .
      .

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      DO 45 K=1,NV
      READ(5,500)N,V
      N=N+1
      WRITE(6,400)N,V
      U(N)=U*SNR(N)
      DZ(N)=U*SNK(N)
45  CONTINUE
50  RETURN

C
100 FORMAT(10X,PIC,F12.4,IS)
200 FORMAT(1H1,40X,2H=,I3,I3,I3, 1FH VELOCITY (VR)=,E12.6/)
300 FORMAT(52X,27HUTHER VEL/CITY DISTRIBUTION/63X,1HN+RX+1HV/)
400 FORMAT(59X,IS,2X,E23.16)
500 FORMAT(5X,IS,F12.6)
END
SUBROUTINE POSITN
*CALL MAIN HERE
DO 50 N=1,NM
  R(N)=P(N)+DR(N)
  Z(N)=D(N)+DZ(N)
50  CONTINUE
IF(NCYCLE .NE. NCYC(MNN))GOTO 75
IF(JCHK(1).LE.0.AND.JCHK(2).LE.0)GOTO 75
      WRITE DISPLACEMENT INCREMENTS, COORDINATES, AND PRESSURE
NCYC=NCYCLE-1
WRITE(6,101)NCYCLE,TINF
WRITE(6,102)NCYC,NCYCLE
WRITE(6,103)
WRITE(6,104)(N,DR(N),DZ(N),R(N),Z(N),P(N),N=1,NM)
75  RETURN
100 FORMAT((15.5(2X,E23.16)))
101 FORMAT(10HITIME STEP,IS,4X,4HTINF,F16.8)
102 FORMAT(1/6X,36HDISPLACEMENT INCR&FMS RET&FFN T.S.,I4,4H AND,I4,
     1 2IX+11H(COORDINATES+27X,8HPRESSURE))
103 FORMAT(4X,1HN+12X,5HDR(N)+20X,5HDZ(N)+21X,4H(N)+21X,4H
     1 P(N))
END
SUBROUTINE DECOM
*CALL MAIN HERE
STRFM=0.0
A11=1.0
H11=0.0
CS21=0.0
C11=0.0
DH11=0.0
1C 41 TP=1,2
      TP=1 DENOTES MESH POINTS.  TP=2 DENOTES MESHFS
1C 40 N=M1A,N2A
1F(TP,FG,2)G(TL,12
  RMSNR(N)
  RMSR=DH(N)
  RNK=SNR(N)
  RAK=SNK(N)
  GOTO 13
12  RNH=SNRH(N)
  RAK=SNRK(N)
C      AVERAGE RADIUS AND CHANGE OF RADIUS. IN MESH
  RMSR=0.5*(R(N)+R(N+1))
  RMSN=0.5*(RN(N)+RA(N+1))
13  CONTINUE
  CALL (RAN)(N)
  IF(TS .EQ. 1)A11=RMSN#2

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I=SIGNT(A22)
DA=A11*E22
SA=S(4T(E4)
CM1=72/1
SNK1=-42/1
TF(IP ,FG, 2)GOTO 22
      14=1 LENGTHS MESH POINTS.  IP=2 LENGTHS FEMSH
IF(LHFFSS .GF, NCYCLE)P(M)=SHAEP(M)
TF(N .EG, N1H .AND. IHCF1 .EQ. 1)GOTO 20
IF(N .FG, N2H .AND. IHCF2 .EQ. 1)GOTO 20
SA(M)=SMHN
SNK(M)=SNKN
GOTO 23
20 SMHN=SMR(P)
SNK1=SNK(M)
GOTO 23
22 SMRN(M)=SMHN
SNKM(M)=SNKN
24 CONTINUE
IF(1S .FG, 1)F11=-SMRN*EMSN
H22=SMRN*F22+SNK1*Z22
IF(NCYCLE.GT.0)GOTO 98
DA2(M) = A22
IF(IP .FG, 2)GOTO 98
TF(P(N)=EFLGAM/SRA
PA1(M) = H11
PH1(M) = H11
PH2(M) = H22
H7(M)=H(N)
Z2(M)=Z(I)
GOTO 98
44 AP11=A22/DA
AH22=A11/DA
HM11=AP11*P11
HM22=AP22*H22
PT=HM11+HM22
IF(1S .FG, 1)CS211=-HMSN*H2*AH22
CS222=(H2*AH22+Z2*Z22)*AH22
PNL2=-((H2*AH22+HMSN*H2*AH22)
DNR2=DNL2*AH22
DN=DNL2*PNL2/(1.0+PPD*SMHN+HNSN*SNKN)
DNH=H2*PNL2+DN*SMHN
DNK=Z2*PNL2+(1*SMKN
IF((1S .FG, 1)OR((IP .GT. 1)OR((HMSN=0.0-.1*HNSN))
DA22=H2*(P2=0.5*H2)+V2*(Z2=0.5*Z2)
IF(1S .FG, 1)DH11=-(HMSN*HNSN+HNSN*PNL1)
FH22=H2*(H22+HNSN*P722+DNR*H22+DAK*Z22
DN 40 K=1,1AYER
EPS1(K)=PA11-ZETA(K)*H11
EPS2(K)=DA22-ZETA(K)*H22
TF(IH .GT. 0) CALL HNSTFS(M,K)
IF(IP .LE. 0)CALL STRESS(M,K)
      100 STORES BEST ESTIMATE OF SURFACE STRAINS.
40 CONTINUE
IF(IP .FG, 2)GOTO 140
PA1(M) = PA1(M) + PA11
PA1(M) = PA1(M) + PH11
PH2(M) = PH2(M) + PH22
EPSL1(M) = EPSL1(M) + PA11 - ZL*PH11
EPSL2(M)=EPSL2(M) - ZL*PH22
EPSU1(M) = EPSU1(M) + PA11 - ZU*H11
EPSU2(M)=EPSU2(M) - ZU*PH22
GOTO 140

```



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C      DIFFERENCES FOR FN (ENTERED AT MESH POINTS, FOR FN IN MESHES    GFAU      6
      N=1:
      IF(10 .LE. 2)GOTO 10
      IF(N .LT. NN) GOTO 9
C      SPECIAL CODING FOR FREE (A FREE END)
      R2=RTHI2*( R(N-2)-4.0* R(N-1)+3.0* R(N))
      Z2=RTHI2*( Z(N-2)-4.0* Z(N-1)+3.0* Z(N))
      RH2=RTHI2*( DH(N-2)-4.0* DH(N-1)+3.0* DH(N))
      DZ2=RTHI2*( DZ(N-2)-4.0* DZ(N-1)+3.0* DZ(N))
      R22=RTHI2*(2.0* R(N)-5.0* R(N-1)+4.0* R(N-2)- R(N-3))
      Z22=RTHI2*(2.0* Z(N)-5.0* Z(N-1)+4.0* Z(N-2)- Z(N-3))
      DH22=RTHI2*(2.0* DH(N)-5.0* DH(N-1)+4.0* DH(N-2)- DH(N-3))
      DZ22=RTHI2*(2.0* DZ(N)-5.0* DZ(N-1)+4.0* DZ(N-2)-DZ(N-3))
      GOTO 20
C      CONTINUE
      R2=RTHI2*( R(N+1)- R(N-1))
      Z2=RTHI2*( Z(N+1)- Z(N-1))
      RH2=RTHI2*( DH(N+1)-DH(N-1))
      DZ2=RTHI2*( DZ(N+1)-DZ(N-1))
      IF(N .LT. NN-1) GOTO 19
C      SPECIAL CODING FOR FREE (A FREE END)
      R22=RTHI2*(3.0* R(N+1)-7.0* R(N)+5.0* R(N-1)- R(N-2))
      Z22=RTHI2*(3.0* Z(N+1)-7.0* Z(N)+5.0* Z(N-1)- Z(N-2))
      DH22=RTHI2*(3.0* DH(N+1)-7.0* DH(N)+5.0* DH(N-1)-DH(N-2))
      DZ22=RTHI2*(3.0* DZ(N+1)-7.0* DZ(N)+5.0* DZ(N-1)-DZ(N-2))
      GOTO 20
C      CONTINUE
      R22=RTHI2*( R(N+1)- R(N)- R(N+2)+ R(N-2))
      Z22=RTHI2*( Z(N+1)- Z(N)- Z(N+2)+ Z(N-2))
      DH22=RTHI2*( DH(N+1)-DH(N)-DH(N+2)+DH(N-2))
      DZ22=RTHI2*( DZ(N+1)-DZ(N)-DZ(N+2)+DZ(N-2))
      20 RETURN
      END
      SUBROUTINE STRESS(MD,KP)
      C      ALL 1000 PLT
C      CONSTITUTIVE RELATION--LINEARLY ELASTIC, OR ELASTIC-(PERFECTLY PLASTIC OR -STRAIN HARDENING). OPTIONAL STRAIN RATE DEPENDENCE
      NEND
      KEND
      SS11=0.0
      SS22=0.0
      LMMK=0
      KN=(K-1)*NSSL
      ZFTAK=2.0*ZETA(K)
      G11 =A11-ZFTAK+H11
      G22 =A22-ZFTAK+H22
      FG=G11+G22
      SG=SQRT(FG)
      KG=1.0/DG
      GH11 = KG * G11
      GH22 = KG * G22
      DFPS11=GH11*DEPS1(K)
      DFPS22=GH22*DEPS2(K)
      D1111 = GH11*(G11 + G22) + GH22*(G11 + G22)
      GFAU      7
      GFAU      8
      GFAU      9
      GFAU      10
      GFAU      11
      GFAU      12
      GFAU      13
      GFAU      14
      GFAU      15
      GFAU      16
      GFAU      17
      GFAU      18
      GFAU      19
      GFAU      20
      GFAU      21
      GFAU      22
      GFAU      23
      GFAU      24
      GFAU      25
      GFAU      26
      GFAU      27
      GFAU      28
      GFAU      29
      GFAU      30
      GFAU      31
      GFAU      32
      GFAU      33
      GFAU      34
      GFAU      35
      GFAU      36
      GFAU      37
      GFAU      38
      GFAU      39
      GFAU      40
      GFAU      41
      GFAU      42
      GFAU      43
      GFAU      44
      GFAU      45
      GFAU      46
      GFAU      47
      STRESS      2
      TA1      -
      STRESS      4
      STRESS      5
      STRESS      6
      STRESS      7
      STRESS      8
      STRESS      9
      STRESS      10
      STRESS      11
      STRESS      12
      STRESS      13
      STRESS      14
      STRESS      15
      STRESS      16
      STRESS      17
      STRESS      18
      STRESS      19
      STRESS      20
      STRESS      21
      STRESS      22

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101 IFEPS2(K)=SIG1(N,KJ)+SIG2(N,KJ)*DSIG11*SIG622+SIG11*SIG622*
2 SIG111+SIG622L+SIG11C+SIG622D,AA,R,PHT
104 FORMAT(10X,7HFSI1 =E15.8,3X,7HEPSI2 =F15.8,3X,7HEPSI1 =E15.8,
1 3X,7HFSI2 =F15.8,3X,7HDFPSI1 =F15.8,3X,7HDFPSI2 =F15.8,3X,
2 7H SIG1 =F15.8,3X,7H SIG2 =F15.8,3X,7HDSIG11=F15.8,3X,
3 7HDSIG22=F15.8,3X,7HSIG11 =F15.8,3X,7HSIG22 =F15.8,3X,
4 7HSIG11 =F15.8,3X,7HSIG22L=F15.8,3X,7HSIG11L=F15.8,3X,
5 7HSIG22L=F15.8,3X,7HSIG11L=F15.8,3X,7HSIG22L=F15.8,3X,
6 7HSIG11L=F15.8,3X,7HSIG22L=F15.8,3X,7HSIG11L=F15.8,3X,
7 7HSIG22L=F15.8,3X,7HSIG11L=F15.8,3X,7HSIG22L=F15.8,3X,
8 NC3DP(MN30)=ACYCLE
CALL PDATA (2)
CALL PDATA (3)
CALL PDATA (4)
STOP! STRESS
102 SS11=SS11+SIG11*WT(J)
SS22=SS22+SIG22*WT(J)
IF(IP.EQ.2)GOTO 122
      IP#1 DENOTES MESH POINTS. IP#2 DENOTES MESHES
SIG1(N,KJ)=GH11*SIG11
SIG2(N,KJ)=GH22*SIG22
GOTO 803
122 SIG1M(N,KJ)=GR11*SIG11
SIG2M(N,KJ)=GR22*SIG22
803 C(NTINIE
IF(N,FE,M1H,0R,N,FC,M2H)SHG=0.5*SHG
IF(TGAUSS .EQ. 1) SFG = K(K)*SHG
IF(IP.EQ.1)STHEN=STHEN+((SS11+SS22)**2-(1.+FNU)*2.*SS11*SS22)*SRC
SS1MN(N)=GH11*SS11
SS2MN(K)=GR22*SS22
IF(IP.EQ.1)LHAT(N,K)=LMNK
IF(IP.EQ.2)LHATHM(N,K)=LMNK
RETURN
END
SUBROUTINE HMSTRS(ND,KD)
      *CALL MATN HMEF
C      STRESS FOR BEAMS
C      CONSTITUTIVE RELATION--LINEARLY ELASTIC, OR ELASTIC-(-)PERFECTLY
C      PLASTIC OR -STRAIN HANZING). OPTIONAL STRAIN HATE DEPENDENCE
      KEND
      KKD
      SS22=0.0
      LMNK=0
      L=0
      KF=(Y-1)*MSFL
      F22 = A22+2.*V22*ETM(K)*P*Z
      IF(IP.EQ.1)SFG = SQRT(G22)
      G22 = 1.0/G22
      DFEPS22=GR22*DFEPS2(K)
      DSIG22=F01*EPS22
      DO 803 J=1,MSFL
      KJ=KF+J
      IP#1 DENOTES MESH POINTS. IP#2 DENOTES MESHES
      IF(IP.EQ.1)SIG22I=G22*SIG2(N,KJ)
      IF(IP.EQ.2)SIG22I=G22*SIG2M(N,KJ)
      SIGYZ=SIG22(J)
      IF((SR,GT,0)*SIGYZ=SIGYZ*(1.+(AHS(DFEPS22)/(DELTAT*DSH(J)))*PSH(J)))*PSH(J))
      SIG22=SIG22I + SIG22
      IF((SR,LT,0)*G22 102
      IF(AHS(SIG22),GT,SIGYZ)=1.0*INT(4.0*((4*KC(SIG22)-SI(Y2))/SIGYZ))
      IF(L,GT,LMNK)LMNK=1
      IF(SIG22 .LT. -SIGYZ) SIG22=-SIGYZ
      IF(SIG22 .GT. SIGYZ) SIG22 = SIGYZ
102 SS22=SS22+SIG22*WT(J)
      IF((L .LT. 1./SI(SIG22))=0.022*SLK)

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1 IF(IF .EQ. 2)SIH2M(K,K)=CH22*SIH22
2 CONTINUE
3 IF(IF .EQ. 1)R=CH22*SIH22*SHG=L*SHG
4 IF(IF .EQ. 1)STLFA=STLFA+3*(455/7292)*A(K)
5 SIH2M(K)=CH22*SS22
6 IF(IF .EQ. 1)IAT(K,K)=LMNK
7 IF(IF .EQ. 2)LMATH(K,K)=LMNK
8 RETURN
9 END
10 SUBROUTINE RESULT(IND)
11 *CALL MAIN HERE
12 THE CALCULATION OF THE STRESS AND MOMENT RESULTANTS FOLLOW
13 IFCF1=1.2*0H3. IFCF2=1.2*3.0H4. H/2A/74 1/7/76
14 NEND
15 TPA=TA*SHA
16 IF(IF .GT. 0)GOTO 7
17 IF(IGAUSS .EQ. 1)GOTO 9
18 SUMA11 = SS1MN(1)
19 SUMA22 = SS2MN(1)
20 SUMR11 = SS1MN(1)*ZFTA(1)
21 SUMR22 = SS2MN(1)*ZFTA(1)
22 SUMC11 = SS1MN(1)*ZETASC(1)
23 SUMC22 = SS2MN(1)*ZETASC(1)
24 IF(LAYER .EQ. 1)GOTO 5
25 DO 6 K=2,LAYER
26 SUMA11 = SUMA11 + SS1MN(K)
27 SUMA22 = SUMA22 + SS2MN(K)
28 SUMR11 = SUMR11 + SS1MN(K)*ZETA(K)
29 SUMR22 = SUMR22 + SS2MN(K)*ZETA(K)
30 SUMC11 = SUMC11 + SS1MN(K)*ZETASC(K)
31 SUMC22 = SUMC22 + SS2MN(K)*ZETASC(K)
32 CONTINUE
33 CONTINUE
34 C1=SUMA11-RT*SUMR11
35 Q22=SUMA22-RT*SUMR22
36 F11=SUMR11-(RT+RM11)*SUMC11
37 F22=SUMR22-(RT+RM22)*SUMC22
38 IF(IF .EQ. 2)GOTO 17
39      IP=1 DENOTES MESH POINTS. IP=2 DENOTES MESHFS
40      DIFFERENCES FOR FM CENTRAL AT MESH POINTS. FOR FN IN MESHFS
41      IF(N .EQ. NTH .AND. IFCF1 .EQ. 1)F22=0.0
42      IF(N .EQ. NTH .AND. IFCF2 .EQ. 3)F22=0.0
43      IF(N .NE. NN)GOTO 16
44      NEAR. ASSUME IFCF2 =4. COMPUTE ALL RESULTANTS.
45      FM11H(N)=TR+F11*SNH(N)
46      FM22L(N)=0.0
47      FM22K(N)=0.0
48      FA11H(N)=TR+F11*FH(N)
49      AT FHFF END. FM2J(N)=(-1*M22/RTFA)*KJ
50      IF(SNH(N-1) .NE. 0.0)F22=FM22H(N-1)/SNH(N-1)
51      IF(SNH(N-1) .EQ. 0.0)F22=FM22K(N-1)/SNK(N-1)
52      IF(SNH(N-2) .NE. 0.0)F11=FM22H(N-2)/SNH(N-2)
53      IF(SNH(N-2) .EQ. 0.0)F11=FM22K(N-2)/SNK(N-2)
54      F22=RTD2*(4.0*F22-F11)
55      FM2H(N)=F22*SNH(N)
56      FM2K(N)=F22*SNK(N)
57      GOTO 1H
58 CONTINUE
59      FM11H(N)=TR+F11*SNH(N)
60      FM11K(N)=TR+F11*SNK(N) IS NOT USED
61      FM22H(N)=TR+F22*SNH(N)
62      FM22K(N)=TR+F22*SNK(N)
63      GOTO 1H
64      CONTINUE
65      FM11H(N)=TR+F11*SNH(N)
66      FM11K(N)=TR+F11*SNK(N)
67      FM22H(N)=TR+F22*SNH(N)
68      FM22K(N)=TR+F22*SNK(N)
69      GOTO 1H
70      CONTINUE
71      FM11H(N)=TR+F11*SNH(N)
72      FM11K(N)=TR+F11*SNK(N)
73      FM22H(N)=TR+F22*SNH(N)
74      FM22K(N)=TR+F22*SNK(N)
75      GOTO 1H

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17 CONTINUE
CSM2=CS211*F11+CS222*F22
FM1TH(N)=TR*G1*(0.5*(F(N)+F(N+1))
C FM1TH(N) ADD FM112(N) ZEE =LT 15E11 IF [S=2,
FM2P(N)=T*(C22*H2+SNR(N)*CSM2)
FM2K(N)=TH*(C22*Z2+SNK(N)*CSM2)
18 CONTINUE
RETURN
7 SUMA22=0.0
SUMH22=0.0
SUMC22=0.0
DO 8 K=1,LAYER
SUMA22 = SUMA22+w(K)*SS2MN(K)
SUMH22 = SUMH22 + SS2MK(K)*WZFTA(K)
SUMC22 = SUMC22 + SS2MC(K)*WZETSO(K)
K CONTINUE
GOTO 5
9 SUMA11 = 0.0
SUMH11 = 0.0
SUMC11 = 0.0
DO 10 K=1,LAYER
SUMA11 = SUMA11 + SS1MN(K)*W(K)
SUMH11 = SUMH11 + SS1MK(K)*WZFTA(K)
SUMC11 = SUMC11 + SS1MC(K)*WZETSO(K)
10 CONTINUE
GOTO 7
END
SUBROUTINE MOTION
*CALL MAIN HERE
IF(LOAD .EQ. 0)GOTO 30
FNS=0.0
CALL PWORK
C
TPCF1=1.2*CH3. THCF1=1.2*0.04. A/2B/74 1/7/76
30 DO 130 N=N1V,N2V
IF(N .NE. NN)GOTO 31
C SPECIAL CODING FOR N=NN (A FREE END)
VR=HD22*(2.0*FM22H(N)-5.0*FM22H(N-1)+4.0*FM22H(N-2)-FM22H(N-3))
1 +2.0*FTD2M*(FM2H(N)-FM2H(N-1))
VZ=HD22*(2.0*FM22K(N)-5.0*FM22K(N-1)+4.0*FM22K(N-2)-FM22K(N-3))
1 +2.0*FTD2M*(FM2K(N)-FM2K(N-1))
IF(TS .EQ. 1)VR=VR-FN11H(N)-FM1TH(N)
GOTO 34
31 CONTINUE
C DIFFERENCES FOR FN (ENTERED AT MESH POINTS, FOR FN IN MESHES
VR=HD22*(FM22F(N-1)-2.0*FM22F(N)+FM22F(N+1))
1 +FTD2M*(FN2H(N)-FN2H(N-1))
VZ=HD22*(FM22K(N-1)-2.0*FM22K(N)+FM22K(N+1))
1 +FTD2M*(FN2K(N)-FN2K(N-1))
IF(TS .EQ. 1)VR=VR-FN11H(N)-0.5*(FN1TH(N-1)+FN1TH(N))
35 CONTINUE
IF(LCAN.EQ.0)GOTO 50
VR=VR-SNP(N)*P(N)
VZ=VZ-SNK(N)*P(N)
50 DFTD2H(N)
D2I=D2(N)
DFS=0.1*VR*TFMP(N)
DZS=0.1*VZ*TFMP(N)
IF(TDAMP.EQ.0.0)GOTO 115
C VISCOSUS DAMPING
C S=1-1.25*DT/1.001
DZS=0.75-(DZS+C21)*C1

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115 DR(N)=0.05
116 DZ(N)=1.75
130 CONTINUE
    CALL RGRAD1
    CALL KINET
    IF(LOAD) 45,75,45
45 CALL PTERM
    EN=0.5*(AR*(EN5+ENH))
    EN=EN5
    IF(IRCFL .EQ. 2)EN=2.0*EN
    IF(IRCFL .EQ. 2)EN=2.0*EN
    TNHG=TNHG+EN
75 PLAST=TNHG-CINET-STEN-TDAMH
C
    IF(ICYCLE .NE. NCYCL(MNN)) GOTO 140
    WRITE(6,99991) NCYCLE,TIME,CINET,STEN,PLAST,TNHG
    MNN=MNN+1
140 RETURN
C
99991 FORMAT(//10H TIME STEP,15.3X,5HTIME=,F16.8,3X,HMKINFTIC=,E15,F+3X,
     1HPLASTIC=,E15,H+3X,AMPLASTIC=,E15,H/14H TOTAL ENERGY=,E15.8)
    EFO
    SUBROUTINE WTAPE(KEY)
C      WTAPE HAS BEEN DELETED.
    STOP! WTAPE DELETED!
    EFO
    SUBROUTINE STRAIN
        *CALL MAIN HERE
C
C      PRINT STRAINS ON INNER OR OUTER FACES
C
C      DIMENSION EPSANG(H),EPSANG6(H)
C          ARRAYS FOR EXTREME STRAINS --- IN STRAIN
C      DIMENSION EPC(4),EPSXT(H),NEPSXT(H),LEPSX(H)
C      DATA(LEPSX(L),L=1+H)/HFFPSU1MAX,HFFPSU2MAX,HFFPSL1MAX,HFFPSL2MAX,
C          1           HFFPSU1MIN,HFFPSU2MIN,HFFPSL1MIN,HFFPSL2MIN/
C      DATA PI/3.141592653589743/
C      IF(ICYCLE .GT. 0)GOTO 25
C
----- INITIAL ENTRY -----
DO 20 I=1,NSTRN
J1=NT1(I)
J2=NT2(I)
J3=M3(I)
A11 = DA1(J1) + UN1(I)*(DA1(J2)-DA1(J1))
R11 = RH1(J1) + UN1(I)*(RH1(J2)-RH1(J1))
R22 = RH2(J1) + UN1(I)*(RH2(J2)-RH2(J1))
A22 = DA2(J3) + UN3(I)*(DA2(J3+1)-DA2(J3))
2H = 7L
IF(NFTAG(J) .EQ. 1)ZH=7L
G11 = A11 - 2.0*ZH*H1
G22 = A22 - 2.0*ZH*H2
25 G111(I)=1.0/G11
G122(I)=1.0/G22
ANGFL=ANGLF(I)*PI/180.0
SA=SIN(ANGLF)
SH=COS(ANGLF)
ASA(I)=2.0*SA**2
HSA(I)=2.0*SH**2
ANGFL=ANGLR(I)*PI/180.0
SA=SIN(ANGLF)
SH=COS(ANGLF)
ASA(I)=2.0*SA**2
HSA(I)=2.0*SH**2
MOTION      38
MOTION      39
MOTION      40
MOTION      41
MOTION      42
MOTION      43
MOTION      44
MOTION      45
MOTION      46
MOTION      47
MOTION      48
MOTION      49
MOTION      50
MOTION      51
MOTION      52
MOTION      53
MOTION      54
MOTION      55
MOTION      56
MOTION      57
MOTION      58
MOTION      59
WTAPE       2
WTAPE       3
WTAPE       4
WTAPE       5
STRAIN      2
T/F         1
STRAIN      4
STRAIN      5
STRAIN      6
STRAIN      7
STRAIN      8
STRAIN      9
STRAIN      10
STRAIN      11
STRAIN      12
STRAIN      13
STRAIN      14
STRAIN      15
STRAIN      16
STRAIN      17
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STRAIN      21
STRAIN      22
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STRAIN      24
STRAIN      25
STRAIN      26
STRAIN      27
STRAIN      28
STRAIN      29
STRAIN      30
STRAIN      31
STRAIN      32
STRAIN      33
STRAIN      34
STRAIN      35
STRAIN      36
STRAIN      37
STRAIN      38

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20 CONTINUE
GOTO 71
C -----
24 LINK=1
C     FIND EXTREMES OF STRAIN AND MAXIMUM DEFLECTION
DO 401 L=1,4
EPSAT(1)=-100000.
401 FPSXT(L)=100000.
DEFXT=L,0
DO 905 N=N14,N24
EPC(1)=EPSU1(N)*GU1(N)  EPC(2)=EPSL2(N)*GL22(N)
EPC(3)=EPSL1(N)*GL11(N)  EPC(4)=EPSL2(N)*GL22(N)
DO 906 L=1,8
IF(L .GT. 4) GOTO 402
IF(FPSXT(L) .GE. EPC(L)) FPSXT(L)=N  GOTO 905
402 IF(FPSXT(L) .LE. EPC(L-4)) GOTO 905
FPSXT(L)=EPC(L-4)  N=FPSXT(L)
905 CONTINUE
DO 910 L=1,8
EPSXT(1)=SQR(1.0+2.0*FPSXT(L))-1.0
IF(L .GT. 4) GOTO 906
IF(FPSX(1) .GE. EPSXT(L)) GOTO 910
GOTO 907
406 IF(FPSX(L) .LE. EPSXT(L)) GOTO 910
407 FPSX(L)=FPSXT(L)  TFPSX(L)=TIME
NFPX(L)=NFPXT(L)
410 CONTINUE
DO 412 N=N1V,N2V
AY3MA=(PZ(N)-H(N))**2 + (ZZ(N)-Z(N))**2
IF(DEFXT .GE. AY3MA) GOTO 412
DEFXT=AY3MA  NDEFXT=N
412 CONTINUE
DEFXT = SQR(TDEFXT)
IF(TDEFXT .LE. NDEFXT) GOTO 413
DEFXT=TDEFXT  NDEFXT=TIME
413 CONTINUE
C     CHECK FOR SURFACE STRAIN PRINT
IF(NCYCLE .LT. NPRINT) GOTO 410
30 NPRINT=NPRINT+1
LINK=2
40 DO 46 I=1,NSTR
J1=NT1(I)
J2=NT2(I)
J3=NI3(I)
DA11 = DA1(J1) + DN1(I)*(DA1(J2)-DA1(J1))
DA12 = DA1(J1) + DN1(I)*(DA1(J2)-DA1(J1))
DH22 = DH2(J1) + DN1(I)*(DH2(J2)-DH2(J1))
DA22 = DA2(J3) + DA2(I)*(DA2(J3+1)-DA2(J3))
ZI=71
IF(INFTAG() .EQ. 1)/F=71
EPSH1 = ((A11 - ZR*DH11)*G11(I)
EPSH2 = ((A22 - ZH*DH22)*G122(I)
45 EPSS1(I)=SQR(1.0+2.0*EPSH1)-1.0
EPSS2(I)=SQR(1.0+2.0*EPSH2)-1.0
EPSAH(I)=SQR(1.0+PSA(I)*EPSH1+ASH(I)*EPSH2)-1.0
EPSAH(I)=SQR(1.0+PSA(I)*EPSH1+ASH(I)*EPSH2)-1.0
46 CONTINUE
C     COMPONENTS OF VECTOR DISPLACEMENT
D1=D1+(C1Z*D1*(NU1)+G11*D1*(H(NU2)))
D2=D2+(C2Z*D2*(NU1)+G11*D2*(H(NU2)))
TE=1/TIM  LS=1/LINK  TS=1
50 WAITF(6+NU1) NCYCLES+T/TE

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      DO 70 I=1,NSTAN
      ALFNSHIFTNFK
      TF(NFTAU(I),FI+1)GOTO 67
      ALFMELCHLTED
      67 WRITE(6,65)FTAU(2(I),FN(I),ALFM+FFS1(I)+EPSS2(I),
      1 ANGLF(I)+DPSANG(I),ANGLH(I)+EHSANG(I))
      70 CONTINUE
C       PRINTS OF FATHEN STRAINS AND MAXIMUM DEFLECTIONS
      WRITE(6,930)TIME,NCYCLE
      DO 921 L=1,8
      921 WRITE(6,932)LEPSX(L),FFSXT(L),NFFSXT(L)
      WRITE(6,931)NCOUNT,NCYCLE
      DO 922 L=1,8
      922 WRITE(6,433)LEPSX(L),EFSX(L),TEFSX(L),NEFSX(L)
      WRITE(6,461)TIME,NCYCLE,DEFEXT,NOFFXT
      WRITE(6,462)NCOUNT,NCYCLE,DEFEX,NOFFEX,TUFFEX
      GOTO 71
      930 FORMAT(//20X,'EXTENMF STRAINS AT TIME=',PF13.6,
      1 ', TIME CYCLE=',I5,'/4IX,1STRAIN',7X,'IN')
      931 FORMAT(20X,'EXTENMF STRAINS BETWEEN CYCLES',I5,', AND',I5,'/',
      1 '32X,1STRAIN',9X,', TIME',4X,'IN')
      932 FORMAT(25X,A10,F13.8,2X,I5)
      933 FORMAT(10X,A10,F13.8,2X,I5)
      941 FORMAT(10X,MAXIMUM DEFLECTION AT TIME=',PF13.6,', CYCLE=',
      1 I5,'/13X,1OFFLECTION',E15.6,4X,'(K = ',I2,')',)
      942 FORMAT(10X,MAXIMUM DEFLECTION BETWEEN CYCLES',I5,', AND',I5,'/13X,
      1 ,OFFLECTION=',PF15.6,E5,'(K = ',I2,')',,10X, 'TIME=',F15.6)
      71 RETURN
C
      60 FORMAT(//10W TIME STEP,15.3X,SHTIME=,E16.8//14X,1SFURFACE STRAINS
      1 ,37X,19HSTRAIN DAGE READING//10X,4FETA2+14X,1HN,5X,
      24HFACE,PX,THANGLE,0.10X,8HANGLE 40.6X,5HANGLE+18X,5HANGLE//)
      65 FORMAT(F15.2, F15.3, 4X ,A5,1X,2(2X,F15.8),2(2X,F6.2,2X,F15.8))
      FAD
      SUBROUTINE ROLND
C       READ DATA HERE
C       IHCF1=1+2*103, IHCF2=1+2*3+0*4, E/28/74 1/7/76
C       INSERT ONE EXTERNAL P AND Z, BOTH ENDS. 8/23/74
      DELTHR(N1H+1) = R(N1H)
      DELT2=Z(N1H+1) = Z(N1H)
      IF(IHCF1 .EQ. 1) GOTO 11
      IF(IHCF1 .EQ. 3) GOTO 13
      IF(IHCF1 .NE. 2) GOTO 25
      / (N1H-1)=R(N1H+1)
      Z(N1H-1)=Z(N1H)-DELTH2
      GOTO 20
      5 WRITE(6,999)IHCF1,IHCF2
      640 FORMAT(' FORCING IN HOURS. IHCF1,IHCF2 = ',2I5)
      STORE ROLND
      11 TEMPH=2.0*(DELT2+SHR(N1H)-DELT2*SNR(N1H))
      R(N1H+1) = R(N1H+1) + SHR(N1H)*TEMPH
      Z(N1H+1) = Z(N1H+1) - SHR(N1H)*TEMPH
      GOTO 20
      13 R(N1H+1) = R(N1H) - DELTH
      Z(N1H+1) = Z(N1H) - DELTZ
      20 DELTH = R(N2H+1) - R(N2H)
      DELTZ = Z(N2H+1) - Z(N2H)
      IF(IHCF2 .EQ. 1) GOTO 21
      IF(IHCF2 .NE. 2) GOTO 22
      IF(IHCF2 .EQ. 3) GOTO 23
      IF(IHCF2 .EQ. 4) GOTO 35
      GOTO 6
      21 TEMPH=2.0*(DELTH+SHR(N1H)-DELT2*SNR(N1H))
      22 TEMPH=2.0*(DELTZ+SHR(N1H)-Z(N2H)-SNR(N1H))
      23 TEMPH=2.0*(R(N2H+1)-R(N2H)-DELTH)
      24 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-DELTZ)
      25 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-SNR(N1H))
      26 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-SNR(N1H))
      27 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      28 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      29 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      30 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      35 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      36 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      37 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      38 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      39 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      40 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      41 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      42 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      43 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      44 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      45 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      46 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      47 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      48 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      49 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      50 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      51 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      52 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      53 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      54 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      55 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      56 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      57 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      58 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      59 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      60 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      61 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      62 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      63 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      64 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      65 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      66 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      67 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      68 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      69 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      70 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      71 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      72 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      73 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      74 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      75 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      76 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      77 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      78 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      79 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      80 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      81 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      82 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      83 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      84 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      85 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      86 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      87 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      88 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      89 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      90 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      91 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      92 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      93 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      94 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      95 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      96 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      97 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      98 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      99 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      100 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      101 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      102 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      103 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      104 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      105 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      106 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      107 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      108 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      109 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      110 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      111 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      112 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      113 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      114 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      115 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      116 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      117 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      118 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      119 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      120 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      121 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      122 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      123 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      124 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      125 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      126 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      127 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      128 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      129 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      130 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      131 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      132 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      133 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      134 TEMPH=2.0*(Z(N2H+1)-Z(N2H)-R(N2H)-DELTZ)
      135 TEMPH=2.0*(R(N2H+1)-R(N2H)-Z(N2H)-DELTH)
      2 TAF
      4 ECINER
      5 HCINDR
      6 HCINDR
      7 HCINDR
      8 HCINER
      9 HCINER
      10 HCINER
      11 HCINER
      12 HCINER
      13 HCINER
      14 HCINER
      15 HCINER
      16 HCINER
      17 HCINER
      18 HCINER
      19 HCINER
      20 HCINER
      21 HCINER
      22 HCINER
      23 HCINER
      24 HCINER
      25 HCINER
      26 HCINER
      27 HCINER
      28 HCINER
      29 HCINER
      30 HCINER

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E(N2H+1) = E(N2H-1) + SNK(N2H)*TEMHU
Z(N2H+1) = Z(N2H-1) - SNK(N2H)*TEMHU
GOTO 30
22 E(N2H+1) = E(N2H-1)
Z(N2H+1) = Z(N2H) - DELT2
GOTO 30
23 E(N2H+1) = E(N2H) - DELTH
Z(N2H+1) = Z(N2H) - DELT2
30 RETURN
END
SUBROUTINE ROUNDU
      *CALL MAIN HERE
C      THCF1=1.2.CH3, THCF2=1.2.3.0R4, R/28/74 1/7/76
C      INSERT DR AND DZ AT ENDS. ONE EXTERNAL POINT. R/23/74
IF(THCF1 .EQ. 1) GOTO 11
IF(THCF1 .EQ. 3) GOTO 13
IF(THCF1 .NE. 2) GOTO 5
DR(N1H+1)=DR(N1H+1)
DZ(N1H+1)=DZ(N1H+1)
DZ(N1H)=0.0
GOTO 20
5 WRITE(6,990)IPCF1,IPCF2
990 FORMAT(1X,ERR1 IN ROUNDU, IPCE1+THCF2 =? 215)
STOP! HOMEDIT
11 TEMPH=2.0*(SNK(N1H)*DZ(N1H+1)-SNK(N1H)*DR(N1H+1))
DR(N1H+1) = DR(N1H+1) + SNK(N1H)*TEMHU
DZ(N1H+1) = DZ(N1H+1) - SNK(N1H)*TEMHU
E(N1H)=0.0
DZ(N1H)=0.0
GOTO 20
13 DR(N1H+1) = -DR(N1H+1)
DZ(N1H+1) = -DZ(N1H+1)
DR(N1H)=0.0
DZ(N1H)=0.0
20 IF(IPCF2 .EQ. 1) GOTO 21
IF(IPCF2 .EQ. 2) GOTO 22
IF(IPCF2 .EQ. 3) GOTO 23
IF(IPCF2 .EQ. 4) GOTO 30
GOTO 5
21 TEMPH = 2.0*(SNK(N2H)*DZ(N2H+1) - SNK(N2H)*DR(N2H+1))
DR(N2H+1) = DR(N2H+1) + SNK(N2H)*TEMHU
DZ(N2H+1) = DZ(N2H+1) - SNK(N2H)*TEMHU
DR(N2H)=0.0
DZ(N2H)=0.0
GOTO 30
22 DR(N2H+1) = DR(N2H+1)
DZ(N2H+1) = -DZ(N2H+1)
DR(N2H)=0.0
DZ(N2H)=0.0
30 RETURN
END
SUBROUTINE SYNTHY
      *CALL MAIN HERE
IF(THCF2 .NE. 2) GOTO 50
FN22P(N2H+1)=FN22P(N2H+1)
FN22P(N2H+1)=-FN22P(N2H+1)
FN1TH(N2H) = FN1TH(N2H+1)
FN1TH(N2H+1) = FN1TH(N2H+1)
FN2K(N2H) = FN2K(N2H+1)

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```

50 IF(IIFCF1 .NE. 2)GOTO 60
FM2PF(N1H-1)= FM2PH(N1H+1)
FN2PK(N1H-1)= FN2PK(N1H+1)
FA1TH(N1H-1)= FA1TH(N1H)
FA2H(N1H-1)= -FN2H(N1H)
FN2K(N1H-1)= FN2K(N1H)
AD RETURN
FAT
SUBROUTINE AFINIT(N)
STOP "AFINIT DELETED"
END
SUBROUTINE KINFT
      *CALL MAIN HERE*
CINET=0.
DO 43 N=NIV,NPV
CN=1.0
IF(N .EQ. N1H)CN=0.5
IF(N .EQ. N2H)CN=0.5
CINET=CINET+(RH(N)**2*DZ(N)**2)/THM(N) *CN
P3 CONTINUE
IF(IIFCF1 .EQ. 2)CINET=2.0*CINET
IF(IIFCF2 .NE. 2)GOTO 9A
CINET=2.0*CINFT
PA CINES=CINES
CINES=0.5*CA*CINET
CTNFT=0.5*(CINES+CINFR)
RETURN
FAD
SUBROUTINE PWCRK
      *CALL MAIN HERE*
DO 20 N=NIV,NPV
CA=0.50
IF(N .EQ. N1H)CN=0.25
IF(N .EQ. N2H)CN=0.25
PWCRH(N)*SNR(A)+DZ(N)*SKR(N)
ENS=ENS-CN*DWRF(N)
20 CONTINUE
RETURN
FAD
SUBROUTINE DAMP
      *CALL MAIN HERE*
      CHECK FOR START OF DAMPING
IF(NCYCLE .LT. 10)RETURN
IF(NCYCLE .GT. 10)GOTO 1A
      MONITOR THE DAMPING
WHITE(6,101)NCYCLE,TIME
101 FORMAT(/' INITIATED DAMPING. NCYCLE = ',I6,I0X,'TIME = ',IHF15.7)
LCAN = 0
LPRESS = 0
LC IF N=1.0N
P(N)=0.0
15 CONTINUE
1A CONTINUE
IF(CINES-CINFR)20,20,60
20 TDAMP=TDAMP+CINFT
WHITE(6,102)NCYCLE,CINES,CINFR,CINFT,TDAMP,CINFR
102 FORMAT(/' STOPPING PROCEDURE NCYCLE,CINES,CINFR,CINFT,TDAMP,CINFR =
1 / I16,I0SF15.7)
IF(CINFT+CINFR .LT. 0.0001)TDAMP=GOTO 50
DO 30 N=1,NN
IF(CINES .LT. 0.0)
DZ(N)=0.0
30

```

	SYNTRY	10
	SYNTRY	11
	SYNTRY	12
	SYNTRY	13
	SYNTRY	14
	SYNTRY	15
	SYNTRY	16
	SYNTRY	17
	AFINIT	2
	AFINIT	3
	AFINIT	4
	KINFT	?
	TAF	1
	KINFT	4
	KINFT	5
	KINFT	6
	KINFT	7
	KINFT	8
	KINFT	9
	KINFT	10
	KINFT	11
	KINFT	12
	KINFT	13
	KINFT	14
	KINFT	15
	KINFT	16
	KINFT	17
	KINFT	18
	PWCRK	2
	TAF	1
	PWCRK	4
	PWCRK	5
	PWCRK	6
	PWCRK	7
	PWCRK	8
	PWCRK	9
	PWCRK	10
	PWCRK	11
	PWCRK	12
	DAMP	2
	TAF	1
	DAMP	4
	DAMP	5
	DAMP	6
	DAMP	7
	DAMP	8
	DAMP	9
	DAMP	10
	DAMP	11
	DAMP	12
	DAMP	13
	DAMP	14
	DAMP	15
	DAMP	16
	DAMP	17
	DAMP	18
	DAMP	19
	DAMP	20
	DAMP	21
	DAMP	22
	DAMP	23
	CALS	?
	DAMP	25

```

30 CONTINUE
CINFP=CINFT
CINFS = CINFR
(TINF2 = LINES
CIAFS=0.0
NCYCLE = NCYCLE + 1
TIME = TIME + DELTAT
CALL MOTION
IF(CINFS .LE. LINES)GOTO 39
CALL DESTEP
39 CALL PDATA(2)
40 TDAMP=TDAMP+C2*CINFS
45 RETURN
50 WRITE(6,100) NCYCLE
MAXC=NCYCLE
NC3DP(NN3D)=NCYCLE
CALL PDATA (2)
GOTO 45
100 FORMAT(1H1+10X,30HRUN SELF-TERMINATED TIME STEP,15)
END
SUBROUTINE DESTEP
      *CALL MAIN HERE
      DECREASE DELTAT (IT IS TOO LARGE FOR STOPPING PROCEDURE)
DELTAT=SGT((CIAFS1/CINFS)+DELTAT)
PSOLD=DELSQ
C1OLD=C1
IFLSQ=DELTA*#2
C2=2.0*DELTAT*DAMP/64#2
C1=C2/(4.0+C2)
(ELG=DELSQ/NSGOLD
DELS=DELR*(1.0-C1)/(1.0-C1OLD)
CINFS=CINFS*DELS*#2/DELG
CINFT=0.5*(CIAFS+CINFR)
PLAST=TNHG-CINFT-STHEN-TDAMP
DO 10 K=1,NN
TFM(N)=DELR*TFM(N)
DR(N)=DELS*DR(N)
RZ(N)=DELS*DZ(N)
10 CONTINUE
RETURN
END
SUBROUTINE PDATA(LINK)
      *CALL MAIN HERE
DIMENSION DAT(20)
PDATA SELCTS AND WRITES DATA ON TAPE(NPLOT) FOR THE WPSIL
PLOTTING PROGRAM

GOTO (10,40,50,60) OF TPK
10 NPLOT=1
II = 2*NSTRN + 1
      INITIATE PLOT FILE. RESTART CAPABILITIES DRAFTED.
15 WRITE(NPLOT)ETAN2,CN,NSTHN
WRITE(NPLOT)(ETAN2(I),FR(I),NFTAG(I),I=1,NSTHN)
NPTSN2H=A19+1
WRITE(NPLOT)NCYCLE,TIME,NHTS,(H(N),Z(N),N=1,H,N#D)

25 CONTINUE
IF (LOAD) = 30+30+35
30 CAT(5)=TNHG
CAT(4)=TNHG
CAT(7)=TNHG

```

```

      LAT(8)=TMHG
35  IFLAG=1
      WRITE(NPLOT) IFLAG
      WRITE(NPLOT) NCYCLE*(LAT(I)+I=1,I)
      GOTO 100

40  LAT(1)=TIME
      LAT(2)=E1
      LAT(3)=E2
      LAT(4)=INFT
      LAT(5)=STHEN+CINET
      LAT(6)=TMHG
      LAT(7)=LAT(6)+TOAMP
      J = 4
      DO 45 I=1,NSTRN
      LAT(J)=FPSS1(I)
      LAT(J+1)=FPSS2(I)
      J=J+2
45  CONTINUE
      IFLAG=1
      WRITE(NPLOT) IFLAG
      WRITE(NPLOT) NCYCLE*(LAT(I)+I=1,I)
      CHECK FOR 3D PLOT
      IF (NCYCLE .NE. NC3DP(NN3D)) GOTO 100
      NR3D=NN3D+1
      IFLAG=2
      WRITE(NPLOT) IFLAG
      NPTS=NPH-NIH+1
      WRITE(NPLOT) NCYCLE,TIME,NPTS,(H(N)+7(N)+N#H),N2H)
      GOTO 100

C   50 END FILE NPLOT & DELETED. MARKED RESTART POSITIONS FOR BRLESC
C   50 CONTINUE
      GOTO 100

C   60 IFLAG=000464
      WRITE(NPLOT) IFLAG
100  RETURN
END
SUBROUTINE PFESS
*CALL MAIN HERE
CONSTANT PHESSURE = P0.
DATA TPRESS/0/
1-(IPRESS .LT. 1.E-12) 2
IPRESS = 1
READ(5,100) P0
WRITE(6,105) P0
? P0 = NINV*NPV
? P(N)=P0
RETURN
100 FORMAT(F12.6)
105 FORMAT(1/36X,1,CONSTANT PRESSURE LOADING, P0=!,1HE15.6/)
END
SUBROUTINE INGEOM
SIGMAIS TO PROGRAM FROM INGEOM.
IFAN (IR>0).
      SLAB SYMMETRY (RADIUS = 0.0, IF=0).
      AXIAL SYMMETRY (RADIUS > 0.0, IF=0).
INGEOM FOR STRAIGHT BEAM OR SLAB SYMMETRIC FLAT SLAB
EVALUATE THE INITIAL GEOMETRY
*CALL MAIN HERE
SET PROGRAM TO USE GAUSSIAN INTEGRATION IN INGEOM.
IPRESS = 1

```

```

C      RADIUS = 0.0          IN(G)M   12
C  RADIUS =0.0 IS A SIGNAL TO USE SLAB SYMMETRY OR REAM    IN(G)M   13
C      READ(5,100) SLAHL,SLAHW,IR      IN(G)M   14
C      RREAD(5,100) SLAHL,SLAHW,IR      IN(G)M   15
C      DETA1 = 2.0*SLAHL      IN(G)M   16
C  ENERGY IS COMPUTED FOR A WIDTH OF DETA1.      IN(G)M   17
C  DETA1 TENTATIVELY SET AT 2*SLAHL.  IF IR > 0 DETA1 SET TO 1.0 IN START  IN(G)M   18
C      DETA2 = SLAHL/FLOAT(NMESH)      IN(G)M   19
C      EVALUATE(P(N)+Z(N),K=N1H+N2H)      IN(G)M   20
C      DO 10 N=N1H,N2H      IN(G)M   21
C      R(N) = 0.0      IN(G)M   22
C      Z(N)=FLOAT(N-N1H)*DETA2      IN(G)M   23
C  10 CONTINUE      IN(G)M   24
C      IF(IP .LE. 0) RETURN      IN(G)M   25
C      READ(5,96) ZU      IN(G)M   26
C      WRITE(6,97) SLAHL,ZU      IN(G)M   27
C  DO 12 K=1,LAYFR      IN(G)M   28
C      READ(5,46) WINTHK,PZFTAK,ZETA(K)      IN(G)M   29
C      W(K)=WINTHK+PZFTAK      IN(G)M   30
C      WRITE(6,44) K,WINTHK,PZFTAK,ZETA(K),W(K)      IN(G)M   31
C  12 CONTINUE      IN(G)M   32
C  97 FORMAT(/,1 IN(G)M  FCH REAM OF LENGTH*,E15.7/
C      *  DISTANCE OF CENTROID FROM TOP OF REAM IS*,E15.7/      IN(G)M   33
C  1 2X,1DIMENSIONS OF LAYER$/,      IN(G)M   34
C  2 PX,*K*,ZA,*WINTHK*, ZX,*DELHEIGHT*,HX,*ZFTA(K)*,PX,*AREAK*/      IN(G)M   35
C  3A FORMAT(3F10.2)      IN(G)M   36
C  3B FORMAT(I3,2X,4F15.7)      IN(G)M   37
C      RETURN      IN(G)M   38
C  100 FORMAT(2F10.4*I5)      IN(G)M   39
C      END      IN(G)M   40
C  SUBROUTINE INNORM      IN(G)M   41
C      *CALL MAIN HERE      IN(G)M   42
C      NEW CONDITIONS ON FIXED ENDS 12/12/73      IN(G)M   43
C      INNORM COMPUTES THE INITIAL NORMAL AT CLAMPED ENDS      IN(G)M   44
C      THIS VERSION ASSUMES A QUADRATIC THRU THE THREE END POINTS      IN(G)M   45
C      IF(IREFL .NE. 1) GOTO 5      IN(G)M   46
C      END1 N=N1H, IS CLAMPED      IN(G)M   47
C      R2=F(N1H+2)+4.0*R(N1H+1)-3.0*R(N1H)      IN(G)M   48
C      Z2=Z(N1H+2)+4.0*Z(N1H+1)-3.0*Z(N1H)      IN(G)M   49
C      T=SQRT(R2**2+Z2**2)+ SNR(N1H)=Z2/R SNK(N1H)=-R2/R      IN(G)M   50
C  5  IF(IREFL .NE. 1) GOTO 6      IN(G)M   51
C      END2 N=N2H, IS CLAMPED      IN(G)M   52
C      R2=Z(N2H-2)-4.0*Z(N2H-1)+3.0*Z(N2H)      IN(G)M   53
C      Z2=Z(N2H-2)-4.0*Z(N2H-1)+3.0*Z(N2H)      IN(G)M   54
C      T=SQRT(R2**2+Z2**2)+ SNR(N2H)=Z2/R SNK(N2H)=-R2/R      IN(G)M   55
C  6  RETURN      IN(G)M   56
C      END      IN(G)M   57

```

RD-A122 336

RPSL1D (A ONE-DIMENSIONAL VERSION OF REPSIL)(U) ARMY  
ARMAMENT RESEARCH AND DEVELOPMENT COMMAND ABERDEEN  
PROV. J D WORTMAN NOV 82 ARBRL-MR-03221

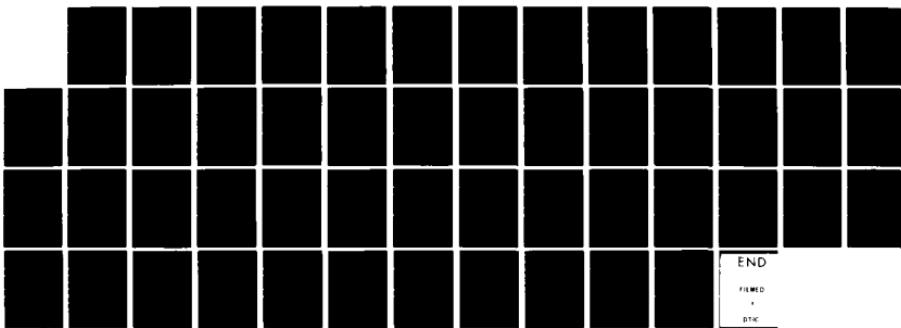
2/2

UNCLASSIFIED

SBI-AD-F300 132

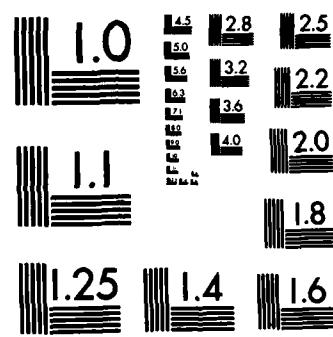
F/G 19/1

NL



END

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DTIC



MICROCOPY RESOLUTION TEST CHART  
NATIONAL BUREAU OF STANDARDS-1963-A

## APPENDIX D

### PLOTTING PROGRAM FOR RPSL1D

The plotting program is an independent program. The RPSL1D program stores data for plotting through unit 3 which is equivalenced to file BLOT on the PROGRAM card. The plotting program, which we have catalogued in file RPSL1DPLOT, reads data from unit 3 also equivalenced to file BLOT. For some runs it may be advantageous to catalogue BLOT and repeat the plotting program to reproduce plots that were unsatisfactory because of scaling (input card) or system problems.

This program uses the BRL plotting subroutines described in ARDC TR6 (BRLESC FORTRAN PLOTTING SUBROUTINES, Monte W. Coleman, John V. Lanahan, July 1970) as amended for CDC by local publication SPB-6-78, May 2, 1978. Conversion to SCOOP, the plotting system used in Reference 2, or another plotting system, would not be difficult.

The main program, RP1PLT, reads binary data from unit 3 (equated to BLOT) and controls the program flow. If the input variable IFLAG is 1, an array of data for one time point is read and stored. If IFLAG equals 2, different data is read and PLOT3D is called. If IFLAG equals 99999, or certain abnormal conditions occur, the program calls subroutine GRAPH.

Subroutine PLOT3D reads a control card on the initial entry containing DEFLM, SOFC, and SF with FORMAT (3E10.3), and sets up scaling for isometric plots of the "center line deflection profile" of Reference 2. This is a plot of the initial position of the reference curve, and the current position of the curve with the difference magnified by DEFLM. If SF is greater than zero, it is the reciprocal of the scale factor between the internal length measure and inches on the plotting surface. If the input SF is not greater than zero, the program assigns an SF that attempts to scale the plot of the initial curve into an SOFC by SOFC inch square which also includes the origin.

Subroutine GRAPH plots displacement increments DR and DZ vs. time at point (ETAD1, ETAD2) and the energy balance plot: time vs. kinetic energy, kinetic energy plus strain energy, total energy, and total energy less damping work. Subroutine GRAPH calls subroutine STRAIN to produce NSTRN plots of strains in the coordinate directions vs. time at points prescribed on input cards 13. If the PLOTP option is included, GRAPH also produces the NNPE prescribed plots of P(N) vs. time.

All the COMMON variables are included in COMDECK MAIN. The longer arrays are put in LEVEL 2 (a special version of FIXSCA was inserted to use them); if more than 3000 time cycles are recorded, these arrays and MAXC must be increased to plot them.

The following list is the COMPILE file image of RPSL1DPLOT formed through UPDATE. This listing gives the correct UPDATE card identifiers.

```

PROGRAM WPIPLT(INPLOT,OUTPUT,HLOT,TAPE13,TAPE5=INPUT,TAPF6=OUTPUT, WPIPLT      2
1  TAPE3=HLOT)          WPIPLT      3
C   PLOTTING FOR I-C PEPSTL.          WPIPLT      4
C   WPPSID PLOTTING PACKAGE (CALCOMP PLOTTING, SPP-E-7B)          WPIPLT      5
C   IF NCYCL>MAXC, INCREASE MAXC AND ARRAYS IN PARAM AND PLOT.          WPIPLT      6
C   PLOTTING PIN). 9/30/75 (PN(3002.4))          WPIPLT      7
C   WPIPLT      8
C
COMMON ETAG2(6),PN(6),NETAG(6),NSTHN=MAXC          MAIN      2
COMMON RR(103),ZZ(103),Y(103),Z(103)          MAIN      3
COMMON DAT(20),NCYCLE,TIME,          ETAD2= 0, 0N=NCYCL          MAIN      4
COMMON NNPE,NPE(9),PDAT(9)          MAIN      5
COMMON /PARRAY/
1  TIM( 3002), U2( 3002), U3( 3002), CIN( 3002), STC( 3002), MAIN      7
2  TNR( 3002),DAMPLT( 3002),EPSS1(1P012),EPSS2(1A012)          MAIN      8
COMMON/HLOUTP/PD( 3002.9)
LEVEL2,TIM,PC
MAXC = 3002
NNPF = 0
NCYCL=0
NPLOT=3
REWIND NPLOT
C
READ(NPLOT) ETAD2,0N,NSTRN          WPIPLT      10
READ(NPLOT) (ETAG2(I),PN(I),NETAG(I),I=1,NSTHN)          WPIPLT      11
READ(NPLOT)NNPE,(NPE(I),I=1,NNPE)          ACTIVATE FOR PLOTP          WPIPLT      12
READ(NPLOT) NCYCLE,TIME,N1,(RR(N),ZZ(N),N=1,N1)          WPIPLT      13
II=2*NSTRN+8          WPIPLT      14
WPIPLT      15
C
SAVE INITIAL SHAPE NEEDED FOR DEFLECTION MAGNIFICATION IN PLOT3D          WPIPLT      16
DO 5 N=1,N1          WPIPLT      17
Y(N)=ZZ(N)          WPIPLT      18
Z(N)=RR(N)          WPIPLT      19
5 CONTINUE          WPIPLT      20
CALL PLOT3D(N1)          WPIPLT      21
C
10 READ(INPLOT) IFLAG
IF(EOF(INPLOT),NE, 0) GOTO 28
IF(IFLAG, EQ, 44999) GOTO 30
IF(IFLAG, EQ, 1)GOT 20
IF(IFLAG, EQ, 2)GOTO 25
WRITE(6,911)IFLAG,NCYCL,NCYCLE
911 FORMAT(//I RAD SIGNAL FROM TAPE. IFLAG,NCYCL,NCYCLE=*,3110)
C
      IF RUN FAILS, GET PARTIAL PLUT.
GOTO 28
C
C
20 READ (INPLOT) NCYCLE,(DAT(I),I=1,IT)
C   1 ,(PDAT(J),J=1,NNPF)          ACTIVATE FOR PLOTP          WPIPLT      39
IF(EOF(INPLOT),NE, 0) GOTO 28
NCYCL=NCYCL+1
IF(NCYCL,GT, MAXC)GOTO 28
TIM(NCYCL)=DAT(1)
U3(NCYCL)=DAT(2)
U2(NCYCL)=DAT(3)
CIN (NCYCL)=DAT(5)
STC (NCYCL)=DAT(6)
TNR (NCYCL)=DAT(7)
DAMPLT(NCYCL)=DAT(8)
DO 22 I=9,II+2
J=NCYCL + MAXC*(I-9)/2
EPSS1(J)=DAT(I)
EPSS2(J)=DAT(I+1)
22 CONTINUE          WPIPLT      40
WPIPLT      41
WPIPLT      42
WPIPLT      43
WPIPLT      44
WPIPLT      45
WPIPLT      46
WPIPLT      47
WPIPLT      48
WPIPLT      49
WPIPLT      50
WPIPLT      51
WPIPLT      52
WPIPLT      53
WPIPLT      54
WPIPLT      55

```

```

      IF(NNPF .EQ. 0)GOTO 10
      ED 23 I=1.NNPF
  23 PD(NCYCL,I) = PDA(I)
      GOTO 10
  25 READ(NPLOT) NCYCL,TIME,N1,(WR(N)+ZZ(N),N=1,N1)
      IF(FOF(NPLOT) .NE. 0) GOTO 26
      CALL PLT3D(N1)
      GOTO 10
C
  26 WRITE(6,100) MAXC
  100 FORMAT(//20H ERROR NCYCL > MAXC=,I5/)
C      THE LAST SET OF DATA IS NO GOOD. PLOT THE REST.
      NCYCL = NCYCL - 1
C
  30 CALL GRAPH
      CALL PLTPGE
      CALL EXIT
      END
      SUBROUTINE PLT3D(I2)
      COMMON FTAG2(6),PN(6),NETAG(6),NSTRN,MAXC
      COMMON RP(103),ZZ(103),Y(103),Z(103)
      COMMON DAT(20),NCYCL,TIME,          ETAD2,  ON,NCYCL
      COMMON NNPF,NPE(9),PDA(9)
      COMMON /PARRAY/
      1 TIM( 3002), U2( 3002), CIN( 3002), STC( 3002), MAIN    7
      ? TNW( 3002),DAMPLT( 3002),EPSS1(1R012),EPSS2(1R012)   MAIN    8
      COMMON/PLOTP/PD( 3002+9)
      LEVEL2,TIM,PD
      DIMENSION X1(103),X2(103),      HEAD2(3),HEAD3(2)
      DIMENSION LAHEL(4)
      DATA LABEL/'WORTMAN','B309','X979','P1PLT'
      DATA T/0/
      DATA (HEAD2(IK),IK=1,3)/10HDEFLECTION,10H MAGNIFTER,1H>/
      DATA (HEAD3(IK),IK=1,2)/10HMICROCON,3HDS>/
      IF(I.EQ.0)GOTO 10
      I=I+1
      YPARN=YRAFN+10.0
      IF(I.LF.3)GOTC 30
      CALL PLTPGE
      GOTO 20
  10 XPAGE=12.0
      READ(5,11) DEFLN,S0FC,SF
  11 FORMAT(3F10.3,1F)
C      ----- SCALE FACTOR FOR 3D.PLOT -----
      YMAX=Y(1)
      YM1N=Y(1)
      ZMAX=Z(1)
      ZMIN=Z(1)
      DO 12 I=1,T2
      YMAX=AMAX1(Y(N),YMAX)
      YM1N=AMIN1(Y(N),YM1N)
      ZMAX=AMAX1(Z(N),ZMAX)
      ZMIN=AMIN1(Z(N),ZMIN)
  12 CONTINUE
      IF(SF .NE. 0.0)GOTO 13
      YS=(YMAX-YM1N)/S0FC
      ZS=(ZMAX-ZMIN)/S0FC
      SF=AMAX1(YS,ZS)
      IF(SF .GT. 0.0 .AND. SF .LT. 0.99)GOTO 13
      SF=AINT(SF)
      IF(SF .LT. 1.0) SF=1.0
  13 CALL PLTSCA(XPAGE,20,.1,17,(AREF))
      CALL PLTSCA (1.0+0.0,0.0+0.0+1.0+1.0)

```



```

1 TIM( 3002), U2( 3002), U3( 3002), CIN( 3002), STC( 3002), MAIN      7
2 TNR( 3002), DAMPLT( 3002), EPSS1(18012), EPSS2(18012)                8
COMMON/PLOTP/PO( 3002,G)                                                 9
LEVEL2,TIM,PC
DIMENSION      SYM1(4),SYM2(2),      ETA2(3),SY4(3)                      10
DATA(SYM1(I),I=1,4)/10HCOMPONENT .10HUF VECTOR .10HFISHLACMF .3HNT        11
1>/.(SYM2(I),I=1,2)/10HTIME (.1ICR.10HUSECONDS)>/
DATA(ETA2(I),I=1,3)/10HETA2 = .10H   (.5H )>/
2 (SY4(I),I=1,2)/10H    ENERGY.9H BALANCE>/                           12
C
CALL PLTPGF
XBAR=3.0
YFAR=2.0
XL=7.9
YL=6.4
N=NCYCL
XS=1.0E6
C
----- GRAPH ONE -----
CALL FIXSCA (TIM(1),N,XL,XS1,XMIN1,XMAX1,DX1)                         13
CALL FIXSCA (U3(1),N,YL,YS1,YMIN1,YMAX1,DY1)                           14
CALL CONSCA (U2(1),N,YL,YS1,YMIN1,YMAX1,DY1)                           15
CALL PLTSCA (XBAR,YFAR,XMIN1,YMIN1,XS1,YS1)                            16
CALL PLTAXS (DX1,DY1,XMIN1,XMAX1,YMIN1,YMAX1+4)                         17
TFM1=XMAX1+DX1/3.0
CALL PLTDOT2 (1.0,TIM(1),U2(1),N,0)
U2N = U2(N)
CALL PLTSYM (.1,3H0Z>,0.0,    TEM1 +U2(N))
CALL PLTDOT2 (1.0,TIM(1),U3(1),N,0)
U3N = U3(N)
CALL PLTSYM (.1,3H0R>,0.0,    TEM1 +U3,N)
CALL LARFLA (DX1,DY1,XMIN1,XMAX1,YMIN1,YMAX1,XS,1.0)                   18
CALL PLTSCA (XBAR,YFAR,0.0,0.0,1.0,1.0)                                 19
CALL PLTSYM (.1,SYM2(1),0.0,    3.0,-0.6)                                20
CALL PLTSYM (.1,SYM1(1),90.0,   -1.2+1.4)                               21
CALL PLTSYM (.1,9HLOCATION>,0.0,   3.0,-1.0)                             22
CALL PLTSYM (.3,2H>,0.0,     3.8,-1.1)                                23
CALL PLTSYM (.3,2H>,0.0,     6.6,-1.1)                                24
CALL PLTSYM (.1,ETA2(1),0.0,   4.1,-1.1)                               25
FNCODE (A,50,STEP)ETAD?
CALL PLTSYM (.1,STFP,0.0,     4.8,-1.1)                                26
FNCODE (A,50,STEP)ON
CALL PLTSYM (.1,STFP,0.0,     5.7,-1.1)                                27
C
----- GRAPH TWO -----
YFAR=12.0
CALL PLTSCA(XBAR,YFAR,0.0,0.0,0.0,1.0,1.0)                            28
CALL PLTSYM(.1,SY4(1),90.0,   -1.0,1.5)                                29
CALL PLTSYM(.1,SYM2(1),0.0,   3.0,-0.6)                                30
CALL FIXSCA (CIN(1),N,YL,YS4,YMIN4,YMAX4,DY4)                         31
CALL CONSCA (STC(1),N,YL,YS4,YMIN4,YMAX4,DY4)                         32
CALL CONSCA (TNR(1),N,YL,YS4,YMIN4,YMAX4,DY4)                         33
CALL CONSCA (DAMPLT(1),N,YL,YS4,YMIN4,YMAX4,DY4)                        34
CALL PLTSCA (XBAR,YFAR,XMIN1,YMIN4,XS1,YS4)                           35
CALL PLTAXS (DX1,DY4,XMIN1,XMAX1,YMIN4,YMAX4,4)                         36
CALL LARFLA (DX1,DY4,XMIN1,XMAX1,YMIN4,YMAX4,XS,1.0)                  37
CALL PLTDOT2 (1.0,TIM(1),CIN(1),N,0)
CALL PLTDOT2 (1.0,TIM(1),STC(1),N,0)
CALL PLTDOT2 (1.0,TIM(1),DAMPLT(1),N,0)
CALL PLTDOT2 (1.0,TIM(1),TNR(1),N,0)
C
----- STRAIN PLOTS -----
DO 100 I=1,ASTRN
J=1+MAXC*(I-1)
CALL STRAIN (TIM,EPSS1,EPSS2,          ETAGC(I),ETAG(I)+N,0)           38
100 CONTINUE

```

```

C          PLOTTING F(N).  9/30/75
IF(NMPP .EQ.0)GOTO 35
YPAR = 40.0
DO 33 I=1,NMPP
YPAR = YPAR + 10.0
IF(YPAR .GT. 30.0)CALL PLTPGF
IF(YPAR .GT. 30.0)YPAR = 2.0
FACODE(7,30,STFP)NPF(I)
30 FORMAT(2MP(.I3.2H)>)
CALL PLTSCA(XHAR,YHAR,0.0+0.0+1.0+1.0)
CALL PLTSYM(.1+SYM2(1)+0.0+ 3.0+-0.6)
CALL PLTSYM(.1+STEP+90.0+ -1.2+2.5)
CALL FIXSCA(PD(1,I),N,YL+YS1+YMIN1+YMAX1,DY1)
CALL PLTSCA(XHAR,YPAR,XMIN1,YMIN1+XS1+YS1)
CALL PLTAXS(DX1,DY1+XMIN1+XMAX1,YMIN1+YMAX1+4)
CALL LAFELA(DX1,DY1+XMIN1+XMAX1,YMIN1+YMAX1+XS+1,0)
CALL PLTOT2(1.0,TIM(1),PD(1,I),N,0)
33 CONTINUE
35 CONTINUE
CALL PLTPGF
RETURN
50 FORMAT (F7.3+IH>)
END
SUBROUTINE STRAIN (X,Y,Z,      ETA2,      PN1,NETA,J,K)
DIMENSION X(),Y(),Z(),      SYM2(3),SYM3(2)+SYM4(2)+SYM5(2),
1SYM6(2),X1(2),X2(2)
LEVFL2+X,Y,Z
UATA I/0+YS/100./+XS/1.0E6/
DATA(SYM2(K),K=1,3)/10HETA2 = .10H   (.5H )>/.
2 (SYM3(K),K=1,2)/10HTIME (MICP,10H0SECONDS)>/.
3 (SYM4(K),K=1,2)/10HSTRAIN (%)+IH>/.
4 (SYM5(K),K=1,2)/10HETAL COMPO,5HNEAT>/.
5 (SYM6(K),K=1,2)/10HETAP COMPO,5HNEAT>/
C
IF(I.FQ.0)GOTP 10
I=I+
YPAR=YPAR+10.0
IF(I.LF.3)GOTC 25
CALL PLTPGF
GOTO 20
10 CALL PLTPGF
XL=7.9
YL=6.4
CALL FIXSCA(X()),N+XL+XS1+XMIN1+XMAX1+DX)
20 YPAR=0.0
I=1
25 YRAH=2.0+YRAHN
XPAR=2.0
CALL PLTSCA (XHAR,YHAR,0.0+0.0+1.0+1.0)
CALL PLTSYM (.1+SYM4(1)+90.0+ -1.2+2.5)
CALL PLTSYM (.1+SYM3(1)+0.0+ 3.0+-0.6)
CALL PLTSYM (.1+0HLOCATION+0.0+ 3.0+-1.0)
CALL PLTSYM (.3+2H>+0.0+ 3.0+-1.0)
CALL PLTSYM (.3+2H>+0.0+ 6.0+-1.0)
IF(NETA.NE.0)GOTO 30
CALL PLTSYM (.1+6HOUTEP>+0.0+ 6.0+-1.0)
GOTO 35
30 CALL PLTSYM (.1+6HINNFR>+0.0+ 6.0+-1.0)
35 CALL PLTSYM (.1+SYM2(1)+0.0+ 4.1+-1.1)
FACODE(8,50,STEP)ETA2
CALL PLTSYM (.1+STEP+0.0+ 4.0+-1.1)
FACODE(8,50,STEP)PN1
CALL PLTSYM (.1+STFP+0.0+ 5.7+-1.1)

```

```

X1(1)=2.05          STRAIN    47
X2(1)=-1.3         STRAIN    43
X1(2)=3.576        STRAIN    44
X2(2)=-1.3         STRAIN    45
CALL PLTOTS (1.0,X1(1),X2(1),2.0)      STRAIN    46
CALL PLTSYM (.1,SYMS(1),0.0,     3.7,-1.3)  STRAIN    47
X2(1)=-1.5         STRAIN    48
X2(2)=-1.5         STRAIN    49
CALL PLTOTS (4.0,X1(1),X2(1),2.0)      STRAIN    50
CALL PLTSYM (.1,SYMS(1),0.0,     3.7,-1.5)  STRAIN    51
CALL FIXSCA (Y(J),N,YL,YS1,YMIN,YMAX,DY)  STRAIN    52
CALL CONSCA (Z(J),N,YL,YS1,YMIN,YMAX,DY)  STRAIN    53
CALL PLTSCA (XRAR,YPAP,XMIN,YMIN,XS1,YS1)  STRAIN    54
CALL PLTAXS (DX,DY,XMIN,XMAX,YMIN,YMAX,DX)  STRAIN    55
CALL PLTDOT2 (1.0,X(1),Y(J),N,0)        STRAIN    56
CALL PLTDOT2 (4.0,X(1),Z(J),N,0)        STRAIN    57
CALL LABELA (DX,DY,XMIN,XMAX,YMIN,YMAX,XS,YS)  STRAIN    58
RETURN           STRAIN    59
50 FORMAT (F7.3+1H>)
END
SUBROUTINE FIXSCA (X,NPTS,S17E,XSCALE,XMIN,XMAX,DX)
DIMENSION X(1)+T(2)
C      MUST READ IN FIXSCA TO PUT X IN LFVEL 2.
LFVFL2,X
C(I) X=A1 INEAH ARRAY OF NUMBERS IN DATA UNITS
C(I) NPTS=THE NUMBER OF X VALUES
C(J) THE LENGTH OF X DIMENSION OF THE GRAPH IN PLOTTER UNITS
C(R) XSCALE=THE SCALE IN DATA UNITS/PLOTTER UNIT
C(R) XMIN=THE ADJUSTED MINIMUM IN DATA UNITS
C(P) XMAX=THE ADJUSTED MAXIMUM IN DATA UNITS
C(P) DX=THE DELTA X FOR AXIS IN DATA UNITS
LOGICAL CONT
CONT=.FALSE.
GOTO 100
ENTRY CONSCA
CONT=.TRUE.
100 TXMI=X(1)
TXMA=X(1)
DO 120 I=1,NPTS
IF(X(I).GE.TXMI)GOTO 110
TXMI=X(I)
GOTO 120
110 IF(X(I).LE.TXMA)GOTO 120
TXMA=X(I)
120 CONTINUE
IF(.NOT.C(NT))GOTO 140
IF(TXMI.LT.XMIN,.OR.TXMA.GT.XMAX)GOTO 125
TXMI=XMIN+ARS(XMIN)*0.000001
TXMA=XMAX-ARS(XMAX)*0.000001
GOTO 140
125 IF(TXMI.LT.XMIN)GOTO 130
TXMI=XMIN+ARS(XMIN)*0.000001
130 IF(TXMA.GT.XMAX)GOTO 140
TXMA=XMAX-ARS(XMAX)*0.000001
140 DIFF=TXMA-TXMI
IF(DIFF.LT.0.000001) DIFF=0.000001
FNCOFF (20, 150, T) DIFF
150 FORMAT(1PF12.5)
DPCODE (20, 160, T) COFF, IEXP
160 FORMAT(FR.5.1X,I3)
170 IF(COFF.GT.2.0)GOTO 180
DELTA=.1
GOTO 200

```

180 IF(COFF.GT.4.0)GOTO 190	FIXSCA	45
DEFLTA=0.2	FIXSCA	46
GOTO 200	FIXSCA	47
190 DELTA=0.5	FIXSCA	48
200 DELTA=DELTa*10.0**TFXP	FIXSCA	49
DX=DEFLTA	FIXSCA	50
XMIN=INT(ARS(TXMI)/DELTA)*DEFLTA	FIXSCA	51
IF(TXMI.LT.0)XMIn=- (XMIN+DELTA)	FIXSCA	52
XMAX=INT(ARS(TXMA)/DELTA)*DELTA	FIXSCA	53
IF(TXMA.LT.0)GOTO 210	FIXSCA	54
XMAX=XMAX+DELTA	FIXSCA	55
GOTO 220	FIXSCA	56
210 XMAX=-XMAX	FIXSCA	57
220 XSCALE=(XMAX-XMIN)/SIZE	FIXSCA	58
RETURN	FIXSCA	59
FNU	FIXSCA	60

## APPENDIX E

### EXAMPLES

Two examples are given in this appendix. These were originally jobs run on the BRLESC computer, now defunct, and transferred to the CDC CYBER 70 Model 76 computer to check the program there. The two jobs test, and hence illustrate, many of the available options and internal variations of RPSL1D. For both examples, the complete SCOPE 2.1 batch job is listed and a sample of the output is given. The sections of the batch job are separated in the listings by question marks which listed for the multipunch end of record and end of information signals.

#### E.1 Example 1: Pressure Loaded Flat Plate

The first example involves finding the response along the shorter symmetry line of a flat 4 by 15 inch soft aluminum plate. (Results from the standard REPSIL and the slab symmetric version of RPSL1D are very similar for this elongated plate.) This example demonstrates the introduction of a new PRESS subroutine, the use of the optional coding PLOTP, and the activating of Gaussian integration and damping. Both the new PRESS routine and the option PLOTP are inserted by making changes in the program through UPDATE; both also require input data cards. With the catalogued INGEOM, Gaussian integration is automatically used with slab symmetry if IB = 0 on input card 14. Damping was activated from time  $\Delta t$  on by setting MDAMP = 0 on input card 5. The damping option was used here to help simulate displacement of the plate by quasi-static pressure loading. This is not the usual use of damping. Damping is a device normally used to bring a responding surface to rest at its final equilibrium configuration after the loading is removed. Since subroutine DAMP is not entered until cycle one, when NCYCLE = 1, the subroutine did not shut off pressure loading as it will if entered with NCYCLE = MDAMP. This run terminated at cycle 3546 with a displacement of .66 inches.

A listing of the batch job deck and samples of output for Example 1 follow. The first part of the job deck is the SCOPE 2.1 control statements to set up and run RPSL1D and the plotting. The next set of cards is UPDATE directives and new FORTRAN statements for RPSL1D. The input data cards for RPSL1D are next. (The input and output for this example are in the now forbidden pound-inch-second system. If this is discomforting, pretend the units are SI units for some very exotic material.) After the data there are UPDATE changes for the plotting program, including changes for the PLOTP option and a change to permit up to 6000 time cycles. Finally, there is a data card to control plotting.

Following the job listing are sample listings of output. This starts with a summary of the input and the initial Cartesian coordinates as prescribed by INGEOM. All the output should be inspected to uncover errors, but these two sets should receive very close attention.

The output at cycle 3000 for this example is listed next. This includes:

- Displacement increments, coordinates, and pressure at each mesh point including the virtual external point at N=1 and the symmetry point at N=13.
- The LMAT matrix. A "1" signifies plasticity, and a "0" denotes an elastic response, on the current cycle. For each column labeled N, there are two rows for each integration station K. The first row is for mesh point N. The second row is for midmesh N.
- The components of the surface normal at each mesh point, the 2 columns on the left, and at each midmesh.
- Surface strains at all mesh points. For each mesh point, the strains on the upper surface are listed first, then the strains on the lower surface are listed. EC1 and EC2 are the elongation strains in the  $\xi^1$  and  $\xi^2$  directions, respectively. E11 and E22 are internal covariant components of strain. For slab symmetry, EC1 and E11 are zero.
- The tabulated output for the NSTRN prescribe surface strains accompanied by prints of extreme strains and maximum deflection for the current cycle, and for the entire run.
- The energy balance summary.

In our listing, this is followed by three prints noting the use of the kinetic energy annihilating procedure (a part of damping which stops motion at kinetic energy maxima) and finally a print of the prescribed surface strains, etc. at cycle 3500.

The occurrence and frequency of this output is controlled by input cards 8, 9, and 10.

Figures E.1.1 through E.1.4 are samples of plotting output. For most publication purposes, it is desireable to do some relabeling, but these plots have not been retouched. Figure E.1.1 shows displacement at selected time steps. Figure E.1.2 shows the displacement of the selected central mesh point and the energy balance as functions of time. The scallops in these plots occur when the stopping procedure was exercised. This energy balance plot shows the four possible curves. These are, from low to high: kinetic energy, kinetic energy plus elastic strain energy, kinetic energy plus elastic strain energy plus the work in damping, and the total energy. Figure E.1.3 is the first two of the six strain plots requested for this run. Figure E.1.4 is a plot of force per initial unit area at mesh point 3. The pressure increased linearly to a maximum of 80 psi and then remained constant, but the area increased throughout the entire run.

Listing of the Batch Job Cards for Example 1

```

KOF TM,STMF7.TRM.  DUPLICATE RUN OF 1/3/77.  TEST GAUSS. PLOT.P LINPRS.
ACC(1,1,T,RRRRR).          XMFTRM,4304,13C79
ATTACH(CLEFF) &PSL1D+TD=JEW)
UPDATEF+F.
FTN(A,I,SL=0,R=0)
MAP,OFF.
LGO.
EXITT(1)           DATA TO PLOT TAPE
PFLINC(PLOT)
RETURN,COMPILE.
RETURN,LGO.
PFLURN,CLDPL.
ATTACH(CLDPPL,WPS11MPLOT+TD=JRW)
REGIN,ATTACH,PLOTLIB.
UPDATEF(F)
FTN(A,I,FL=F,SL=0,R=0,LCM=1)
MAP,OFF.
LGO.
EXITT(1)
REGIN,PLOT,CALCOMP+TAPE13.
?
*UDENT PLOTP
*T PSL1D,S
C          PLUTTING F(N).  9/30/75          PLOTP
*T MAIN,PR
    COMMON NPF,NPE( 9 ),PDAT( 9 )
*T START,24
    READ(E,110)NPF+NPE(I),I=1,NNPF
*T PDAT8,13
    WRITF(PLOT)NPF+NPE(I)+I=1,NNPF
*T PDAT8,10
    DO 24 I=1,NNPF
        J=NPF(I)
    24 PDAT(I)=E(J)
*T PDAT8,27
    1 +(PDAT(J)+J=1,NNPF)
*T PDAT8,42
    DO 46 T=1,NNPF
        U=NPF(I)
    46 PDAT(T)=E(J)
*T PDAT8,45
    1 +(PDAT(J)+J=1,NNPF)
*T FFT 11PFS
*P DFFSS,4,DFFSS,14
C          PRESSURE BY LINEAR INTERPOLATION WITH RESPECT TO TIME IN      LINPRS
C          INPUT TABLE TPF(T),PPR(T).   10/7/76      LINPRS
C          FORMAT (ZF10.3)      TERMINATE WITH NEGATIVE TPF.      LINPIN
C          DIMENSION TPF(51),PPR(51)      LINPRS
C          DATA TPFSS/0/
C          IF(TPFSS .GT. 0)GOTO 4      LINPRS
C          WRITE(A,100)
C          T = 0      LINPRS
C          1 = 0      LINPRS
C          2 = 1      LINPRS
C          IF(E(1,101)TPF(I),PPR(I))GOTO 3      LINPRS
C          IF(TPF(I) .LT. 0.0)GOTO 3      LINPRS

```

```

      WRITE(6,112)T,T-(I)+PPI(I)
      GOTO 2
  2  IPRESS= T-1
  4  CALL DVI(TIT(TIMF,FN,THK+PRH+IPRESS+?))
  FN = MELV+NZV
  P(N)= FG
  5  CONTINUE
  RETURN
100 FORMAT(//1 PRESSURE FROM A TABLE!/4X,1T1.5X,1TIME!,9X,1PRESSURE!)
101 FORMAT(2E10.3)
102 FORMAT(1E.2E15.7)

?
STRUCT. MODEL. .020 PLATE.  GRADUALLY INCREASING PRESSURE TO MPASI 1/3/77
      5   10    4 2.0
      4000    0 0000 0.400000E-6
      2   2   1   1
      1 0000    0 0.1          0.001
10.000000FF 0.33      5000.      2.530000E-4 .020           4   0
5000.          .0005H
5700.          .035
11400.          .10
13050.          .25
      500    1   1   1
      6  500 1000 1500 2000 3000 3500
      6  500 1000 1500 2000 3000 3500
      2  500 3500
      1   3
      0.0    2.0      6
      0.0    0.0    45.0    135.0      0
      0.0    1.0    45.0    135.0      0
      0.0    2.0    45.0    135.0      0
      0.0    6.0    45.0    135.0      1
      0.0    1.0    45.0    135.0      1
      0.0    2.0    45.0    135.0      1
      2.0    7.5      0
      0.0    0.0
      0.00125  45.0
      1.0    45.0
      -1.0
?
*PLOT PLOTH
*P1PLT.1H
      PEAR(MPLT)MPF*(MPF(I)+[I]+MPF)
*P1PLT.40
      1 +(PDATA(J)+J=1+NUPF)
*TP  HED02
*TP  MAINT.0
      1  TIME( HED02)+ 0.2( HED02)+ 0.3( HED02)+ CIN( HED02)+ STC( HED02)+ F002
      2  TNE( HED02),DAMPIT( HED02)*FHSS1(2E012)+EPSS2(3E012)           6E02
      COMMON/PLOTH/F0( HED02+0)
*P1PLT.10
      MAX1 = HED02
?
      1.0    3.0
?

```

ANALYSIS CODE  
 STRUCT. MODEL. -928 PLATE. MANUALLY INCREASING PRESSURE TO NPSI 1/3/77  
 COMPUTATIONS FOR FAULT USE DEFLATE -1500000.02  
 10 REPEATS IN E1E2 DIRECTION (IN TIME .200000000001)  
 READING TIME INCREMENT -164400E-04  
 READING TIME INCREMENT -1.000000E-06  
 INPUT TIME INCREMENT -0.000000E+00  
 TIME INCREMENT USED BY NPSI - .000000000001  
  
 YOUNG'S MODULUS = .100000E+00  
 POSITION'S RADIUS = .230000E+00  
 MASS DENSITY = .0733999E-01  
  
 START AT TIME STEP 0  
 FINAL TIME STEP 0.000000000001  
 SURFACE STRESSES EVERY 500 TIME STEP  
 RESTANT WHILE EVERY 4000 TIME STEP  
 LAYER = 1  
 LAYER = 0  
 PRESSURE = 99999  
 LAYER = 99999  
  
 1/2/3/4/ = CLAMPED/SYMMETRIC/ROTATIONAL/ROTATIONAL  
 FND1 SINCE41 = 1  
 FND2 SINCEP1 = 1  
  
 PRINT OPTION CONTROL CARD  
 0/1 = NO PRINT/PRINT  
 1 DISPLACEMENT INCREMENTS  
 1 CONSTITUTIVE COORDINATES, PRESSURE  
 1 SURFACE NORMAL VECTOR COMPONENTS  
  
 PRINT INFORMATION AT THE FOLLOWING TIME STEPS  
 500 1000 1500 2000 3000 3500  
 PRINT L MATRIX (LPRINT) AT THE FOLLOWING TIME STEPS  
 500 1000 1500 2000 3000 3500  
 3-D PLOTS FROM THE FOLLOWING TIME STEPS  
 500 3500  
  
 CONSTITUTIVE RELATION, ELASTOPLASTIC-YOUNG'S-MODULUS-STRAIN RATE INDEPENDENT  
 STRESS-STRAIN APPROXIMATION HAS 4 SUBLAYERS  
  
 STRESS-STRAIN AND STRAIN RATE PARAMETERS  
 1 2 3 4  
 S111111 S222222 S333333 S444444  
 1.000000E+03 5.000000E+04 0.000000E+00 0.000000E+00  
 2 9.799999E+02 3.333333E+02 0.000000E+00 0.000000E+00  
 3 1.140000E+03 1.000000E+03 0.000000E+00 0.000000E+00  
 4 1.395000E+03 2.333333E+03 0.000000E+00 0.000000E+00  
  
 START BARING AFTER TIME STEP 0.100000000001  
 PRESSURE = .100000E+00

Selected Output from Example 1



11CC1.0.CP0.0110.02271  
Upper Main. 210-211m.

20-1999-0000000000.

LITERATURE

PRESSURE (MPA) 1.0 0.8 0.6 0.4 0.2 0.0



СИДАР СИДАЛАН. УПРАВЛЕНИЕ. ИССЛЕДОВАНИЯ. ОБРАЗОВАНИЯ.

卷之三

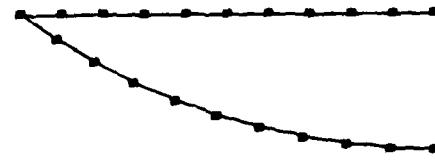
FRAT	N	FACT	ANGLE φ	ANGLE θ	ANGLE ϕ	ANGLE
0.000	2.000	UNDEF	-11993241E+00	45.00	-61662063E-01	-61662063E-01
1.000	1.000	UNDEF	-631655241F-01	45.00	.32865969E-01	.32865969E-01
7.000	17.000	UNDEF	-63966343E-01	45.00	.32491038E-01	.32491038E-01
1.000	2.000	UNDEF	-77662247E-02	45.00	.3N662247E-02	.3N662247E-02
1.000	7.000	UNDEF	-66019965E-01	45.00	.3521378E-01	.3521378E-01
1.000	12.000	UNDEF	-67953347E-01	45.00	.3556362E-01	.3556362E-01

S1801118466 AD TIME

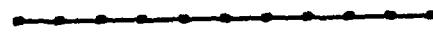
113



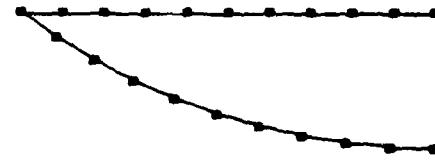
CYCLE 500  
460.0 MICROSECONDS



CYCLE 3546  
3191.4 MICROSECONDS



CYCLE 0  
0 MICROSECONDS  
DEFLECTION MAGNIFIED 1.000  
SCALE 1/ 0



CYCLE 3546  
3190.5 MICROSECONDS

Figure E.1.1. Plots of the reference curve at selected times.

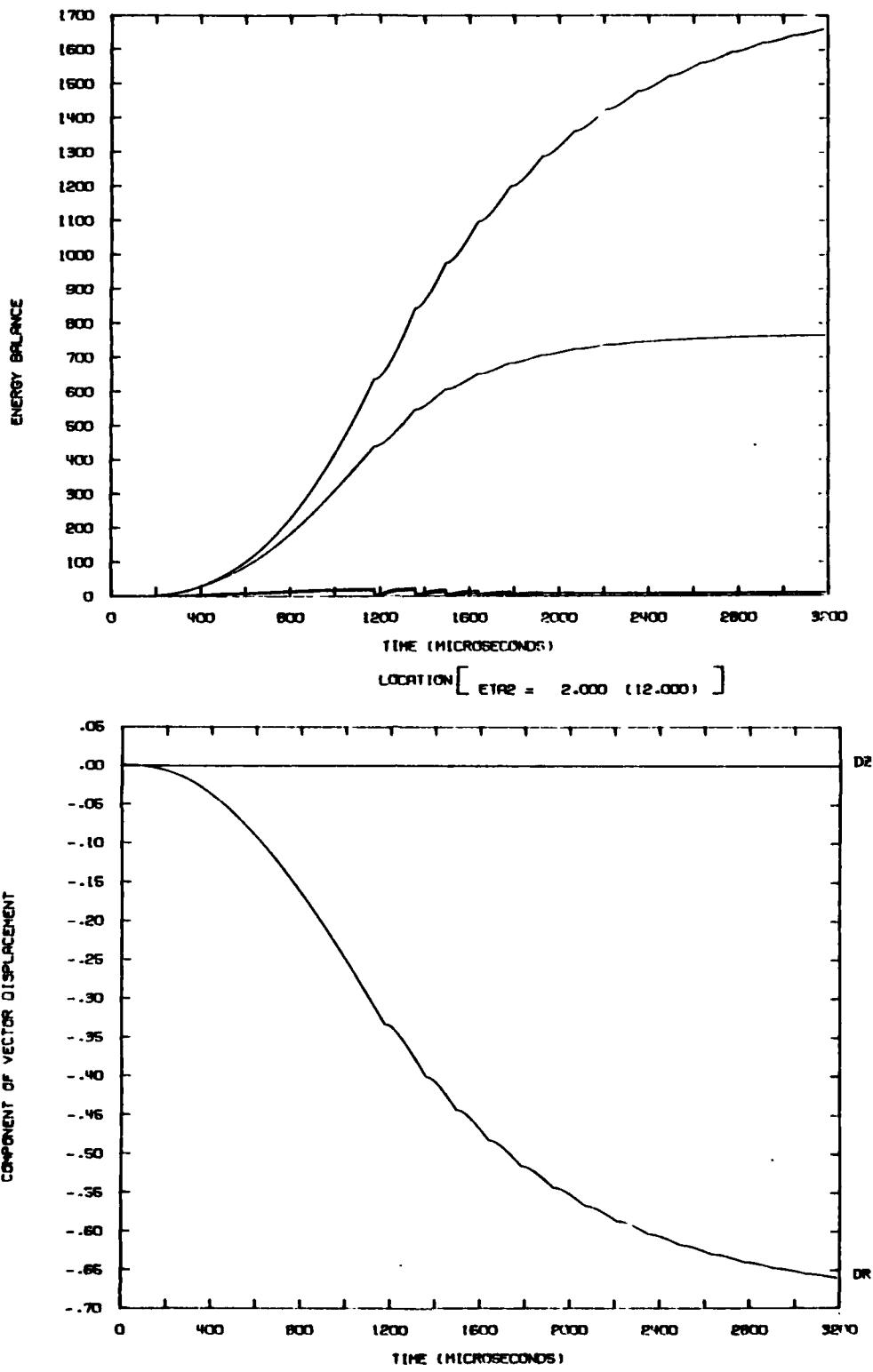


Figure E.1.2. Displacement history of the central point and energy balance curves.

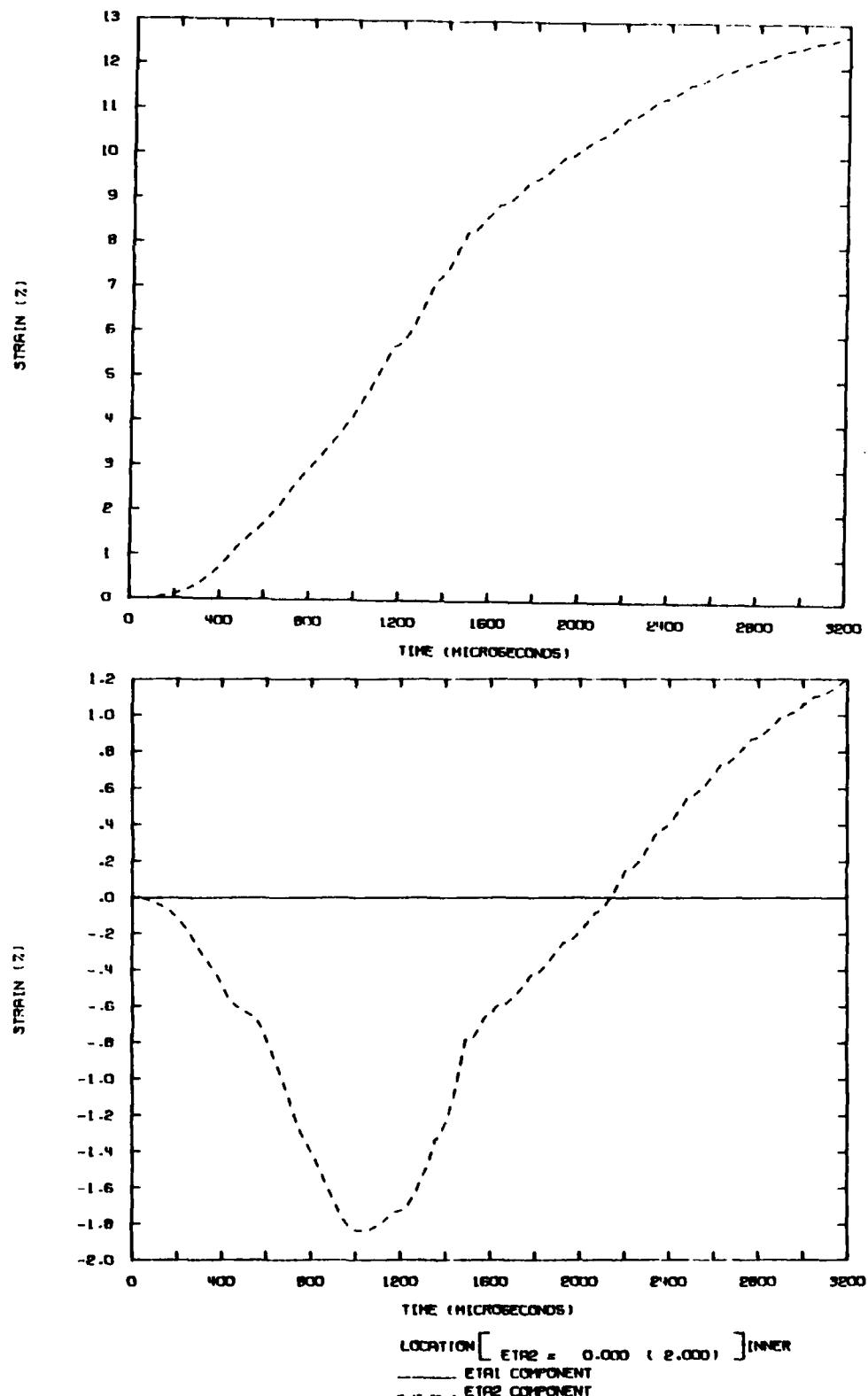


Figure E.1.3. Strain histories at the edge on the top and bottom surfaces

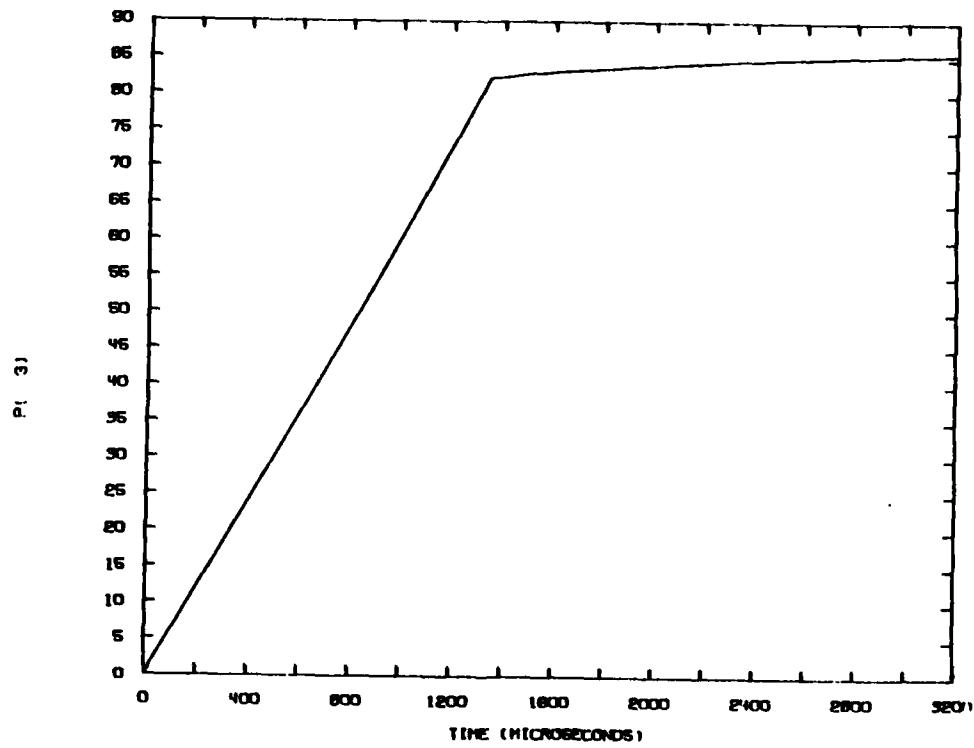


Figure E.1.4. Force per initial unit area at mesh point B.

## E.2 Example 2: Eroding Rod

This example uses the beam option of RPSL1D and utilizes all seven of the options listed in Section 5. This program was designed to illustrate that the test strain-time records at two locations along a steel rod penetrating a steel plate could be reproduced by varying the force and the rate of erosion on the end. Two quite different tables of erosion and force were used, and both returned strains that were close to the test values at three locations. This example used a fast erosion rate. Mesh points 2 and 3 were eroded by time cycle 5.

The batch job cards and some sample output are listed. The batch job for this example is rather long. The first section of the run stream is the SCOPE 2.1 control statements. The next section is the UPDATE directives and new FORTRAN statements for RPSL1D. Breaking this section into the nine groups with different UPDATE card identifiers we have: PLOTP, SHR3/1, MSQSVS, BSTRS, APLFRC (including a subroutine ENDFRC which is adjusted for erosion), EAPFRC, 11/2/76 (a modification for EAPFRC), INVELC (an insert into INVEL which creates a uniform initial velocity toward end 1), and ERODE.

The next section in the job cards is the input data. The input, and output, for this example are in SI units: meters, seconds, kilograms, Pascals, etc. This leads to some awkward scaling on both input and output, but assures a consistent, unambiguous system. Along with the usual input data, this section contains a long table of strain vs. stress for the BSTRS option, input to approximate a solid cylindrical beam for INGEOM, no input for PRESS, a card with initial velocity for INVEL, three tables for ENDFRC, and a table for ERODE.

The next section is UPDATE changes for the file RPSL1DPLOT for options MSQSVS and PLOTP. Finally, there is an input data card to control the plotting.

The tabular output listed is two of the seven pages which reflect input (most of the omitted tabulations are simply copies of input tables), and the output at cycle 130. The output at cycle 130 includes:

- The displacement and coordinates. R and DR are effectively zero.
- The output from SHR3/1. The moment, MS, and shear, VS, are zero. Only axial force, QS, is significant.
- The table of plastic stress occurrence.
- The components of the unit normal vector. The rod has not bent.
- The surface strains.
- The energy balance.

Six of the output plots are shown next. (An isometric plot of the rod and plots of MS, QS, and VS were made at cycles 70 and 130. The isometric plot is uninteresting, and MS and VS are zero.) A plot of QS vs. mesh point number at cycle 130 is included to illustrate output of the MSQSVS option. We also include plots of several functions of time: the displacement of mesh point 5, strain plots at 19.26 mm and 39.67 mm, a plot of force on end 1, EFZ1, which was stored in P(53), and the erosion at end 1 which was stored in P(54).

Listing of the batch job cards for Example 2

```

WOPTM,STMF7,T30.   DUPLICATE RUN OF 6/8/77 AND 10/12/77.  ERUDING RCD.
ACCOUNT.           WORTMAN,B309,X3979
ATTACH,OLDPL,HPSSL1D.ID=JRW.
UPDATE,F.
FTN(A,I,SL=0,R=0)
MAP,OFF.
LGO.
EXIT(U)           DATA TO PLOT TAPE
REWIND(PLOT)
RETURN,COMPILE.
RETURN,LGO.
RETURN,OLDPL.
ATTACH(OLDPL,HPSSL1DPLOT.ID=JRW)
HEGIN,ATTACH,PLOTLP.
UPDATE(F)
FTN(A,I,EL=F,SL=0,R=0)
MAP,OFF.
LGO.
EXIT(U)
BEGIN,PLOT,CALCOMP,TAPE13.
?
*IDENT PLOTP
*I HPSSL1D.5          PLOTP
C      PLOTTING P(N).  9/30/75
*I MAIN.28            PLOTP
COMMON NNPE,NPE( 9),PDATA( 9)
*I START.24           PLOTP
READ(5,110)NNPE,(NPE(I),I=1,NNPE)
*I PDATA.13           PLOTP
WRITE(NPLOT)NNPE,(NPE(I),I=1,NNPE)
*I PDATA.19           PLOTP
DO 26 I=1,NNPE
J=NPE(I)
26 PDATA(I)=P(J)
*I PDATA.27           PLOTP
1.(PDATA(J),J=1,NNPE)
*I PDATA.42           PLOTP
DO 46 I=1,NNPE
J=NPE(I)
46 PDATA(I)=P(J)
*I PDATA.45           PLOTP
1.(PDATA(J),J=1,NNPE)
*IDENT SHF3/1
*I HPSSL1D.5          SHF 3/1
C      ESTIMATE SHEAR FORCE. VS. AND MS AND QS.  3/1/76 (FOR A PEAN)  SHF 3/1
*I MAIN.28            SHF 3/1
COMMON AMS(103),NS(103),VS(103)
*I RESULT.48           SHF 3/1
C      MS AT FREE END IS ZERO (4/5/76).  CHANGE WITH APLFAC.  SHF 3/1
AMS(NPR)=0.0
*I RESULT.54           SHF 3/1
AMS(N)=A22*F22
*RS RESULT.62          SHF 3/1
VS(N)=1.45*(N+1)+1.45*(N+1)/(CFT32*SHF)
200 NS(N)=A22*(C22-RH22*F22)  SHF 3/1

```

```

C      VS,MS,OS PRINT CONTROLLED BY CODING.  SHR 3/1/76.
C      TO PRINT EVERY 10 CYCLES.
C      IF(MOD(NCYCLE,10) .EQ. 0) GOTO 212
C      TF(NCYCLE ,EQ. 70 .OR. NCYCLE .EQ. 130)GOTO 212
C      GOTO 218
210 FORMAT(// CYCLE N,FX,'MS',13X,'VS',13X,'QS',13X,'A22',12X,
1 'H22',12X,'Q22',12X,'M22',/)
211 FORMAT(I6,I3,1P7E15.7)
212 IF(N .EQ. NIP )WRITE(6,210)
      WRITE(6,211)NCYCLE,N,AMS(N),VS(N),QS(N),A22,BM22,Q22,F22
C      OUTPUT OF MS(N2B) 4/5/76
      IF(N .EQ. N2B-1)WRITE(6,211)NCYCLE,N2B,AMS(N2B)
C      218 CONTINUE
*IDFNT  MSQSVS
*I  RPSL1D.5
C      PLUTTING MS, QS, AND VS. 3/19/76
*I  PDATA.52
      WRITE(NPLOT)(AMS(N),QS(N),VS(N),N=N1E,N2B)
*IDENT  HSTRS
*I  RPSL1D.5
C      NEW UNIAXIAL STRESS-STRAIN CURVE FOR BEAMS.  4/1/77
C      NOT COMPATIBLE WITH STREN OPTION.
*D  MAIN.5
      3 SIG1(103,6),SIG2(103,6),LMAT(103,6)
*D  MAIN.26
      COMMON SNRM(103),SNKM(103),SIG1M(103,6),SIG2M(103,6),LMATM(103,6)  ESTRS
*I  MAIN.28
      COMMON EPSR(103,6),EPSPRM(103,6),EPLR(103,6),EPLRM(103,6),
      2 EFR(100),SSH(100),NEST,EPSZ
*I  RPSL1D.60
      EPSR(N,K) = 0.0
      EPSPRM(N,K) = 0.0
      EPLR(N,K) = 0.0
      EPLRM(N,K) = 0.0
*D  START.16,START.20
      DO 700 I=1, 99
      READ(5,702)EFA(I),SSP(I)
      IF(EFR(I) .LT. 0.0) GOTO 701
700 CONTINUE
701 NFST = I
      NSFL = 1
      SSP(I) = SSH(I-1)
      EFA(I) = 1000.
      SSP(I) = SIGZ
      EPSZ = SIGZ/F
      EFR(I) = EPSZ
      WRITE(6,703)(I,EEB(I),SSB(I),I=1,NFST)
      IF(SSB(NFST) .GT. 2.0*SIGZ)WRITE(6,704)NEST
702 FORMAT(2F15.7)
703 FORMAT(3X,'STRESS-STRAIN TABLE FOR HSTRS',3X,'I',6X,'FPS(I)',9X,
2 'SIG(I)',/(5,1P2F15.7))
704 FORMAT(// ***WARNING*** STRAIN FNFNGY SUSPICIOUS.  SIG(*,I3,
2 ') .GT. 2*SIGZ')
*D  HMSTRS,73,HNSTRS,30
C      CHANGES IN HMSTRS FOR BSTRS 4/1/77
      IF(IP .EQ. 1)GOTO 10
      PEPS = FPSRM(N,K)
      HEP1 = EPLRM(N,K)
      GOTO 11
10  HEP1 = FPSR(N,K)
      HFPL = FPLH(N,K)
11  HFPLM = HFPL
      IF(HFPLM .GE. 0.0) GOTO 13

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```

SIG22I = -SIG22I          BSTHS
DEPS22 = -DEPS22          BSTHS
HFPS = -HFPS              BSTHS
REPL = -REPL               BSTHS
13 REPS = HFPS + DEPS22   BSTHS
IF (DEPS22 .GE. 0.0) GOTO 14 BSTHS
IF (HFPS .GE. REPL-FPSZ) GOTO 15 BSTHS
IF (REPS .GT. -FPSZ) GOTO 16 BSTHS
CALL DVINT( -HFPS,SIG22,EEH,SSB,NEST,2) BSTHS
SIG22 = -SIG22             BSTHS
GOTO 17                   BSTHS
14 IF (REPS .LE. REPL+EPSZ) GOTO 15 BSTHS
CALL DVINT(BFPS,SIG22,EEH,SSB,NEST,2) BSTHS
HPL = BFPS - EPSZ          BSTHS
LMNK = 1                   BSTHS
GOTO 1K                   BSTHS
15 SIG22 = SIG22I + E*DEPS22 BSTHS
GOTO 18                   BSTHS
16 CALL DVINT(REPS+2.0*EPSZ,SIG22,EEH,SSB,NEST,2) BSTHS
SIG22 = SIG22 - 2.0*SIGZ   BSTHS
17 LMNK = 1                 BSTHS
REPL = REPS + EPSZ         BSTHS
18 IF (RFPLM .GE. 0.0) GOTO 19 BSTHS
SIG22 = -SIG22             BSTHS
HFPS = -BFPSS              BSTHS
REPL = -REPL               BSTHS
19 IF (IP .EQ. 1) GOTO 20   BSTHS
FPSHM(N,K) = HFPS          BSTHS
EPLRM(N,K) = BFPSS         BSTHS
GOTO 21                   BSTHS
20 EPSB(N,K) = REPS         BSTHS
FPLH(N,K) = RFPL            BSTHS
21 CONTINUE                BSTHS
*IDENT APLFRC
*1 RPSL1D.5
C      APPLIED FORCE (AND MOMENT) AS FND CONDITION 4.  4/12/76      AFL FRC
C      APL FRC ASSUMES SHP 3/1 IS INCLUDED IN THE DECK.           AFL FRC
C      THE APPLIED FORCE CHANGE TO (1-D) REPSTL FOR BEAMS REQUIRES AFL FRC
C      A USER SUBROUTINE, ENDFRC, TO SUPPLY END FORCES AND MOMENTS. AFL FRC
*I MAIN.28
COMMON EFR1,EFZ1,EM1,FFR2,EFZ2,FM2                           AFL FRC
*I START.54
IF (IRCF1 .EQ. 4) N1V=N1R                                     AFL FRC
*# START.230
IF (IRCF1 .LT. 5 .AND. IBCE2 .LT. 5) RETURN                 AFL FRC
*D GRAD.9,GRAD.10
IF (IRCF1 .NE. 4 .OR. N .GT. N1B) GOTO 10                  AFL FRC
C      IRCF1=4 AND N=N1R.  USE FORWARD DIFFERENCES
R2 = RTD2*(-3.0* R(N)+4.0* R(N+1)- R(N+2))                 AFL FRC
Z2 = WTD2*(-3.0* Z(N)+4.0* Z(N+1)- Z(N+2))                 AFL FRC
DR2 = RTD2*(-3.0*DR(N)+4.0*DR(N+1)-DR(N+2))                 AFL FRC
DZ2 = RTD2*(-3.0*DZ(N)+4.0*DZ(N+1)-DZ(N+2))                 AFL FRC
R22 = RD22*(2.0* R(N)-5.0* R(N+1)+4.0* R(N+2)- R(N+3))    AFL FRC
Z22 = RD22*(2.0* Z(N)-5.0* Z(N+1)+4.0* Z(N+2)- Z(N+3))    AFL FRC
DR22 = RD22*(2.0*DR(N)-5.0*DR(N+1)+4.0*DR(N+2)-DR(N+3))   AFL FRC
DZ22 = RD22*(2.0*DZ(N)-5.0*DZ(N+1)+4.0*DZ(N+2)-DZ(N+3))   AFL FRC
GOTO 20
# IF (IBCE2 .NE. 4 .OR. N .LT. N2R) GOTO 9
C      IRCF2=4 AND N=N2R.  USE BACKWARD DIFFERENCES.
*D GRAD.34,GRAD.35
IF (IRCF1 .NE. 4 .OR. N .GT. N1R ) GOTO 18                 AFL FRC
C      IRCF1=4 AND N=N1R (+1/2).  USE NON-CENTRAL 4 POINT PND DIFFERENCES
R22 = RD22*(3.0* R(N)-7.0* R(N+1)+5.0* R(N+2)- R(N+3))   AFL FRC

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Z22 = RD22M*(3.0* Z(N)-7.0* Z(N+1)+5.0* Z(N+2)- Z(N+3)) AFL FRC
DR22 = RD22M*(3.0*DR(N)-7.0*DR(N+1)+5.0*DR(N+2)-DR(N+3)) AFL FRC
DZ22 = RD22M*(3.0*DZ(N)-7.0*DZ(N+1)+5.0*DZ(N+2)-DZ(N+3)) AFL FRC
GOTO 20 AFL FRC
18 IF(IPCE2 .NE. 4 .OR. N .LT. N2B-1)GOTO 19 AFL FRC
C IHCE2=4 AND N=N2B-1 (+1/2). USE NON-CENTRAL 4 POINT 2ND DIFFERENCES AFL FRC
*P RESULT.34,PRESULT.49 AFL FRC
    IF(N .EQ. N2B .AND. IPCE2 .EQ. 4)GOTO 13 AFL FRC
    IF(N .NE. N1B .OR. IPCE1 .NE. 4)GOTO 16 AFL FRC
    N=N1B AND IPCE1=4. APPLIED FORCE FND1. AFL FRC
    CALL FNDFRC AFL FRC
    AMS(N1B) = EM1 AFL FRC
    GOTO 18 AFL FRC
    N = N2B AND IPCE2=4. APPLIED FORCE END2. AFL FRC
13 IF(IPCE1 .NE. 4)CALL ENDFRC AFL FRC
    AMS(N2B) = EM2 AFL FRC
    GOTO 18 AFL FRC
*P MOTION.11,MOTION.18 AFL FRC
C     APPLIED FORCE (AND MOMENT) AS FND CONDITION 4. 4/12/76 AFL FRC
    IF(IPCE1 .NE. 4 .OR. N .GT. N1B+1)GOTO 13 AFL FRC
    IF(N .GT. N1B)GOTO 11 AFL FRC
C     N=N1B AND IPCE1=4. USE DIFFERENCE (F(N+1/2)-F(N))/(DETA2/2). AFL FRC
    VR = 2.0*RTD2M*(VS(N)*SNRM(N)-QS(N)*SNKM(N)-EFR1) AFL FRC
    VZ = 2.0*RTD2M*(VS(N)*SNKM(N)+QS(N)*SNRM(N)-EFZ1) AFL FRC
    GOTO 35 AFL FRC
11 M=N-1 AFL FRC
C     N=N1B+1 AND IPCE2=4. USE DIFFERENCE (F(N+1/2)-F(N-1/2))/DETA2 AFL FRC
    VR=RTD2M*(VS(N)*SNRM(N)-QS(N)*SNKM(N)-VS(M)*SNRM(M)+QS(M)*SNKM(M)) AFL FRC
    VZ=RTD2M*(VS(N)*SNKM(N)+QS(N)*SNRM(N)-VS(M)*SNKM(M)-QS(M)*SNRM(M)) AFL FRC
    GOTO 35 AFL FRC
13 IF(IPCE2 .NE. 4 .OR. N .LT. N2B-1)GOTO 31 AFL FRC
    IF(N .LT. N2B)GOTO 16 AFL FRC
C     N=N2B AND IPCE2=4. USE DIFFERENCE (F(N)-F(N-1/2))/(DETA2/2) AFL FRC
    M=N2B-1 AFL FRC
    VR = 2.0*RTD2M*(EFR2-VS(M)*SNRM(M)+QS(M)*SNKM(M)) AFL FRC
    VZ = 2.0*RTD2M*(EFZ2-VS(M)*SNKM(M)-QS(M)*SNRM(M)) AFL FRC
    GOTO 35 AFL FRC
16 M = N2B-2 AFL FRC
C     N=N2B-1 AND IPCE2=4. USE DIFFERENCE (F(N+1/2)-F(N-1/2))/DETA2 AFL FRC
    VP=RTD2M*(VS(N)*SNRM(N)-QS(N)*SNKM(N)-VS(M)*SNRM(M)+QS(M)*SNKM(M)) AFL FRC
    VZ=RTD2M*(VS(N)*SNKM(N)+QS(N)*SNRM(N)-VS(M)*SNKM(M)-QS(M)*SNRM(M)) AFL FRC
    GOTO 35 AFL FRC
*I ROUND#9 AFL FRC
    IF(IPCF1 .EQ. 4)GOTO 20 AFL FRC
*I ROUND#7 AFL FRC
    IF(IPCE1 .EQ. 4)GOTO 20 AFL FRC
*AF ENDFPC AFL FRC
*DFPC SUBROUTINE ENDFPC AFL FRC
C     THE AFL FRC CHANGE TO (1-0) REPSIL FOR FEAMS REQUIRES A USER AFL FRC
C     SUBROUTINE ENDFPC TO SUPPLY FORCES AND MOMENTS AT BOTH ENDS. AFL FRC
C     FOR AN INITIALLY STRAIGHT BEAM, R(N)=0.0, Z(N)=(N-2)DETA2. AFL FRC
C     FFH1>0 DECREASES DZ(2), FFH1>0 DECREASES DR(2) AFL FRC
C     EFZ2>0 INCREASES DZ(N2B), EFR2>0 INCREASES DR(N2B) AFL FRC
C     EM1>0 DECREASES DR(2) AND INCREASES DR(3) AFL FRC
C     EM2>0 INCREASES DR(N2B-1) AND DECREASES DR(N2B) AFL FRC
C
*CALL MAIN AFL FRC
C     TANGLAR INPUT FOR FFZ1, EFTH, AND EFMU. 10/27/76 10/27/76
C     MODIFIED (12/7/76) FORCE RELATIVE TO NORMAL 12/7/76
C     EFR1 = EFMU*EFZ1. EM1 = 2U*(EFZ1*SIN(EFTH)+EFR1*COS(EFTH)) 10/27/76
C     DIMENSION (APELV(7)),FF7T(401),TMF7(401),EFTHT(401),TMTH(401), 3/9/77
C     1 EFMUT(401),TMMU(401) 3/8/77

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DATA IFENDF/0/
IF(IENDFW .GT. 0)GOTO 1
IENDFW = 1
10/27/76
READ (5,100)NFZPTS,(LARELV(I),I=1,7)
10/27/76
WRITE(6,101)(LARELV(I),I=1,7),NFZPTS
10/27/76
READ (5,102)(EFZT(I),TMFZ(I),I=1,NFZPTS)
10/27/76
WRITE(6,103)(I,EFZT(I),TMFZ(I),I=1,NFZPTS)
10/27/76
READ (5,100)NFTHS,(LARELV(I),I=1,7)
10/27/76
WRITE(6,104)(LARELV(I),I=1,7),NFTHS
10/27/76
READ (5,102)(EFTHT(I),TMTH(I),I=1,NFTHS)
10/27/76
WRITE(6,103)(I,EFTHT(I),TMTH(I),I=1,NFTHS)
10/27/76
READ (5,100)NFMUS,(LARELV(I),I=1,7)
10/27/76
WRITE(6,105)(LARELV(I),I=1,7),NFMUS
10/29/76
READ (5,102)(EFMUT(I),TMMU(I),I=1,NFMUS)
10/27/76
WRITE(6,103)(I,EFMUT(I),TMMU(I),I=1,NFMUS)
10/27/76
EFFR2 = 0.0
10/27/76
EFZ2 = 0.0
10/27/76
EM2 = 0.0
10/27/76
1 CALL DVINT(TIME,EFZ1,TMFZ,FFZT,NFZPTS,2)
10/27/76
CALL DVINT(TIME,EFTH,TMTH,EFTHT,NFTHS,2)
10/27/76
CALL DVINT(TIME,EFMU,TMMU,EFMUT,NFMUS,2)
10/27/76
EFFR1 = EFMU*EFZ1
10/27/76
EM1 = -7U*(SIN(EFTH)*EFZ1 + (1.0-COS(EFTH))*EFFR1)
10/29/76
C      CODE CHECKING PLOTS. 10/28/76
CC/10/76
P(53) = EFZ1
7/20
C      REPLACE COMPUTATION OF FFZ1 AND EFZ2 IN ENDFRC 12/7/76 CARDS
C      INTERPOLATE FOR NORMAL AT END WITH QUADRATIC INTERPOLATION
XMX1 = DE - DETA2
EFC021
XMX2 = DE - 2.0*DETA2
EFD021
C      X-X0 = DE. X2-X1 = X1-X0 = DETA2.
FFC021
SNRFND=(XMX2*(XMX1*SNR(N1H)-2.0*DF*SNR(N1H+1))+DE*XMX1*SNR(N1H+2))
EFD021
1 1*RD22M
EFD021
SNKFND=(XMX2*(XMX1*SNK(N1H)-2.0*DF*SNK(N1H+1))+DE*XMX1*SNK(N1H+2))
EFD021
1 1*RD22M
EFD021
AAA = SQRT(SNRFND**2 + SNKFND**2)
EHD021
SNREND = SNREN/AAA
FRD021
SNKFND = SNKEND/AAA
EF0D21
TEFR1 = EFFR1*SNREND - EFZ1*SNKEND
EF0D21
EFZ1 = EFFR1*SNKEND + EFZ1*SNREND
EF0D21
EFFR1 = TEFR1
EH0D21
C      TEFR1 = EFFR1*SNR(N1H) - EFZ1*SNK(N1H)
12/7/76
C      FFZ1 = EFFR1*SNK(N1H) + EFZ1*SNR(N1H)
12/7/76
C      EFFR1 = TEFR1
12/7/76
      PFTURN
10/27/76
100 FORMAT(I10,7A10)
10/27/76
101 FORMAT('ISUBROUTINE ENDFRC 10/27/76. TABULAR DATA FFZ1,EFTH,EFMU',10/27/76
1      /* TABLE 1. 7A10,15. POINTS/4X,1,6X,1EFZT 1,9X,1TIME')10/27/76
102 FORMAT(2E15.7)
10/27/76
103 FORMAT(15.1P2F15.7)
10/27/76
104 FORMAT(/' TABLE 2. 7A10,15. POINTS/4X,1,6X,1EFTHT 1,9X,1TIME')10/27/76
105 FORMAT(/' TABLE 3. 7A10,15. POINTS/4X,1,6X,1EFMUT 1,9X,1TIME')10/27/76
END
10/27/76
*IDFNT EAPFPC
EAP FPC
*I RPSL10.5
EAP FPC
C      EXTERNAL WORK FOR APL FRC OF 4/12/76. (6/3/76)
C MAIN.2A
COMMON DWF1P,LTH1P,DWF2P,DTM2P
EAP FRC
*I WPSL10.43
      DTM1P = 0.0
EAP FPC
      DWF1P = 0.0
EAP FPC
      DWF2P = 0.0
EAP FPC
      DTM2P = 0.0
EAP FPC
*ON MOTION.40

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IF(N .GT. N1R)GOTO 130          EAP FRC
DRII = DRI                      EAP FRC
DZII = DZI                      EAP FRC
130 CONTINUE                     EAP FRC
IF(IRECE1 .NE. 4)GOTO 131       EAP FRC
N=N1R                           EAP FRC
DWF = 0.5*(EFH1*(DRII+DR(N))+EFZ1*(DZII+DZ(N)))   EAP FRC
R2=HTD2*(-3.0*(R(N)+DR(N))+4.0*(R(N+1)+DR(N+1))-(R(N+2)+DR(N+2)))   EAP FRC
Z2=PTU2*(-3.0*(Z(N)+DZ(N))+4.0*(Z(N+1)+DZ(N+1))-(Z(N+2)+DZ(N+2)))   EAP FRC
D = SQRT(Z2**2 + R2**2)         EAP FRC
SNRN = Z2/D                      EAP FRC
SNKN = -R2/D                     EAP FRC
DTHTF = ARCSIN(SNK(N)*SNRN - SNR(N)*SNKN)           EAP FRC
DWM = -0.5*EM1*(DTHIP + DTNET)                   EAP FRC
TNRG = TNRG + 0.5*(DWF + DWM + DWE1P)             EAP FRC
DWE1P = DWF + DWM                  EAP FRC
DTHIP = DTNET                     EAP FRC
131 IF(IRECE2 .NE. 4)GOTO 132       EAP FRC
N=N2R                           EAP FRC
DWF = 0.5*(EFH2*(DRI+DRS) + EFZ2*(DZI+DZS))        EAP FRC
R2=HTD2*(R(N-2)+DR(N-2)-4.0*(R(N-1)+DR(N-1))+3.0*(R(N)+DR(N)))   EAP FRC
Z2=HTD2*(Z(N-2)+DZ(N-2)-4.0*(Z(N-1)+DZ(N-1))+3.0*(Z(N)+DZ(N)))   EAP FRC
D = SQRT(Z2**2 + R2**2)         EAP FRC
SNRN = Z2/D                      EAP FRC
SNKN = -R2/D                     EAP FRC
DTHTF = ARCSIN(SNK(N)*SNRN - SNR(N)*SNKN)           EAP FRC
DWM = -0.5*EM2*(DTHP + DTNET)                   EAP FRC
TNRG = TNRG + 0.5*(DWF + DWM + DWE2P)             EAP FRC
DWE2P = DWF + DWM                  EAP FRC
DTHP = DTNET                     EAP FRC
132 CONTINUE                     EAP FRC
*IDENT 11/2/76
*I RPSL1D.5
C     INSERT TO REMOVE FAILURE HALT AT FREE OR FORCED END.  11/2/76  11/2/76
*I RMSTRS.13
C     INSERT TO REMOVE FAILURE HALT AT FREE OR FORCED END.  11/2/76  11/2/76
    IF(G22 .GT. 0.0)GOTO 2          11/2/76
    IF(IP .EQ. 2)GOTO 2            11/2/76
    IF(N .EQ. N1R .AND. IRECE1.EQ.4)G22=1.0E-10      11/2/76
    IF(N .EQ. N2R .AND. IRECE2.EQ.4)G22=1.0E-10      11/2/76
? CONTINUE                       11/2/76
*IDENT INVELC
*I INVEL.6
C     LATERAL VELOCITY = VH, TOWARD END 1.                1/8/76
C     ENTIRE ROD MOVING WITH CONSTANT VELOCITY.  7/20/76
    WRITE(6,333)                    1/8/76
333 FORMAT(//1 LATERAL VELOCITY TOWARD END 1. 1/8/76//)  1/8/76
    DO 29 N=N1V,N2V                 7/20/76
    DZ(N)=VR                      1/8/76
29    WRITE(6,666)N,DZ(N),DR(N)      1/8/76
    RETURN                         1/8/76
    666 FORMAT(' N='!,I3,' DZ(N)= ',1PF13.6,' DR(N)= ',F13.6)  1/8/76
*IDENT EROD26
*I RPSL1D.5
C     EROSION OF END1.  12/26/76.  PUT AFTER APLFRC AND SHR3/1.  EROD
C     INCLUDE SUBROUTINE ERODE AND CORRECT SUBROUTINE ENDFRC.
*I MAIN.28
    COMMON DE                      FFC021
*I RPSL1D.129
    CALL ERODE                     EF0021
*I RESULT.61
    IF(N .EQ. N1R)VS(N)=(AMS(N+1)-AMS(N))/(SRA*(DETA2-CF)) 1
    IF(N .EQ. N1R)GOTO 200

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*I MOTION.26
  IF(N .NE. N1B .OR. IHC1 .NE. 4) GOTO 36
C   REPLICATES VP,VZ COMPUTATION IN MOTION WHEN N=N1B AND IHC1=4
  VH = (VS(N)*SHRM(N)-QS(N)*SNKV(N)-FFR1)/(0.5*DETA2 -DF)
  VZ = (VS(N)*SNKM(N)+QS(N)*SNRM(N)-FFZ1)/(0.5*DETA2 + DF)
36 CONTINUE
*I STRAIN.81
  IF(J1 .GE. N1R) GOTO 46
  EPSS1(I) = EPSS2(I) = 0.0
  EPSANG(I) = EPSANH(I) = -1.0E20
  GOTO 46
44 CONTINUE
*I STRAIN.97
  IF(NQ1 .GE. N1H) GOTO 49
  D1 = D2 = 0.0
  GOTO(71,50),LINK
44 CONTINUE
*I PDATA.7
  N1H = ?
*D PDATA.61
  100 N1R = N1A
  RETURN
*AF
*DECK ERODE
  SUBROUTINE ERODE
C   INSERT ANY INPUT FOR ERODE AFTER INPUT FOR ENDFRC.
C   THIS USER SUBROUTINE COMPUTES THE EROSION ON FN01 OF A ROD.
C   THIS VERSION ASSUMES TABLE OF FROSTON FROM END OF ROD VS TIME.
C   THE TABLE PRESUMABLY AGREES WITH ENDFRC VS TIME TABLE IN ENDFRC
C   AND STRESS-STRAIN TABLE READ IN INPUT.
*CALL MAIN
  DIMENSION LAPLTE(7),TEROD(100),XEND(100)
  DATA IERODE/0/
  IF(IERODE .EQ. 1) GOTO 10
  DECOM = DETA2/4.0
  IERODE = 1
  FRODP = 0.0
  DF = 0.0
  READ(5,100)NTEPTS,(LARLTC(I),I=1,7)
  WRITE(6,101)NTEPTS,(LARLTC(I),I=1,7)
  READ(5,102)(TEROD(I)+XEND(I),I=1,NTEPTS)
  WRITE(6,103)(I,TEROD(I),XEND(I),I=1,NTEPTS)
100 FORMAT(10,7A10)
101 FORMAT('1SUBROUTINE ERODE 5/26/77, TABULAR DATA TIME VS XEND')
2 15.1 POINTS:5X,7A10,/4X,'I',7X,'TIME',9X,'XEND')
102 FORMAT(2E15.7)
103 FORMAT(15,2E15.7)
10 CALL DIVINT(TIME,EROD,TEROD,XEND,NTEPTS,2)
  DELDF = EROD - ERDP
  DF = DF + DELDF
  ERDP = EROD
  P(54) = EROD
C   CODE CHECKING PLOT 5/27/77
10 IF(DF .LT. DECOM) GOTO 20
  DF = DF -DETA2
  R(N1R) = Z(N1R) = -1.0E20
C   THE FOLLOWING CARD IS NEEDED IF THE DECK INCLUDES MSQSVS.
  AMS(N1H) = QS(N1H) = VS(N1B) = 0.0
  N1R = N1V = N1A = N1R+1
  GOTO 19
20 RETURN
  END
?

```

FRODING ROD (254 RY 8.12H MM) HFPEAT OF 6/P/77 RUN  
 36 6 2.0  
 130 0 9999 .500000E-6  
     4  
     0 9999 68.64757 .001  
 1.985714E11 .27 1.3900000F9 7790. .008128 4 0  
 0.007 1.340 E9  
 0.008 1.51E E9  
 0.009 1.564 E9  
 0.010 1.597 E9  
 0.011 1.624 E9  
 0.012 1.648 E9  
 0.013 1.669 E9  
 0.014 1.688 E9  
 0.015 1.705 E9  
 0.016 1.721 E9  
 0.017 1.736 E9  
 0.018 1.750 E9  
 0.019 1.763 E9  
 0.020 1.776 E9  
 0.022 1.799 E9  
 0.024 1.820 E9  
 0.026 1.840 E9  
 0.028 1.860 E9  
 0.030 1.879 E9  
 0.032 1.897 E9  
 0.034 1.915 E9  
 0.036 1.932 E9  
 0.038 1.949 E9  
 0.040 1.966 E9  
 0.045 2.008 E9  
 0.050 2.048 E9  
 0.055 2.08E E9  
 0.060 2.126 E9  
 0.065 2.164 E9  
 0.070 2.202 E9  
 0.075 2.239 E9  
 0.080 2.276 E9  
 0.085 2.312 E9  
 0.090 2.348 E9  
 0.095 2.384 E9  
 0.100 2.419 E9  
 0.105 2.454 E9  
 0.110 2.489 E9  
 0.115 2.523 E9  
 0.120 2.557 E9  
 0.125 2.590 E9  
 0.130 2.624 E9  
 0.135 2.656 E9  
 0.140 2.689 E9  
 0.145 2.721 E9  
 0.150 2.753 E9  
 0.30 3.713 E9  
 -1.0  
 20 1 1 1  
 2 70 130  
 2 70 130  
 2 70 130  
 2 53 54  
 0.021167 6  
 0.005 45.0 135.0 0  
 0.010 45.0 135.0 0  
 0.015 45.0 135.0 0

.01926	45.0	135.0	0
.03967	45.0	135.0	0
.06	45.0	135.0	0
.254	.004064	1	
.004064			
4.14606E-31	3.35467E-3	3.3543E-3	
6.98023E-31	3.35467E-3	2.0124E-3	
7.97487E-31	3.35467E-3	6.7086E-4	
7.97487E-31	3.35467E-3	6.7086E-4	
6.98023E-31	3.35467E-3	2.0126E-3	
4.14606E-31	3.35467E-3	3.3543E-3	
1000.			
29 FFZ1 AND TIME 6/3/77 CASE B			
0.0	0.0		
72160.00	1.000	E-6	
78750.00	1.590	E-6	
81140.00	1.790	F-6	
82460.00	2.143	E-6	
85550.00	2.715	E-6	
87650.00	2.415	F-6	
89380.00	3.115	F-6	
90490.00	3.315	F-6	
92220.00	3.515	F-6	
95090.00	3.490	F-6	
97600.00	4.372	F-6	
99940.00	4.847	F-6	
102160.00	5.257	E-6	
104310.00	5.579	E-6	
106390.00	5.968	E-6	
110430.00	6.500	E-6	
114350.00	7.123	E-6	
118200.00	7.574	E-6	
121460.00	7.943	E-6	
125650.00	8.240	E-6	
129260.00	8.495	F-6	
132790.00	8.717	E-6	
136250.00	8.917	E-6	
139630.00	9.117	E-6	
142940.00	9.317	F-6	
159310.00	9.817	E-6	
192050.00	19.817	F-6	
192050.00	1.00000		
? THETA = 0.0. 3/8/77			
0.0	0.0		
0.0	1.0		
? MU = 0.0. 3/8/77			
0.0	0.0		
0.0	1.0		
29 TIME AND XEND 6/3/77 CASE B			
0.000	F-6	3.000	E-3
1.040	F-6	4.000	E-3
1.590	F-6	9.000	E-3
1.790	F-6	9.256	E-3
2.143	F-6	9.256	E-3
2.715	F-6	9.256	E-3
2.915	F-6	9.403	E-3
3.115	F-6	9.482	E-3
3.315	F-6	9.644	E-3
3.515	F-6	9.455	E-3
3.490	F-6	9.455	E-3
4.372	F-6	9.455	F-3
4.847	F-6	9.455	F-3
5.257	F-6	9.455	F-3

5.579	F-6	9.855	E-3	
5.968	E-6	9.855	F-3	
6.580	F-6	9.855	F-3	
7.123	F-6	9.855	F-3	
7.574	F-6	9.855	F-3	
7.943	E-6	9.855	F-3	
8.240	F-6	9.855	F-3	
8.495	F-6	9.855	E-3	
8.717	E-6	9.855	F-3	
8.917	F-6	9.864	E-3	
9.117	F-6	9.903	E-3	
9.317	F-6	9.915	F-3	
9.817	E-6	9.915	F-3	
19.817	E-6	9.915	E-3	
1.00000		9.915	F-3	
?				
*IDFNNT PLOTP				PLOTP
*D RP1PLT.18 READ(NPLOT)NNPF,(NPF(I),I=1,NNPF)				PLOTP
*D RP1PLT.40 1,(PDAT(J),J=1,NNPF)				PLOTP
*ID MSQSVS				MSQSVS
*D MAIN.10				MAIN.10
C     PLOTTING MS, QS, AND VS. 3/19/76				MSQSVS
COMMON/MSUSVS/ AMS(103),QS(103),VS(103),XSN(103)				MSQSVS
LFVFL2,TIM,PR,AMS				MSQSVS
*I RP1PLT.61				RP1PLT.61
C     PLOTTING MS, QS, AND VS. 3/19/76				MSQSVS
READ (NPLOT)(AMS(N),QS(N),VS(N),N=1,N)				MSQSVS
IF(EOF(NPLOT) .NE. 0)GOTO 2H				MSQSVS
*D PLOT3D.53 XHAR = 3.0				PLOT3D.53
*I PLOT3D.93				PLOT3D.93
C     PLOTTING MS, QS, AND VS. 3/19/76				MSQSVS
IF(NCYCLE .EQ. 0) PFTL.PN				PFTL.PN
ENCODE(10,711,HOH)NCYCLE				NCYCLE
711 FORMAT(5HCYCLE,I4,1H)				5HCYCLE
HDF1 = 10HMESH PTS.>				10HMESH
CALL PLTPGE				PLTPGE
DO 72 I=1,I2				I=1,I2
XSN(I) = FLOAT(I)+1.0				FLOAT(I)+1.0
72 CONTINUE				CONTINUE
CALL FIXSCA(XSN(1), I2,7.9,0SX5,XMINS,XMAXS,DELDXS)				FIXSCA
CALL FIXSCA(AMS(1), I2,6.4,0SYS,YMINS,YMAXS,DELUYS)				FIXSCA
CALL PLTSCA(3.0, 2.0,0.0,0.0,1.0,1.0)				PLTSCA
HD = 5H MD				5H MD
CALL PLTSYM(.1,MD, 90.0, -1.2,2.5)				PLTSYM
CALL PLTSYM(.1,MDP+0.0, 3.5,-1.0)				PLTSYM
CALL PLTSYM(.1,MDP+0.0, 3.5,-0.6)				PLTSYM
CALL PLTSCA(3.0, 2.0,XMINS,YMINS,0SX5,PSYS)				PLTSCA
CALL PLTDT2(1.0,XSN(1),AMS(1), I2,0)				PLTDT2
CALL PITAXS(DFLUXS,DELDYS,XMINS,XMAXS,YMINS,YMAXS,4)				PITAXS
CALL LAHELA(DFLUXS,DELDYS,XMINS,XMAXS,YMINS,YMAXS,1.0+1.0)				LAHELA
I2M1 = I2-1				I2-1
DO 73 I=1,I2M1				I=1,I2M1
XSN(I) = XSN(I)+0.5				XSN(I)+0.5
73 CONTINUE				CONTINUE
CALL FIXSCA(QS(1),I2M1,6.4,0SYS,YMINS,YMAXS,DELUYS)				FIXSCA
CALL PLTSCA(3.0,12.0,0.0,0.0,1.0,1.0)				PLTSCA
HD = 5H Q>				5H Q>
CALL PLTSYM(.1,MD, 90.0, -1.2,2.5)				PLTSYM
CALL PLTSYM(.1,MDP+0.0, 3.5,-1.0)				PLTSYM
CALL PLTSYM(.1,MDP+0.0, 3.5,-0.6)				PLTSYM

CALL PLTSCA(3.0,12.0,XMINS,YMINS,DXS,DSYS)	MSQSVS
CALL PLTTT2(1.0,XSN(1),VS(1),I2M1,0)	MSQSVS
CALL PLTAXS(DFLUXS,DELRYS,XMINS,XMAXS,YMINS,YMAXS,4)	MSQSVS
CALL LAHFLA(DFLUXS,DELRYS,XMINS,XMAXS,YMINS,YMAXS,1.0,1.0)	MSQSVS
CALL FIXSCA(VS(1),I2M1,6,4,DSYS,YMINS,YMAXS,DFLORYS)	MSQSVS
CALL PLTSCA(3.0,22.0,0.0,0.0,1.0,1.0)	MSQSVS
HD = SH VS	ANYUNITS
CALL PLTSYM(1,HD,90,0,-1.2,2.5)	MSQSVS
CALL PLTSYM(1,HD,0,0,3.5,-1.0)	MSQSVS
CALL PLTSYM(1,HD,1,0,0,3.5,-0.6)	MSQSVS
CALL PLTSCA(3.0,22.0,XMINS,YMINS,DXS,DSYS)	MSQSVS
CALL PLTDOT2(1.0,XSN(1),VS(1),I2M1,0)	MSQSVS
CALL PLTAXS(DFLUXS,DELRYS,XMINS,XMAXS,YMINS,YMAXS,4)	MSQSVS
CALL LAHFLA(DFLUXS,DELRYS,XMINS,XMAXS,YMINS,YMAXS,1.0,1.0)	MSQSVS

I = 4

?  
1.0      3.0  
?

Selected output from Example 2

```

HAL PENNSIL CRPF

FRONTING PPN 1254 BY 0.170 MM) PEPAT OF 4/8/77 RUN
COMPUTATIONS FOR ENERGY USE IN TAIR -10000E+01
36 MESHES IN FLAP DIRECTION (14142) .70455E+02

PENNING TIME INCREMENT -0.00000E+00
IN PENNING TIME INCREMENT -0.130747E+00
INPUT TIME INCREMENT -0.500000E-06

TIP. INCREMENT USED BY STPSL -50000E-06

YOUNG'S MODULUS = 194571E+12 YIELD STRESS = 13990E+10
POISSON'S RATIO = .27000E+00 THICKNESS = .01125E+02
MASS DENSITY = .77000E+00

START AT TIME STEP 0
FINAL TIME STEP 130
SURFACE STRAINS EVERY 20 TIME STEP
RESTART WRITE EVERY 9999 TIME STEP

LAYER = 6 NODIN = 0
LAYER = 0 LPRESS = 0

1/2/77/0/ = CLAMPED/SYMMETRICAL/FREE/
BOUNDARY CONDITIONS
END1 (LINEAL) = 4
END2 (PIPE?) = 4

PRINT OPTION CONTROL CARD
0/1 = NO PRINT/PPT/1
1 DISPLACEMENT INTEGRALS
1 CARTESIAN COORDINATES, PRESSURE
1 SURFACE NORMAL VECTOR COMPONENTS

POINT INFORMATION AT THE FOLLOWING TIME STEPS
70 130
POINT L MATHIS (LPTA) AT THE FOLLOWING TIME STEPS
70 130
3-IN POINTS FOR THE FOLLOWING TIME STEPS
70 130

CONSTITUTIVE RELATION ELASTOPLASTIC-NO WORK HARDENING-STRAIN RATE INDEPENDENT
STRESS-STRAIN AND STRAIN RATE PARAMETERS
1 1.39900E+00 7.00000E-03 0. 0. 1/PEN(J)
1 0. 0. 0. 0. 0. 0.

START DAMPING AFTER TIME STEP 9999 TIP = .50000E-02
DAMP = .00000E+00

LATENT VISCOSITY IN STEP PPN 1. 1/4/76

N# 2 07(0) = -1.00000E+03 0.0(0) = 0.
N# 3 07(1) = -1.00000E+03 0.0(1) = 0.
N# 4 07(2) = -1.00000E+03 0.0(2) = 0.
N# 5 07(3) = -1.00000E+03 0.0(3) = 0.
N# 6 07(4) = -1.00000E+03 0.0(4) = 0.
N# 7 07(5) = -1.00000E+03 0.0(5) = 0.
N# 8 07(6) = -1.00000E+03 0.0(6) = 0.

```



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PHYSICAL ATTRIBUTES AND CULTURE IN 1920 AND 1930

PRESSURE  
COORDINATES  
7 IN.)

*Signs and symptoms of type incipient in sinus*

MANTON

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SNR (m)	SNR (m)	SNR (m)
-6.36672454694745E-15	-6.36672454694745E-15	-6.36672454694745E-15
-4.655235661102768E-15	-4.655235661102768E-15	-4.655235661102768E-15
-3.10272510224095E-15	-3.10272510224095E-15	-3.10272510224095E-15
-1.20272510224095E-15	-1.20272510224095E-15	-1.20272510224095E-15
-2.4446323604635E-16	-2.4446323604635E-16	-2.4446323604635E-16
-8.16622351631269E-16	-8.16622351631269E-16	-8.16622351631269E-16
-2.04016222925604E-16	-2.04016222925604E-16	-2.04016222925604E-16

11	-1.0461100762460614E-17	.1100000000000000E+00	.2065111934430H3H3H-15
12	.14344177548P1701E-15	.1000000000000000E+00	.30478421742421E-16
13	.1000000000000000E+00	.1000000000000000E+00	.143794684684684623E-17
14	.71445453P13095A9H029316H-16	.1000000000000000E+00	.-1.04H4653H2P039E-15
15	.-207440329357155E-16	.1000000000000000E+00	.4755922010159E-16
16	.-2764412P463430627-16	.1000000000000000E+00	.-1.04067012465101E-16
17	.-476317225238612549-16	.1000000000000000E+00	.-30442P709P606177E-16
18	.-354541877957241644E-17	.1000000000000000E+00	.-61094683363807950E-16
19	.-900464676119041F-16	.1000000000000000E+00	.-5365449544654E-16
20	.-A7067466666661311E-16	.1000000000000000E+00	.-1244444925513919E-15
21	.-280336366210920001-16	.1000000000000000E+00	.-206161865931590E-15
22	.-1046373573011300E-15	.1000000000000000E+00	.-1.042P00404228E1E-15
23	.-547627106732R72u-16	.1000000000000000E+00	.-30442P709P606177E-15
24	.-124440656449110P6E-15	.1000000000000000E+00	.-227244221114370E-15
25	.-1000000000000000E+00	.1000000000000000E+00	.-47204523816P2021E-16
26	.-1073488467312356u-15	.1000000000000000E+00	.-12P252905553396E-15
27	.-3725541691105174E-17	.1000000000000000E+00	.-1.045396594347H67E-15
28	.-1712453067607054E-15	.1000000000000000E+00	.-116954147778692E-15
29	.-1224130647914917E-16	.1000000000000000E+00	.-1277346466467463E-15
30	.-1000027694611027E-15	.1000000000000000E+00	.-4990046847663631E-15
31	.-4117063451293377F-16	.1000000000000000E+00	.-333303R611133514E-14
32	.-16R363169316931693169-01	.1000000000000000E+00	.-3R24349981P2324E-14
33	.-16064252530803247E-16	.1000000000000000E+00	.-6990566490611323E-16
34	.-21P12942346239H-14	.1000000000000000E+00	.-25280286446772E-16
35	.-3942927937163H-16	.1000000000000000E+00	.-5376939730n391E-16
36	.-10476647747004780E-16	.1000000000000000E+00	.-2317280938093419E-16
37	.-8R774998934659P0P5-17	.1000000000000000E+00	.-8012458643123491E-17
38	.-80274300367160431E-17	.1000000000000000E+00	.-206693371Raa2n2n7E-16
	.-401132442657P624P-16	.1000000000000000E+00	

### SUM OF SINES = 0.0000000000000000E+00

4	.4.37H7012725P7R21F-01	
5	.4.37H7012725P7R21F-01	
6	.4.34272264936775E-01	
7	.4.36727264936775F-01	
8	.4.30519574R097F-01	
9	.4.24450552P4556F-01	
10	.4.26460527454KAF-01	
11	.3.070565819P9236E-01	
12	.1.76556519P9236E-01	
13	.2.0522366026137F-01	
14	.2.0522366026137F-01	
15	.1.72R64616P34e3F-01	
16	.1.72R64616P34e3F-01	
17	.3.054767616P9236E-01	
18	.3.054767616P9236E-01	
19	.3.054767616P9236E-01	
20	.3.054767616P9236E-01	
21	.3.054767616P9236E-01	
22	.3.054767616P9236E-01	
23	.3.054767616P9236E-01	
24	.3.054767616P9236E-01	
25	.3.054767616P9236E-01	
26	.3.054767616P9236E-01	
27	.3.054767616P9236E-01	
28	.3.054767616P9236E-01	
29	.3.054767616P9236E-01	
30	.3.054767616P9236E-01	

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PLASTIC - 1368627487-03 PLASTIC - 137161363-00  
PLASTIC - 137161363-00 PLASTIC - 137161363-00

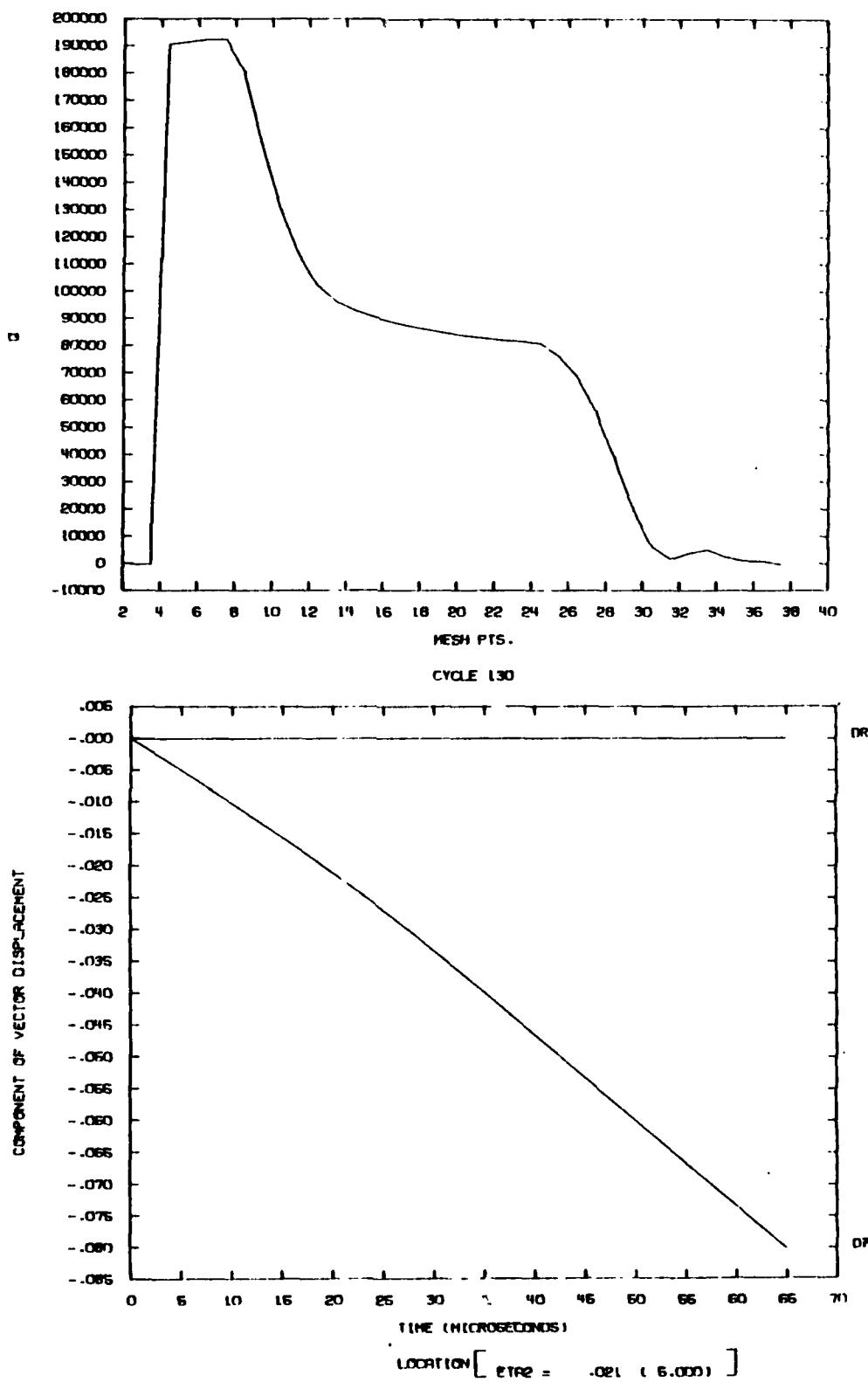


Figure E.2.1. Plot of axial force, Q, at cycle 130 and the displacement history at mesh point 5.

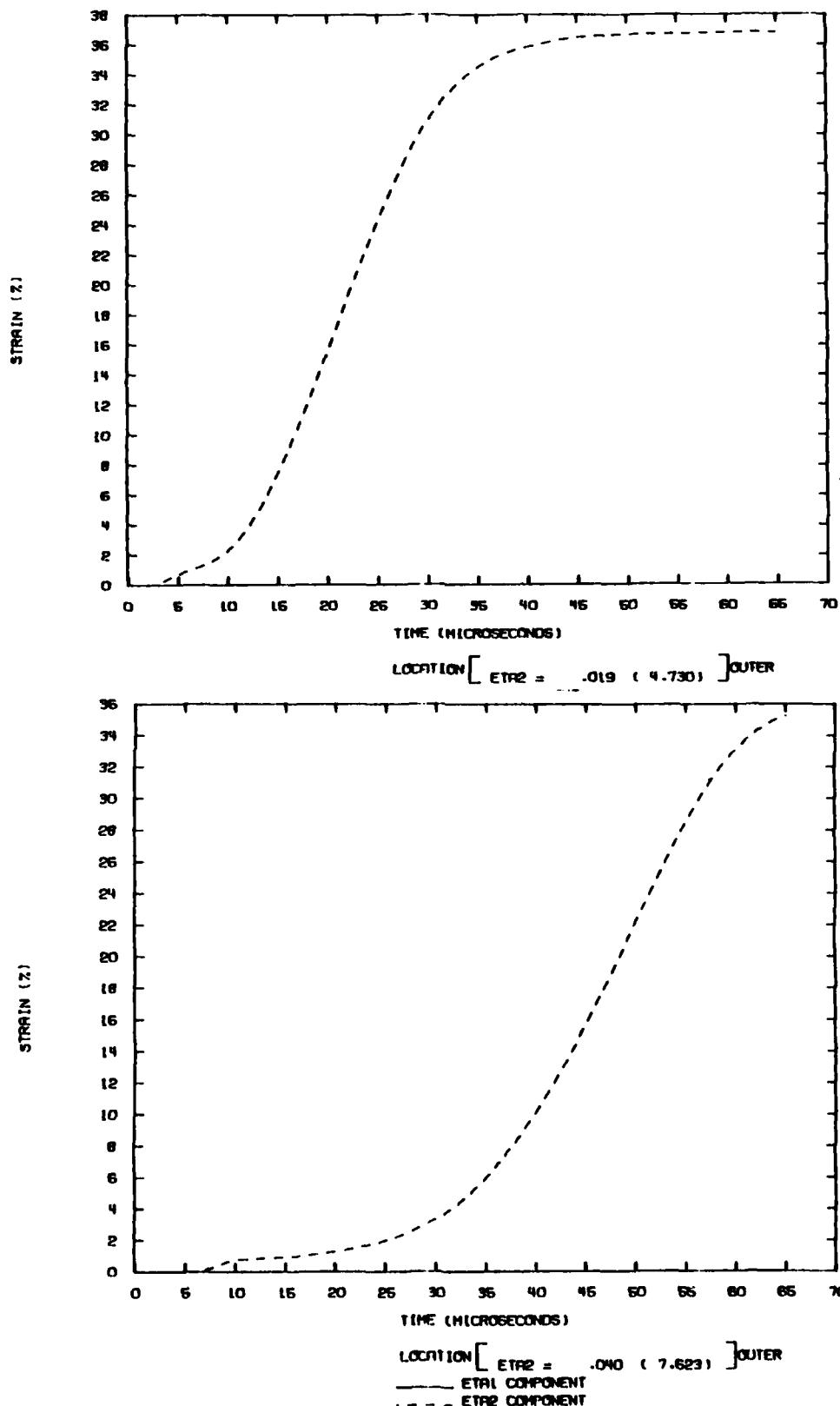


Figure E.2.2. Strain plots at 19.26 mm and 39.67 mm.

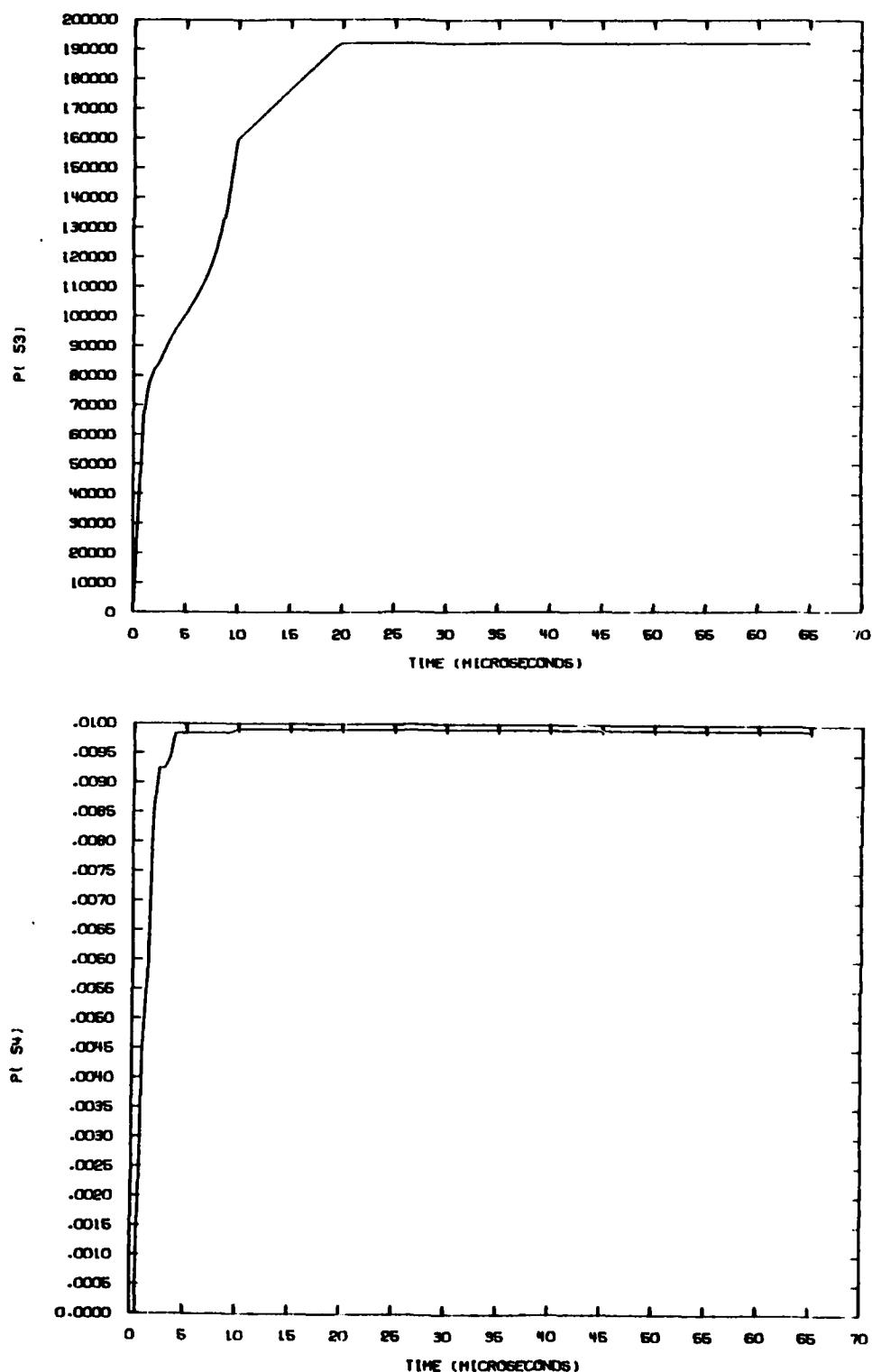
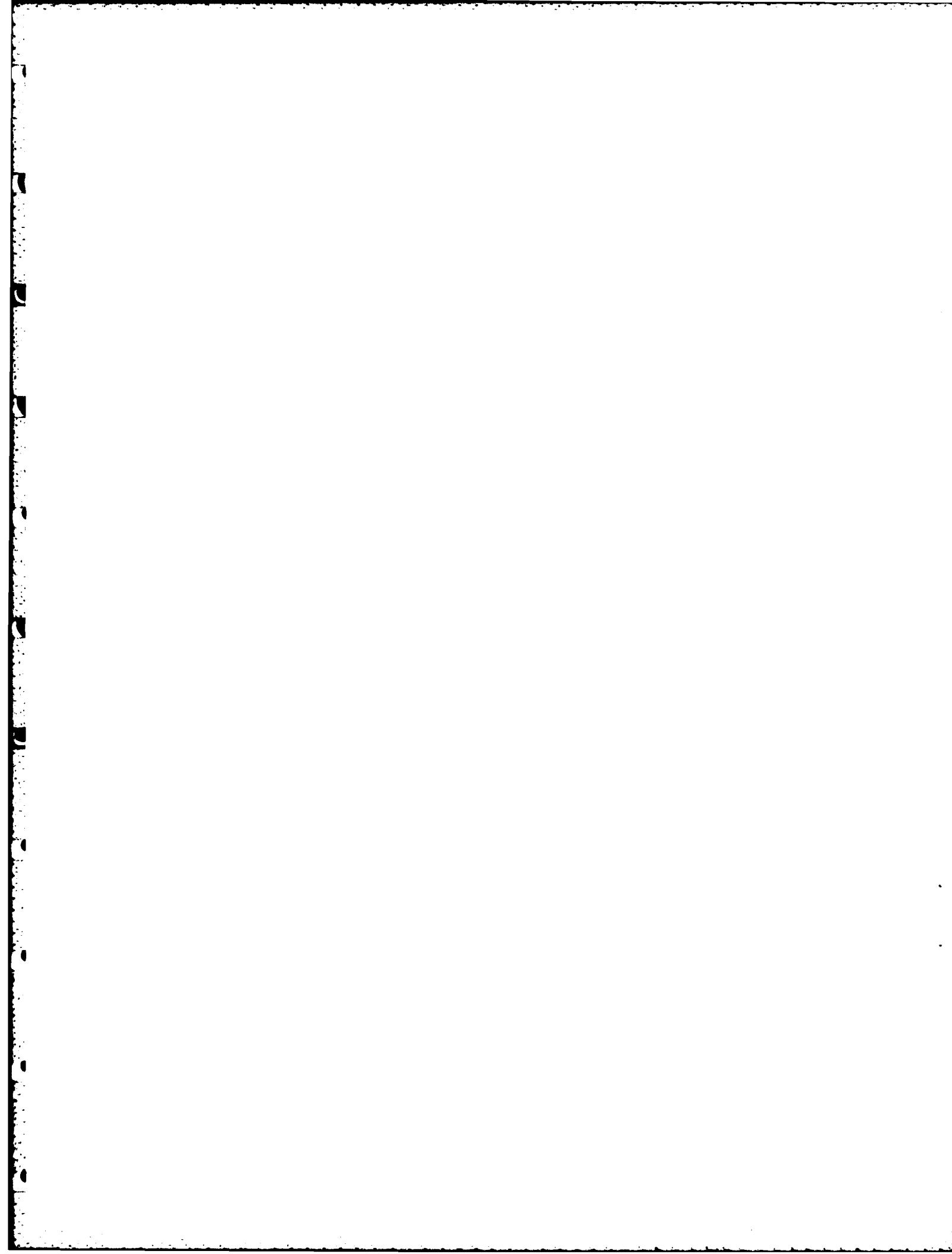


Figure E.2.3. Plots of force on end1,  $P(53)$ , and erosion at end1,  $P(54)$ , as functions of time.



## APPENDIX F

### SOME ALTERNATE USER SUBROUTINES

UPDATE input cards for alternate versions of the user subroutines INGEOM, PRESS, and ENDFRC are listed in this appendix. Other versions of the user subroutines are listed in Appendices C and E. Descriptions for user subroutines are given in Section 4.

The following are listed in the order given:

PRESS for Two Phase Pressure Decay in a Cylinder. This is described in Section 4.2.3.

INGEOM for an axisymmetric cylinder. This is described in Section 4.1.2.

ENDFRC for one or both ends free. This is described in Section 4.5.1.

```

*IN  PPRESS.4,PRESS.14
C      PRESS SUBROUTINE, 3/10/76. TABULAR DATA.
C      POSITION ZSI = 0 CORRESPONDS TO THE I=0 REPSIL POINT N=2.
C      THIS PRESS SET IN FOR A SYMMETRIC AXIYMMETRIC CYLINDER
C      DIMENSION ZSI(50),TSI(50),PSI(50),TS(103),PS(103),TSH(103).
1  PSR(103),PIU(6)
DATA IPRESS/0/
IF(IPRESS .GT. 0)GOTO 30
IPRESS =1
READ(5,100) PRAR,BETA,(PID(I),I=1,e)
READ(5,105) CONCZ,CONCT,CONCP,TDIF
WRITE(6,102) PRAR,BETA,(PID(I),I=1,6),TDIF,CONCZ,CONCT,CONCP
READ(5,103) ALPHAS,ALUSF,ISM
READ(5,101) (ZSI(I),TSI(I),PSI(I),I=1,ISM)          12/2/75
READ(5,104) ALPHAS,(ZSI(I),TSI(I),PSI(I),I=1,ISM)    12/2/75
C      CONVERT DATA TO Z=IN. T=SEC. P=LBS/SGIN.          12/2/75
      DO 2 I=1,ISM
ZSI(I) = CONCZ*ZSI(I)
TSI(I) = CONCT*(TSI(I)-TDIF)
PSI(I)=CONCP*PSI(I)                                     12/2/75
? CONTINUE
      WRITE(6,106)(ZSI(I),TSI(I),PSI(I),I=1,ISM)
C      INITIATE DATA FOR COMPUTING THE PRESSURE
DO 2H N=NIV,N2V
DEN = FLOAT(N-2)*BETA2
CALL PVPOINT(CFN,TS(N),ZSI,TSI,ISM+2)
CALL IVPOINT(DEN,PS(N),ZSI,PSI,ISM+2)
TSH(N) = TS(N) - ALOG(PBAR/PS(N))/ALPHAS
PS(N) = PS(N)*EXP(ALPHAS*TS(N))
PSH(N) = PRAR*EXP(BETA*TS(N))
2H CONTINUE
      WRITE(6,111)(Z(N),TS(N),PS(N),TSH(N),PSB(N),N=NIV,N2V)
C      COMPUTE THE PRESSURE
30 EALPHT = EXP(-ALPHAS*TIME)
BHEAT = EXP(-BETA*TIME)
DO 30 N=NIV,N2V
IF(TIME .LT. TS(N))GOTO 40
IF(TIME .GT. TSH(N))GOTO 31
P(N) = -EALPHT*PS(N)
GOTO 30
31 P(N) = -BHEAT*PS(N)
39 CONTINUE
40 RETURN
100 FORMAT(2E10.3,6A10)
101 FORMAT(3E10.3)
105 FORMAT(4E10.3)
102 FORMAT(//,3X,'PRESS SUBROUTINE FOR 1-D STRUCTURE,3/10/76.')
1 /* FROM TABLES OF ARRIVAL TIME AND PEAK PRESSURE VS POSITION.*/
2 /* POSITION = 0 CORRESPONDS TO N=2 OR 1-D REPSIL.*/
3 /* THE QUASI-STATIC PRESSURE IS !PF!5.7,5X,1WITH EXPONENTIAL */
4 /* DAMPING COEFFICIENT = !F15.7/10X,6A10/
* /* INPUT TIME, TSI, REDUCED BY TDIF = !F15.7/
5 /* CONVERSION COEFFICIENTS FOR Z, T, AND P ARE !3E15.7)
103 FORMAT(2E10.3,I10)
104 FORMAT(//20X,'INPUT DATA           /* ALPHAS=!,1E15.7,
1                                //7X,!ZSI!,12X,!TSI!,12X,!PSI!/(!3E15.7))
106 FORMAT(//5X,'TABULAR DATA AFTER CONVERSION'
1 7X,!ZSI!,12X,!TSI!,12X,!PSI!/(!PF!3E15.7))
111 FORMAT(//5X,'FINAL DATA           /*HX!Z!14X!TS!13X!PS!13X
1 !TS!12X!PS!//(!PF!3E15.7))
END

```

```

*P INGEOM.7.INGEOM.47
*CALL MAIN
C
C TAGHON FOR ASYMMETRIC CYLINDER. 12/2/75.
C
C      SET PROGRAM TO USE GAUSSIAN INTEGRATION
IGAUSS = 1
MFAD(5,100)CYLL,RADIUS
WRITE(6,101)CYLL,RADIUS
DETA1 = 2.0*3.141592653589793*RADIUS
C      DFTA1 = 2*PI*RADIUS GIVES ENFPGY FOR ENTIRE CYLINDER
DFTA2 = CYLL/FLUAT(NMFSH)
IB = 0
***** EVALUATE R(N) AND Z(N), N=N1P,N2B *****

DO 10 N=N2B
R(N)=RADIUS
Z(N)=FLOAT(N-2)*DFTA2
10 CONTINUE
RETURN
100 FORMAT(2E12.6)
101 FORMAT(//' INGEOM FOR CYLINDER 12/3/75. LENGTH =1PF15.6,5X,
     1   RADIUS =1,F15.6/')
END

```

```

*AF
*DECK  ENDFRC
      SUBROUTINE ENDFRC
*CALL  MAIN
C
C      THE APL FRC CHANGE TO (1-0) REPSIL FOR HEAMS REQUIRES A USER
C      SUBROUTINE ENDFRC TO SUPPLY FORCES AND MOMENTS AT BOTH ENDS.
C      FOR AN INITIALLY STRAIGHT BEAM, R(N)=0.0, Z(N)=(N-2)DETA2,
C      FFZ1>0 DECREASES DZ(2), EFR1>0 DECREASES DR(2)
C      FFZ2>0 INCREASES DZ(N2H), EFP2>0 INCREASES DR(N2H)
C      FM1>0 DECREASES DR(2)      AND INCREASES DH(3)
C      FM2>0 INCREASES DR(N2H-1) AND DECREASES DR(N2H)
C
C      THIS VERSION IS FOR A RFAM WITH ONE, TWO, OR NO FREE ENDS.
C      DATA TFMFR/0/
C      IF(TFMFR .GT. 0)RETURN
C      TFMFR = 1
C      FFR1 = 0.0
C      FFZ1 = 0.0
C      FM1 = 0.0
C      FFP2 = 0.0
C      FFZP = 0.0
C      FM2 = 0.0
C      RETURN
C      END

```

```

*ID  STREN
*T  RPSLID.5
C      NEW STRAIN ENERGY COMPUTATION FOR A BEAM.  1/20/76
*T  ANSTS.7
C      NEW STRAIN ENERGY COMPUTATION 1/20/76
DIMENSION SST(5),DEST(5),LST(5)
*T  HMSTS.33
C      NEW STRAIN ENERGY COMPUTATION 1/20/76
IF(IP .EQ. 2)GOTO 403
SST(J) = SIG22
LST(J) = J
*D  RMSTS.36
C      NEW STRAIN ENERGY COMPUTATION 1/20/76
*T  HMSTS.34
C      NEW STRAIN ENERGY COMPUTATION 1/20/76
TF(IP .EQ. 2)GOTO 35
TF(NSFL .GT. 1)GOTO 10
NSTREN = (SS22)**2
GOTO 30
10 IF(SS22 .LE. 0.0)GOTO 12
SS22 = -SS22
DO 11 L=1,NSFL
11 SST(L) = -SST(L)
12 DO 13 L=1,NSFL
13 DEST(L) = (SIGZZ(L) - SST(L))/F
LLAST=NSFL-1
DO 15 L=1,LLAST
JSTART = L+1
DO 14 J=JSTART,NSFL
IF(DEST(L) .LT. DEST(J))GOTO 14
NSTEMP = DEST(L)
DEST(L) = DEST(J)
DEST(J) = NSTEMP
LSTEMP = LST(L)
LST(L) = LST(J)
LST(J) = LSTEMP
14 CONTINUE
15 CONTINUE
FM = 1.0
ESTHEN = 0.0
DO 25 L=1,NSFL
ESTAR = FM*E
DEPSTR = -SS22/ESTAR
DSTREN = DSTREN + (SS22)**2/FM
IF(DEPSTR .LE. DEST(L)) GOTO 30
      CORNER AT DEST(L) IN UNLOADING CURVE
SS22 = -ESTAR*(DEPSTR-DEST(L))
DSTREN = DSTREN - (SS22)**2/FM
L1 = LST(L)
EM = FM - WT(L1)
DESTL = DEST(L)
DO 20 J=L,NSFL
DEST(J) = DEST(J) - DESTL
20 CONTINUE
25 CONTINUE
30 STREN = STREN + SPG*N(F)*DSTREN
35 CONTINUE

```

## LIST OF SYMBOLS

[FORTRAN name in brackets]

a	determinant of surface metric [DA]
E	Young's modulus of elasticity [E]
h	shell thickness [THICKN]
$\hat{i}_1, \hat{i}_2, \hat{i}_3$	orthonormal vector basis
$\hat{M}^{*\alpha\beta}$	Normal vectors of stress moment resultant (See Reference 1). $\hat{M}^{*\alpha\beta} = \hat{M}^{*\alpha\beta} \underline{n} = a^{\frac{1}{2}} \hat{M}^{\alpha\beta} \underline{n} = a^{\frac{1}{2}} (M^{\alpha\beta} + M^{\beta\alpha}) \underline{n}$ .
$\hat{N}^{*\alpha\beta}$	Modified stress resultant tensor (See Reference 1).
n or N	index associating variables with the N'th mesh point (e.g. R(N)) [N]
$\underline{n}$	unit normal vector to reference surface, $\underline{n} = n_k \hat{i}_2 + n_r \hat{i}_3$
$\underline{r}$	position vector to a point on the reference surface
R, Z	position components. $\underline{r} = R \hat{i}_3 + Z \hat{i}_2$ [R(N),Z(N)]
t	time [TIME]
$w_k$	area or weight associated with $\zeta_k$ [W(K)]
$\Delta t$	time increment [DELTAT]
$\Delta\xi, \Delta\xi^2$	spacing of $\xi^2$ between mesh points [DETA2]
$\xi$	$\xi$ is $\xi^2$
$\xi^1, \xi^2$	material (Lagrangian) coordinates (called $\eta^1, \eta^2$ in Reference 2)
$\zeta$	normal distance from reference surface
$\zeta_k$	$\zeta$ at k'th integration station [ZETA(K)]
$\zeta_l, (\zeta_u)$	$\zeta$ on lower (upper) surface of shell [ZL,(ZU)]
v	Poisson's ratio [FNU]
$\rho$	mass density [RHO]

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